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**FIRES - T3 A COMPUTER PROGRAM FOR THE  
FIRE RESPONSE OF STRUCTURES --  
THERMAL (THREE DIMENSIONAL VERSION)**

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United States Department of Commerce  
Technology Administration  
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October 1977  
Issued June 1996



**U.S. Department of Commerce**  
Michael Kantor, *Secretary*  
**Technology Administration**  
Mary L. Good, *Under Secretary for Technology*  
National Institute of Standards and Technology  
Arati Prabhakar, *Director*



Notice

This report was prepared for the Building and Fire Research Laboratory of the National Institute of Standards and Technology under grant number NBS-G7-9006-10/77. The statement and conclusions contained in this report are those of the authors and do not necessarily reflect the views of the National Institute of Standards and Technology or the Building and Fire Research Laboratory.





# **Fires - T3**

A COMPUTER PROGRAM FOR THE  
FIRE RESPONSE OF STRUCTURES –  
THERMAL

(THREE-DIMENSIONAL VERSION)

by

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RESEARCH SPONSORED BY:  
NATIONAL SCIENCE FOUNDATION  
and  
NATIONAL BUREAU OF STANDARDS

**REPORT NO. UCB FRG 77 - 15 • OCTOBER 1977**



FIRE RESEARCH GROUP REPORT UCB FRG 77-15

F I R E S - T 3

A Computer Program for the  
FIre REsponse of Structures - Thermal  
Three - Dimensional Version

by

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This research was supported by

National Science Foundation Grant No. ERT70-01080 A05  
(Formerly Grant No. GI-43)

and

National Bureau of Standards Grant No. NBS-G7-9006-10/77

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October 1977



## ABSTRACT

This report documents the computer program FIRES-T3 (FIre REsponse of Structures - Thermal - Three-Dimensional Version). The program evaluates the temperature distribution history of general three-dimensional solids or composites such as reinforced concrete subjected to fire environments. There are also options for two-dimensional and one-dimensional heat flow analyses. FIRES-T3 is based on a finite element formulation which considers the temperature dependence of thermal properties and the nonlinearities inherent in modeling the fire boundary condition. The finite element formulation, the idealization of the fire boundary condition, and the general numerical approach used in the program are discussed. Included in the report are appendices that provide input instructions for FIRES-T3, sample problems with listings of their input and output, and a complete Fortran listing of the computer program.



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## 1. INTRODUCTION

This report presents FIRES-T3, a computer program for the FIre REsponse of Structures - Thermal - Three-Dimensional Version, which evaluates the temperature distribution history of structures in fire environments. There are options for fully three-dimensional solids, two-dimensional cross-sections, and structures in which heat flow is one-dimensional. The program also permits the use of one-, two-, and three-dimensional elements together in the same structure. Structures may consist of one material or may be composites such as reinforced concrete. The temperature distributions generated by FIRES-T3 can be used in conjunction with stress analysis programs such as FIRES-RC II (a computer program for the response of Reinforced Concrete Frames [ 6 ]) or FIRES-SL (a computer program for the response of SLabs [ 7 ]), together providing an overall capability of predicting the response of structures subjected to fires.

The heat flow problem is modeled in FIRES-T3 by the heat conduction boundary value problem. These equations are nonlinear because of the temperature dependence of the thermal properties of structural materials and the heat transfer mechanisms associated with fire environments. The solution technique used in FIRE-T3 is a finite element method coupled with time step integration. This general approach has been presented in the work of Wilson [10,11] and Zienkiewicz [12,13] and extended to the fire situation by Bizri [ 3 ]. The nonlinearity of the problem requires an iterative solution process within each step. The element library includes isoparametric 8-node hexahedra and 4-node tetrahedra for three-dimensional solids, 4-node isoparametric quadrilaterals and triangles for two-dimensional modeling, and 2-node isoparametric bar elements for one-dimensional problems. Fire environments are represented by a linear model or a nonlinear model that includes both convective and radiative mechanisms.

In the report, the theoretical model and solution techniques are derived, the organization of the computer program is explained, and a commentary on practical aspects of using the program is made. The appendices contain fully annotated input instructions, sample problems with input and output, and a Fortran listing of the program.

## 2. THERMAL MODEL AND SOLUTION PROCEDURE

### 2.1 Heat Flow Equations

Three dimensional heat flow is governed by the following partial differential equation:

$$\rho C_p \frac{\partial T(x, y, z, t)}{\partial t} = \frac{\partial}{\partial x} (K \frac{\partial T(x, y, z, t)}{\partial x}) + \frac{\partial}{\partial y} (K \frac{\partial T(x, y, z, t)}{\partial y}) + \frac{\partial}{\partial z} (K \frac{\partial T(x, y, z, t)}{\partial z}) + Q_{exo}(x, y, z, t) \quad (2.1)$$

where

$x, y, z$	- spatial coordinates
$t$	- time
$T(x, y, z, t)$	- temperature distribution history
$\rho$	- density (temperature- and space-dependent)
$C_p$	- specific heat (temperature- and space-dependent)
$K$	- isotropic conductivity (temperature- and space-dependent)
$Q_{exo}$	- internal heat generation rate (time- and space-dependent)

An integral part of the above equation is its boundary conditions and initial conditions. The initial conditions consist of the temperature of every point in the structure when the analysis begins, i.e.,

$$T(x, y, z, t_0) = T_0(x, y, z) \quad (2.2)$$

where the temperature distribution  $T_0$  is specified. Boundary conditions must be defined at every point on the structure's surface and can be a specified temperature history or a specified heat flow history. The solution to the heat flow problem is the internal temperature history

that satisfies the field equations (2.1), the initial conditions (2.2), and the prescribed boundary conditions.

For the special case of two-dimensional heat flow, the governing equations are

$$\rho C_p \frac{\partial T(x, y, t)}{\partial t} = \frac{\partial}{\partial x} (K \frac{\partial T(x, y, t)}{\partial x}) + \frac{\partial}{\partial y} (K \frac{\partial T(x, y, t)}{\partial y}) + q_{exo}(x, y, t) \quad (2.3)$$

with appropriate boundary conditions and initial conditions defined over a two-dimensional area.

One-dimensional heat flow is governed by the equation

$$\rho C_p \frac{\partial T(x, t)}{\partial t} = \frac{\partial}{\partial x} (K \frac{\partial T(x, t)}{\partial x}) + q_{exo}(x, t) \quad (2.4)$$

Boundary conditions are defined by specifying a flow rate or temperature at each end of the one-dimensional domain.

## 2.2 Finite Element Matrix Equations

The heat flow equations for two- and three-dimensional bodies described above are very complex and in almost all cases have no closed-form solution; approximate numerical methods must be used in order to obtain a solution. In program FIRES-T3, a finite element method discretizes spacial variables and a step-by-step integration technique discretizes the time variable. The advantages of the finite element method are numerous. The method is completely general with respect to geometry and material properties; complex shapes composed of different materials (e.g. a reinforced concrete structure) are easily represented. Nonlinearity of material and boundary conditions can be treated by the finite element method quite economically. Most numerical methods convert the continuous governing differential or integral equations of heat transfer to a set of linear algebraic equations. However, in the finite element method, the linear equations produce a symmetric positive-definite matrix in banded form which is readily solved with a minimum of computer storage and time.

In the finite element method, the continuum over which Eq. (2.1) or Eq. (2.3) is defined is discretized into a finite number of elements connected at nodal points. The complex partial differential equation is transformed into a system of simultaneous first order differential equations, one at each node. This transformation is effected element by element as explained in the next three sections. The system of nodal equations is then solved by step-by-step integration over the time domain, as explained in Section 2.6.

The finite element equations can be visualized physically as nodal heat balance equations. That is, at each node in the discretization,

$$Q^S + Q^I = Q^E \quad (2.5)$$

where

$Q^S$  = rate at which heat is stored within the elements adjacent to node; in steady state conditions, this term is zero at all nodes

$Q^I$  = rate of internal heat transfer by conduction in the elements adjacent to the node

$Q^E$  = rate at which heat enters the node from an external source

or, in matrix form,

$$[C] \{T\} + [K] \{T\} = \{Q\} \quad (2.6)$$

where

$[C]$  = Capacity matrix  
(temperature-dependent)

$[K]$  = Conductivity matrix  
(temperature-dependent)

$\{Q\}$  = External heat flow vector  
(depends on exothermic reactions and fire boundary conditions)

$\{T\}$  = Temperature vector  
(time-dependent)

$\{T\}$  = Temperature time rate of change vector  
(time-dependent)

The derivation of the three terms of this equation (conductivity matrix, capacity matrix, and heat flow vector) are discussed in the next three sections.

### 2.3 Conductivity Matrix [K]

The terms of the conductivity matrix are associated with the rate of heat flow from the elements adjacent to each node. The conductivity matrix for the system being analyzed,  $\underline{K}$ , is assembled from element conductivity matrices,  $\underline{K}_m$ . That is:

$$\underline{K} = \sum_{m=1}^M \underline{K}_m \quad (2.7)$$

where  $m$  is the element number and  $M$  is the total number of elements. The development of the conductivity matrices for three-dimensional, two-dimensional, and one-dimensional isoparametric finite elements is presented below. Three-dimensional elements can be used only for a three-dimensional analysis. Two-dimensional elements can be used in a two-dimensional analysis or as a component in a three-dimensional analysis. The one-dimensional elements can be used in conjunction with two- and three-dimensional elements in a multidimensional analysis or by themselves to model one-dimensional heat flow.

#### 2.3.1 Three-Dimensional Isoparametric Element

This element is called isoparametric because the geometry of the element and the assumed temperature distribution within the element are described in terms of the same ("iso") parameters (or shape functions) and are of the same order (linear in this case).

An eight node three-dimensional isoparametric element is shown in Fig. 2.1. The natural coordinates  $(\bar{x}, \bar{y}, \bar{z})$  of the eight corner nodes are  $(\pm 1, \pm 1, \pm 1)$ . A natural coordinate system is a local system which permits the specification of a point within the element by a set of

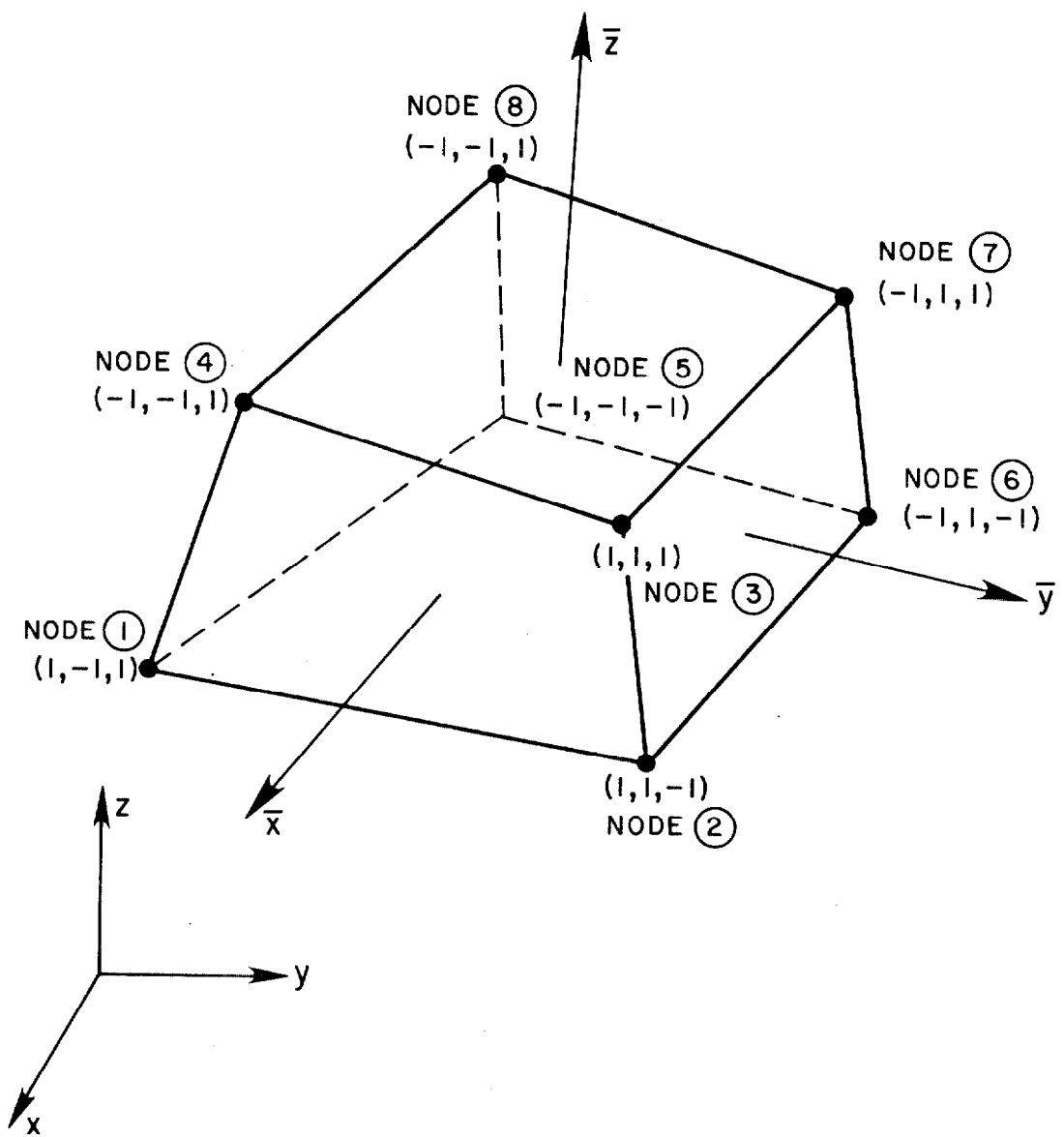


FIGURE 2.1 THREE-DIMENSIONAL ISOPARAMETRIC ELEMENT

dimensionless numbers whose magnitudes never exceed unity. This system is usually arranged so that the natural coordinates have unit magnitude at the nodal points of the element. Such a system simplifies the formulation and facilitates the numerical integration which is required to obtain element matrices. The symbols  $x$ ,  $y$ , and  $z$  denote the global coordinate axes.

The temperature at any point within the element is expressed in the natural coordinate system  $(\bar{x}, \bar{y}, \bar{z})$  in terms of the temperatures at nodes 1 to 8 by the following equation:

$$T(\bar{x}, \bar{y}, \bar{z}, t) = \sum_{i=1}^8 H_i(\bar{x}, \bar{y}, \bar{z}) \cdot T_i(t) \quad (2.8a)$$

or, in matrix form:

$$\begin{matrix} T(\bar{x}, \bar{y}, \bar{z}, t) & = & \langle H_i \rangle & \{T_i\} \\ 1 \times 1 & & 1 \times 8 & 8 \times 1 \end{matrix} \quad (2.8b)$$

where

$$H_i(\bar{x}, \bar{y}, \bar{z}) = \frac{1}{8} (1 + \bar{x} \bar{x}_i)(1 + \bar{y} \bar{y}_i)(1 + \bar{z} \bar{z}_i) \quad (2.9)$$

(linear interpolation function)

$\bar{x}_i, \bar{y}_i, \bar{z}_i$  - natural coordinates of node  $i$  (either 1 or -1)

$T_i(t)$  - temperature of node  $i$

$\langle H_i \rangle = \langle H_1 H_2 \dots H_8 \rangle$

$\{T_i\}$  - column vector of nodal point temperatures

Similarly, the global coordinates of any point within the element are related to the global coordinates of the nodal points by the following equation:

$$\begin{matrix} \left\{ \begin{array}{c} x \\ y \\ z \end{array} \right\} & = & \begin{bmatrix} \langle H_i \rangle & 0 & 0 \\ 0 & \langle H_i \rangle & 0 \\ 0 & 0 & \langle H_i \rangle \end{bmatrix} & \left\{ \begin{array}{c} \{x_i\} \\ \{y_i\} \\ \{z_i\} \end{array} \right\} \\ 3 \times 1 & & 3 \times 24 & 24 \times 1 \end{matrix} \quad (2.10)$$

where

$$\{x_i\}^T = \langle x_1 \ x_2 \ \dots \ x_8 \rangle$$

$$\{y_i\}^T = \langle y_1 \ y_2 \ \dots \ y_8 \rangle$$

$$\{z_i\}^T = \langle z_1 \ z_2 \ \dots \ z_8 \rangle$$

are the nodal point coordinates in global system.

When calculating the element conductivity matrix, an expression for temperature gradient within the element is needed. Differentiation of Eq. (2.8) with respect to spatial coordinates yields:

$$\begin{Bmatrix} \frac{\partial T}{\partial x} \\ \frac{\partial T}{\partial y} \\ \frac{\partial T}{\partial z} \end{Bmatrix}_{3 \times 1} = \begin{Bmatrix} \frac{\partial H_i}{\partial x} \\ \frac{\partial H_i}{\partial y} \\ \frac{\partial H_i}{\partial z} \end{Bmatrix}_{3 \times 8} \{T_i\}_{8 \times 1} = [B] \{T_i\} \quad (2.11)$$

The spatial derivatives in the natural and global coordinate systems are related by the following equation (using the chain rule of differentiation).

$$\begin{Bmatrix} \frac{\partial}{\partial \bar{x}} \\ \frac{\partial}{\partial \bar{y}} \\ \frac{\partial}{\partial \bar{z}} \end{Bmatrix}_{3 \times 1} = [J] \begin{Bmatrix} \frac{\partial}{\partial x} \\ \frac{\partial}{\partial y} \\ \frac{\partial}{\partial z} \end{Bmatrix}_{3 \times 3} \quad (2.12)$$

where

$$[J] = \begin{bmatrix} \frac{\partial x}{\partial \bar{x}} & \frac{\partial y}{\partial \bar{x}} & \frac{\partial z}{\partial \bar{x}} \\ \frac{\partial x}{\partial \bar{y}} & \frac{\partial y}{\partial \bar{y}} & \frac{\partial z}{\partial \bar{y}} \\ \frac{\partial x}{\partial \bar{z}} & \frac{\partial y}{\partial \bar{z}} & \frac{\partial z}{\partial \bar{z}} \end{bmatrix} \quad (2.13)$$

is the Jacobian matrix. From Eq. (2.10) it can be shown that:

Finally, using Eq. (2.12) in Eq. (2.11), one obtains

$$[\mathbf{B}] = \begin{bmatrix} < \frac{\partial H_i}{\partial x} > \\ < \frac{\partial H_i}{\partial y} > \\ < \frac{\partial H_i}{\partial z} > \end{bmatrix}_{3 \times 8} = [J]^{-1}_{3 \times 3} \begin{bmatrix} < \frac{\partial H_i}{\partial \bar{x}} > \\ < \frac{\partial H_i}{\partial \bar{y}} > \\ < \frac{\partial H_i}{\partial \bar{z}} > \end{bmatrix}_{3 \times 8} \quad (2.15)$$

Using the virtual work principle, it can be shown that the element conductivity matrix is given by [12]

$$\underline{\underline{K}}_m = \int_{Vol.} \underline{\underline{B}}^T \underline{k}_m \underline{\underline{B}} dv \quad (2.16)$$

8x8                  8x3    3x3    3x8

in which  $\underline{k}_m$  is the material thermal conductivity tensor of the element.

For isotropic conductivity, the above equation reduces to

$$\underline{\underline{K}}_m = \int_{Vol.} k_m \underline{\underline{B}}^T \underline{\underline{B}} dv \quad (2.17)$$

8x8                  8x3    3x8

Note that [12]

$$dv = dx \cdot dy \cdot dz = |J| d\bar{x} \cdot d\bar{y} \cdot d\bar{z} \quad (2.18)$$

where  $|J|$  is the Jacobian (determinant of  $[J]$ ). By making a change in coordinates to the natural coordinate system, Eq. (2.17) becomes:

$$\underline{\underline{K}}_m = \int_{-1}^{+1} \int_{-1}^{+1} \int_{-1}^{+1} k_m \underline{\underline{B}}^T \underline{\underline{B}} |J| d\bar{x} d\bar{y} d\bar{z} \quad (2.19)$$

8x8                  8x3    3x8

If  $k_m$  is constant throughout the element, the above integral reduces to

$$\underline{\underline{K}}_m = k_m \int_{-1}^{+1} \int_{-1}^{+1} \int_{-1}^{+1} \underline{f}(\bar{x}, \bar{y}, \bar{z}) d\bar{x} d\bar{y} d\bar{z} \quad (2.20)$$

8x8

where

$$\underline{f}(\bar{x}, \bar{y}, \bar{z}) = \underline{\underline{B}}^T \underline{\underline{B}} |J|$$

8x8                  8x3    3x8

It is impractical to integrate expression (2.20) in closed form, and approximate numerical integration methods must be used [11,12]. Although there are several numerical integration schemes available, Gauss quadrature is used in FIRES-T3. Using Gauss' Formula, expression (2.20) is evaluated as follows:

$$\int_{-1}^{+1} \int_{-1}^{+1} \int_{-1}^{+1} f(\bar{x}, \bar{y}, \bar{z}) d\bar{x} d\bar{y} d\bar{z} = \sum_{i=1}^I \sum_{j=1}^J \sum_{n=1}^N w_i w_j w_n f(\bar{x}_i, \bar{y}_j, \bar{z}_n) \quad 8x8 \quad (2.22)$$

where

$I, J, N$  - number of integration points in directions

$\bar{x}, \bar{y}, \bar{z}$ , respectively

$\bar{x}_i, \bar{y}_j, \bar{z}_n$  - coordinates of integration points

$w_i, w_j, w_n$  - weight functions

Values of the coordinates and weight functions for various values of the number of integration points are tabulated in Reference 12 and in other texts on numerical methods. In FIRES-T3 the number of integration points is taken equal to two in each direction, yielding a total of eight integration points within the element. That is:

$$I = J = N = 2$$

Then, from appropriate tables [12], it is found that

$$\bar{x}_i, \bar{y}_j, \bar{z}_n = \pm \frac{\sqrt{3}}{3}, \pm \frac{\sqrt{3}}{3}, \pm \frac{\sqrt{3}}{3}$$

and

$$w_i, w_j, w_n = 1, 1, 1$$

The computational scheme proceeds as follows. First, the spatial derivatives of  $\langle H_i \rangle$  are evaluated at integration point  $(\bar{x}_i, \bar{y}_j, \bar{z}_n)$ .

Then these derivatives are used in Eq. (2.14), along with the known global coordinates,  $(x_1, \dots, x_8)$ ,  $(y_1, \dots, y_8)$  and  $(z_1, \dots, z_8)$  of the element corners, to obtain the Jacobian matrix  $\underline{J}(\bar{x}_i, \bar{y}_j, \bar{z}_n)$  at point  $(\bar{x}_i, \bar{y}_j, \bar{z}_n)$ . This matrix is inverted and used in Eq. (2.15), along with the spatial derivatives of  $\langle H_i \rangle$  at point  $(\bar{x}_i, \bar{y}_j, \bar{z}_n)$ , to obtain the matrix  $\underline{B}(\bar{x}_i, \bar{y}_j, \bar{z}_n)$ . Finally, the determinant of the Jacobian  $|J(\bar{x}_i, \bar{y}_j, \bar{z}_n)|$  is calculated and used together with  $\underline{B}(\bar{x}_i, \bar{y}_j, \bar{z}_n)$  in Eq. (2.21) to obtain:

$$\underline{f}(\bar{x}_i, \bar{y}_j, \bar{z}_n) = \underline{B}^T(\bar{x}_i, \bar{y}_j, \bar{z}_n) \underline{B}(\bar{x}_i, \bar{y}_j, \bar{z}_n) |J(\bar{x}_i, \bar{y}_j, \bar{z}_n)| \quad (2.23)$$

8x8	8x3	3x8
-----	-----	-----

Equation (2.23) is evaluated at the eight integration points and added in Eq. (2.22) to get a total value for the integral. Substitution into Eq. (2.20) completes the derivation of element conductivity.

### 2.3.2 Two-Dimensional Isoparametric Element

The two-dimensional isoparametric element is very similar to the three-dimensional element, so the derivation below is brief to avoid redundancy.

A typical four node two-dimensional isoparametric element is shown in Fig. 2.2. The natural coordinates  $(\bar{x}, \bar{y})$  of the four corner nodes are  $(\pm 1, \pm 1)$ . Global coordinates are denoted by  $x$  and  $y$ .

The temperature at any point within the element is expressed in the natural coordinate system  $(\bar{x}, \bar{y})$  in terms of the temperatures at nodes 1 to 4 by the following equation:

$$T(\bar{x}, \bar{y}, t) = \sum_{i=1}^4 H_i(x, y) \cdot T_i(t) \quad (2.24a)$$

or, in matrix form

$$T(\bar{x}, \bar{y}, t) = \langle H_i \rangle \{T_i\} \quad (2.24b)$$

1 x 1	1 x 4	4 x 1
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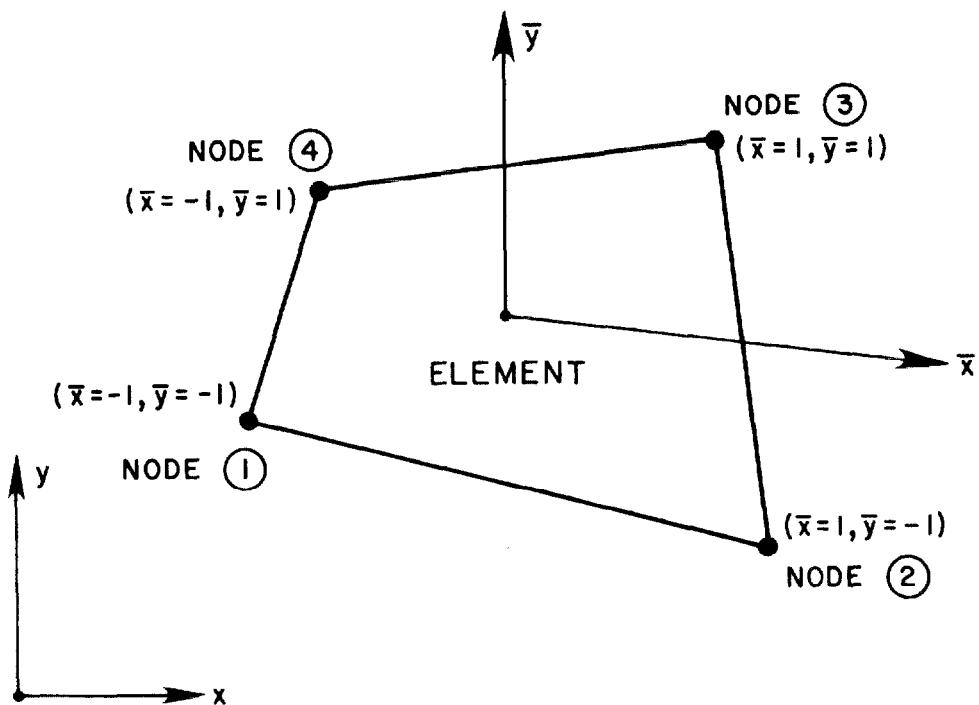


FIGURE 2.2 TWO-DIMENSIONAL ISOPARAMETRIC ELEMENT

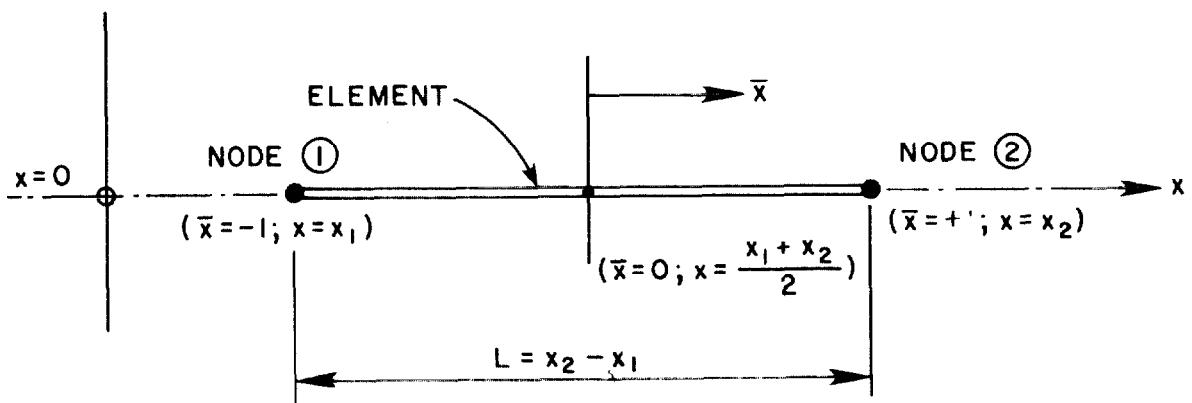


FIGURE 2.3 ONE-DIMENSIONAL ISOPARAMETRIC ELEMENT

where

$$H_i(\bar{x}, \bar{y}) = \frac{1}{4} (1 + \bar{x} \bar{x}_i) (1 + \bar{y} \bar{y}_i) \quad (2.25)$$

(linear interpolation function)

- $\bar{x}_i, \bar{y}_i$  - natural coordinates of node  $i$   
(either 1 or -1)
- $T_i(t)$  - temperature of node  $i$
- $\langle H_i \rangle$  -  $\langle H_1 H_2 H_3 H_4 \rangle$
- $\{T_i\}$  - column vector of nodal point temperatures

Similarly, the global coordinates of any point within the element are related to the global coordinates of the nodal points by the following equation:

$$\begin{Bmatrix} x \\ y \end{Bmatrix}_{2 \times 1} = \begin{bmatrix} \langle H_i \rangle & 0 \\ 0 & \langle H_i \rangle \end{bmatrix}_{2 \times 8} \begin{Bmatrix} \{x_i\} \\ \{y_i\} \end{Bmatrix}_{8 \times 1} \quad (2.26)$$

where

$$\{x_i\}^T = \langle x_1 x_2 x_3 x_4 \rangle$$

$$\{y_i\}^T = \langle y_1 y_2 y_3 y_4 \rangle$$

are the nodal point coordinates.

Differentiation of Eq. (2.24) with respect to spatial coordinates yields:

$$\begin{Bmatrix} \frac{\partial T}{\partial x} \\ \frac{\partial T}{\partial y} \end{Bmatrix}_{2 \times 1} = \begin{bmatrix} \langle \frac{\partial H_i}{\partial x} \rangle \\ \langle \frac{\partial H_i}{\partial y} \rangle \end{bmatrix}_{2 \times 4} \{T_i\}_{4 \times 1} = [B] \{T_i\} \quad (2.27)$$

The relationship between spatial and natural derivatives is

$$\begin{Bmatrix} \frac{\partial}{\partial \bar{x}} \\ \frac{\partial}{\partial \bar{y}} \end{Bmatrix} = [J] \begin{Bmatrix} \frac{\partial}{\partial x} \\ \frac{\partial}{\partial y} \end{Bmatrix} \quad (2.28)$$

2 x 1      2 x 2      2 x 1

where

$$[J] = \begin{bmatrix} \frac{\partial x}{\partial \bar{x}} & \frac{\partial y}{\partial \bar{x}} \\ \frac{\partial x}{\partial \bar{y}} & \frac{\partial y}{\partial \bar{y}} \end{bmatrix} \quad (2.29)$$

is the Jacobian matrix. From Eq. (2.26) it can be shown that

$$\begin{matrix} [J] &= & \begin{bmatrix} \frac{\partial H_1}{\partial \bar{x}} & \frac{\partial H_2}{\partial \bar{x}} & \frac{\partial H_3}{\partial \bar{x}} & \frac{\partial H_4}{\partial \bar{x}} \\ \frac{\partial H_1}{\partial \bar{y}} & \frac{\partial H_2}{\partial \bar{y}} & \frac{\partial H_3}{\partial \bar{y}} & \frac{\partial H_4}{\partial \bar{y}} \end{bmatrix} & \begin{bmatrix} x_1 & y_1 \\ x_2 & y_2 \\ x_3 & y_3 \\ x_4 & y_4 \end{bmatrix} \\ 2 \times 2 & & 2 \times 4 & 4 \times 2 \end{matrix} \quad (2.30)$$

Use of Eq. (2.28) in Eq. (2.27) yields

$$\begin{matrix} [B] &= & \begin{bmatrix} < \frac{\partial H_i}{\partial x} > \\ < \frac{\partial H_i}{\partial y} > \end{bmatrix} & = & [J]^{-1} & \begin{bmatrix} < \frac{\partial H_i}{\partial \bar{x}} > \\ < \frac{\partial H_i}{\partial \bar{y}} > \end{bmatrix} \\ 2 \times 4 & & 2 \times 4 & & 2 \times 2 & 2 \times 4 \end{matrix} \quad (2.31)$$

The element conductivity matrix is given by

$$\underline{\underline{K}_m} = \frac{\int_{\text{Vol.}} k_m \underline{\underline{B}}^T \underline{\underline{B}} dv}{4x4 \quad 4x2 \quad 2x4} \quad (2.32)$$

or integrating in the natural coordinate system

$$\underline{\underline{K}_m} = t \int_{-1}^{+1} \int_{-1}^{+1} k_m \underline{\underline{B}}^T \underline{\underline{B}} |J| d\bar{x} d\bar{y} \quad (2.33)$$

$$4x4 \quad 4x2 \quad 2x4$$

where

$$\begin{aligned} t & - \text{ thickness of element} \\ |J| & - \text{ determinant of Jacobian} \end{aligned}$$

If  $k_m$  is constant throughout the element, the above integral reduces to

$$\underline{\underline{K}_m} = k_m t \int_{-1}^{+1} \int_{-1}^{+1} \underline{\underline{f}}(\bar{x}, \bar{y}) d\bar{x} d\bar{y} \quad (2.34)$$

$$4x4 \quad 4x2 \quad 2x4$$

where

$$\underline{\underline{f}}(\bar{x}, \bar{y}) = \underline{\underline{B}}^T \underline{\underline{B}} |J| \quad (2.35)$$

$$4x4 \quad 4x2 \quad 2x4$$

Numerical integration of Eq. (2.34) by 2-point Gauss quadrature gives the following equation:

$$\int_{-1}^{+1} \int_{-1}^{+1} \underline{\underline{f}}(\bar{x}, \bar{y}) d\bar{x} d\bar{y} = \sum_{i=1}^2 \sum_{j=1}^2 \underline{\underline{f}}(\bar{x}_i, \bar{y}_j) \quad (2.36)$$

$$4x4 \quad 4x2 \quad 2x4$$

where

$$\bar{x}_i, \bar{y}_j - \text{ coordinates of integration points either } +\sqrt{3}/3 \text{ or } -\sqrt{3}/3$$

The computational scheme is as follows. First, spatial derivatives of  $\langle H_i \rangle$  are evaluated at integration point  $(\bar{x}_i, \bar{y}_i)$ . Then these derivatives are used in Eq. (2.30), along with the known global coordinates,  $(x_1, x_2, x_3, x_4)$  and  $(y_1, y_2, y_3, y_4)$ , of the element corners, to obtain the Jacobian matrix  $J(\bar{x}_i, \bar{y}_i)$  at the integration point  $(\bar{x}_i, \bar{y}_i)$ . This matrix is inverted and used in Eq. (2.31) along with the spatial derivatives of  $\langle H_i \rangle$  at point  $(\bar{x}_i, \bar{y}_i)$ , to obtain the matrix  $B(\bar{x}_i, \bar{y}_i)$ . Finally, the determinant of the Jacobian  $|J(\bar{x}_i, \bar{y}_i)|$  is calculated and used together with  $B(\bar{x}_i, \bar{y}_i)$  in Eq. (2.35) to obtain

$$\begin{matrix} f(\bar{x}_i, \bar{y}_i) \\ 4 \times 4 \end{matrix} = \begin{matrix} B^T(\bar{x}_i, \bar{y}_i) \\ 4 \times 2 \end{matrix} \begin{matrix} B(\bar{x}_i, \bar{y}_i) \\ 2 \times 4 \end{matrix} |J(\bar{x}_i, \bar{y}_i)| \quad (2.37)$$

Equation (2.37) is evaluated at the four integration points to determine the total value of the integral in Eq. (2.36). Substitution into Eq. (2.34) completes the formation of the two-dimensional element stiffness matrix.

### 2.3.3 One-Dimensional Isoparametric Element

A two node one-dimensional isoparametric element is shown in Fig. 2.3. The natural coordinates,  $\bar{x}$ , of the two nodes are  $\pm 1$ . The x-axis represents the global coordinate system for the element.

The temperature at any point within the element is expressed in the natural coordinate system ( $\bar{x}$ ) in terms of the temperature at nodes 1 and 2 by the following equation:

$$T(\bar{x}, t) = \sum_{i=1}^2 H_i(\bar{x}) \cdot T_i(t) \quad (2.38a)$$

or in matrix form

$$\begin{matrix} T(\bar{x}, t) \\ 1 \times 1 \end{matrix} = \begin{matrix} \langle H_i \rangle \\ 1 \times 2 \end{matrix} \{T_i\} \quad (2.38b) \quad \begin{matrix} 2 \\ 2 \times 1 \end{matrix}$$

where

$$H_i(\bar{x}) = \frac{1}{2} (1 + \bar{x} \bar{x}_i) \quad (2.39)$$

(linear interpolation function)

- $\bar{x}_i$  - natural coordinates of node i (either 1 or -1)  
 $T_i(t)$  - temperature of node i  
 $\langle H_i \rangle = \langle H_1 \ H_2 \rangle$   
 $\{T_i\}$  - column vector of nodal point temperatures

Similarly, the global coordinate of any point within the element is related to the global coordinates of the nodal points by the following equation:

$$x = \begin{matrix} \langle H_i \rangle \\ 1 \times 1 \end{matrix} \begin{matrix} \{x_i\} \\ 1 \times 2 \end{matrix} \quad (2.40)$$

$$\begin{matrix} & 2 \times 1 \end{matrix}$$

where

$$\{x_i\}^T = \langle x_1 \ x_2 \rangle$$

Differentiation of Eq. (2.38) with respect to  $x$  yields:

$$\frac{\partial T}{\partial x} = \begin{matrix} \frac{\partial H_i}{\partial x} \\ 1 \times 2 \end{matrix} \begin{matrix} \{T_i\} \\ 2 \times 1 \end{matrix} = [B] \begin{matrix} \{T_i\} \\ 1 \times 2 \end{matrix} \quad (2.41)$$

$$\begin{matrix} & 2 \times 1 \end{matrix}$$

Similar to the three-dimensional case, one obtains:

$$\frac{\partial}{\partial \bar{x}} = \frac{\partial x}{\partial \bar{x}} \frac{\partial}{\partial x} = J \frac{\partial}{\partial x} \quad (2.42)$$

where

$$J = \frac{\partial x}{\partial \bar{x}} \quad (2.43)$$

is the Jacobian. From Eq. (2.40) it can easily be shown that:

$$\frac{\partial \underline{x}}{\partial \bar{x}} = \frac{\partial H_1}{\partial \bar{x}} x_1 + \frac{\partial H_2}{\partial \bar{x}} x_2 = -\frac{1}{2} x_1 + \frac{1}{2} x_2 = \frac{L}{2} \quad (2.44)$$

where L is the length of the element.

Use of Eq. (2.42) in Eq. (2.41) yields:

$$[B] = \left\langle \frac{\partial H_i}{\partial x} \right\rangle = J^{-1} \left\langle \frac{\partial H_i}{\partial \bar{x}} \right\rangle = \frac{2}{L} \left\langle -\frac{1}{2} \frac{1}{2} \right\rangle = \left\langle -\frac{1}{L} \frac{1}{L} \right\rangle \quad (2.45)$$

Finally, the element conductivity matrix is given by:

$$\underline{\underline{K}_m} = \int_{Vol.} \frac{B^T}{2x2} \frac{k_m}{2x1} \frac{B}{1x1} dv = \int_{Vol.} k_m \begin{Bmatrix} -1 \\ \frac{1}{L} \\ 1 \\ \frac{1}{L} \end{Bmatrix} \left\langle -\frac{1}{L} \frac{1}{L} \right\rangle dv \quad (2.46)$$

Since in this case all the terms under the integration sign are constant, there is no need for numerical integration and the integral can be evaluated explicitly as follows:

$$\begin{aligned} \underline{\underline{K}_m} &= k_m \begin{Bmatrix} -1 \\ \frac{1}{L} \\ 1 \\ \frac{1}{L} \end{Bmatrix} \left\langle -\frac{1}{L} \frac{1}{L} \right\rangle \int_{Vol.} dv \\ &= k_m \begin{Bmatrix} -1 \\ \frac{1}{L} \\ 1 \\ \frac{1}{L} \end{Bmatrix} \left\langle -\frac{1}{L} \frac{1}{L} \right\rangle \cdot A \cdot L \quad (2.47) \\ &= \frac{k_m \cdot A}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \end{aligned}$$

where A is the area of the cross-section of the element.

## 2.4 Capacity Matrix [C]

The heat capacity associated with a node is the rate at which heat is absorbed for a unit rate of change of the temperature of that node. The capacity matrix is idealized through a system analogous to the lumping of mass in dynamic analysis. This approach has the important advantage of resulting in a diagonal capacity matrix. This lumping is achieved by delineating the volume adjacent to a node by a perimeter drawn through the centroids of the surrounding elements. Thus, the volume tributary to a node is found by adding a contribution from each element bounding the node, calculated as follows:

### A. Three-Dimensional Isoparametric Element

The contribution of an element,  $m$ , to a node,  $i$ , is given as :

$$C_{m,i} = \frac{1}{8} V_m \rho(T) C_p(T) \quad (2.48)$$

where  $V_m$  is the volume of element adjacent to node  $i$ .

### B. Two-Dimensional Isoparametric Element

The contribution of an element,  $m$ , to a node,  $i$ , is given as:

$$C_{m,i} = \frac{1}{4} V_m \rho(T) C_p(T) \quad (2.49)$$

where  $V_m = t \cdot A$  is the volume of element adjacent to node  $i$ .  $A$  is the area and  $t$  is the thickness of the quadrilateral element.

### C. One-Dimensional Isoparametric Element

The contribution of an element,  $m$ , to a node,  $i$ , is given as:

$$C_{m,i} = \frac{1}{2} V_m \rho(T) C_p(T) \quad (2.50)$$

where  $V_m = A \cdot L$  is the volume of element adjacent to node  $i$ .

Therefore, the heat capacity of node  $i$  is

$$c_i = \sum_{m=1}^M c_{m,i} \quad (2.51)$$

where

$M$  - all elements adjacent to node  $i$

Finally, the diagonal system capacity matrix  $C$  is assembled by repeating the above calculation for each node.

## 2.5 External Heat Flow Vector $\{Q\}$

The external heat flow vector term in the matrix equations (2.6) can be separated into its constituent parts as follows:

$$Q = Q_F + Q_p + Q_E \quad (2.52)$$

where

- $Q_F$  - heat flow caused by the exposure of the system to an external source (e.g., a fire) on the boundary
- $Q_p$  - a prescribed heat flow on the boundary
- $Q_E$  - heat flow from an exothermic reaction within the system (e.g., concrete hydration, wood combustion)

Each of these three terms is calculated individually in FIRES-T3 as follows:

### A. Convection and Radiation Boundary Conditions

The term  $Q_F$  is considered a function of both convective and radiative mechanisms. Fires normally affecting structures are here considered to be those in a post-flashover phase, which leads to the assumption of a uniform room temperature. The time-temperature relationship for a fire is represented by the function  $T_f(t)$ . It must be realized that this temperature is at best a pseudo-representation of a very complex phenomenon.

The boundary of the system exposed to fire is assumed to be composed of surfaces bounding adjacent nodes. Thus, the external heat flow for a node can be represented by:

$$Q_F = A_F \cdot q(T_s, T_f) \quad (2.53)$$

where

$A_F$	-	tributary area for the node
$q$	-	rate of heat flow per unit area
$T_s$	-	surface temperature
$T_f$	-	temperature of pseudo-fire

The value of  $Q_F$  for a surface node exposed to fire is assembled by considering the contributions of surrounding surfaces.

The rate of heat flow can be modeled linearly or nonlinearly:

### 1. Linear heat transfer for fire:

$$q = h(T') (T_f - T_s) \quad (2.54)$$

where

$h(T')$	-	heat transfer coefficient
$T'$	=	$(T_f + T_s)/2$

### 2. Nonlinear heat transfer from fire:

$$q = A(T_f - T_s)^N + V \cdot \sigma [a \varepsilon_f \theta_f^4 - \varepsilon_s \theta_s^4] \quad (2.55)$$

where

$A$	-	convection coefficient
$N$	-	convection power factor
$V$	-	radiation view factor
$\sigma$	-	Stefan-Boltzmann constant
$a$	-	absorption of surface

$\epsilon_f$	-	emissivity of the flame associated with fire
$\theta_f$	-	absolute temperature of fire
$\epsilon_s$	-	surface emissivity
$\theta_s$	-	absolute temperature of surface

### B. Prescribed Heat Flow Boundary Condition

In this case, the prescribed heat flow at the nodes at different times,  $Q_p$ , are input directly to the computer program FIRES-T3.

### C. Internal Heat Generation due to Exothermic Reaction

The heat flow at each node due to internal heat generation,  $Q_E$ , is assembled by considering the contribution of each element bounding the node, i.e.,

$$Q_{E,i} = \sum_{m=1}^M q_m(v, t) V_{m,i} \quad (2.56)$$

where

$Q_{E,i}$	=	internal heat flow at node i
M	=	all elements adjacent to node i which generate heat
$q_m(v, t)$	=	heat generation per unit volume in element m (time-dependent)
$V_{m,i}$	=	portion of volume of element m that is tri- butary to node i
	=	$\frac{1}{8} V_m$ for three-dimensional elements
	=	$\frac{1}{4} V_m$ for two-dimensional elements
	=	$\frac{1}{2} V_m$ for one-dimensional elements

In FIRES-T3, the heat rate function  $q_m(v, t)$  is input directly. Cooling is indicated by a negative sign.

## 2.6 Time Integration of Matrix Equations

The unknown nodal temperature vector,  $\{T\}$ , in the matrix equation (2.6) is a function of time,  $\{T(t)\}$ . Hence, this equation is actually a first order differential equation in  $n$  dimensions and must be further discretized by step-by-step integration. The continuous nodal temperature history  $\{T(t)\}$  is represented by a finite sequence of temperature distributions  $\{T(t_0)\}$ ,  $\{T(t_1)\}$ ,  $\{T(t_2)\}$ , . . . . By assuming that the variation of the temperature with time is linear within each time step (Fig. 2.4), the temperature rate of change  $\{\dot{T}\}$  at any time  $t_i$  can be approximated in terms of nodal temperatures:

$$\dot{\{T(t_i)\}} = \{T(t_i) - T(t_{i-1})\}/\Delta t_i \quad (2.57)$$

where  $\Delta t_i$  is the time step between  $t_{i-1}$  and  $t_i$ . The differential equations (2.6) can now be reduced to a set of linear algebraic equations in the independent variable  $\{T(t_i)\}$ . There will be a set of equations for each time  $t_i$ ,  $i = 1, 2, \dots$ , and the step-by-step solution of each set results in an approximation to the nodal temperature history  $\{T(t)\}$  and, using Equations (2.8), (2.24) or (2.38), an approximation to the overall temperature distribution history  $T(x, y, x, t)$ ,  $T(x, y, t)$  or  $T(x, t)$ . The accuracy of this finite element step-by-step integration approximation increases with the number of elements and with the smallness of time steps.

To facilitate discussion of the solution scheme and the incorporation of boundary conditions into the equations, let Eq. (2.6) be partitioned in the following form:

$$\left[ \begin{array}{c|c} C_{aa} & 0 \\ \hline 0 & C_{bb} \end{array} \right] \left\{ \begin{array}{l} \dot{T}_a(t_i) \\ \dot{T}_b(t_i) \end{array} \right\} + \left[ \begin{array}{c|c} K_{aa} & K_{bb} \\ \hline K_{ba} & K_{bb} \end{array} \right] \left\{ \begin{array}{l} T_a(t_i) \\ T_b(t_i) \end{array} \right\} = \left\{ \begin{array}{l} Q_a(t_i) \\ Q_b(t_i) \end{array} \right\} \quad (2.58)$$

where

$T_a$  - unknown nodal temperatures

$T_b$  - known (prescribed) nodal temperatures

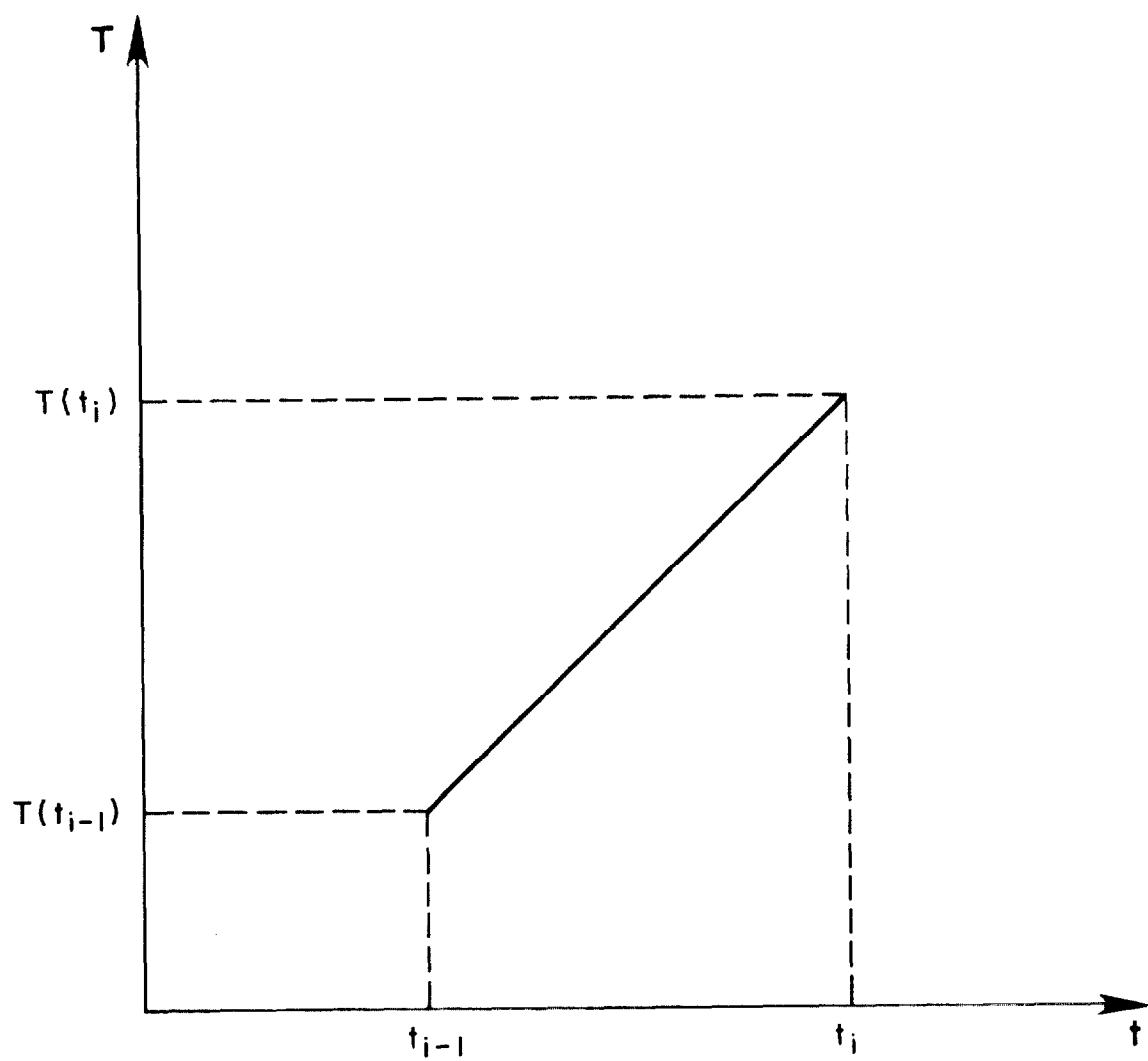


FIGURE 2.4 VARIATION OF TEMPERATURE WITH TIME WITHIN A TIME INCREMENT

$Q_a$	-	known external nodal heat flow
$Q_b$	-	unknown nodal heat flow

Static condensation can be used to remove known temperature nodes from the equations, resulting in:

$$[C_{aa}] \{\dot{T}_a(t_i)\} + [K_{aa}] \{T_a(t_i)\} = \{Q_a(t_i)\} - [K_{ab}]\{T_b(t_i)\} \quad (2.59)$$

Substitution of Eq. (2.57) into Eq. (2.59) yields:

$$[K_{aa} + \frac{C_{aa}}{\Delta t_i}] \{T_a(t_i)\} = \{Q_a(t_i) - [K_{ab}]\{T_b(t_i)\} + \frac{C_{aa}}{\Delta t_i} \{T_a(t_{i-1})\}\} \quad (2.60a)$$

or in compact form

$$[K]^* \{T_a(t_i)\} = \{Q(t_i)\}^* \quad (2.60b)$$

where

$[K]^*$  - effective conductivity matrix

$\{Q(t_i)\}^*$  - effective heat flow vector

The nodal temperatures at  $t_i$ ,  $T(t_i)$ , are found directly by solving the above set of linear algebraic equations.

Both  $K^*$  and  $Q^*$  can be functions of the current temperature  $T(t_i)$ , since material properties and fire boundary conditions can be temperature-dependent. There are two basic approaches that can be used to resolve this problem.

1. Use the temperature distribution from the previous time step,  $T(t_{i-1})$ , to calculate the necessary values and thus solve Eq. (2.60) directly, or
2. Use an iterative solution technique that allows the necessary variables to be continually revised on the basis of the current solution.

The type of numerical solution technique often used in solving linear algebraic equations - i.e. triangularization of  $\underline{K}^*$  and then the back substitution of  $\underline{Q}^*$  in order to obtain  $\underline{T}(t_i)$  - offers an additional variation on the previously mentioned options. It is possible to derive  $\underline{K}^*$  on the basis of the previous temperature distribution, triangularize  $\underline{K}^*$ , and then, with the use of an iterative technique, achieve a more accurate temperature vector in the determination of  $\underline{Q}^*$ . This separation into triangularization and back substitution is ideal for use in simulating fire phenomena since  $\underline{Q}^*$  is extremely sensitive to the surface temperature of the structure. The effective conductivity matrix  $\underline{K}^*$  tends to be far less sensitive to temperature than  $\underline{Q}^*$ . FIRES-T3 contains the option of either solving the entire problem ( $\underline{K}^*$  and  $\underline{Q}^*$ ) iteratively, or considering only the fire boundary condition term ( $\underline{Q}^*$ ) in the iteration.

To help accelerate convergence in the iterative processes, an over-convergence factor,  $\beta$ , is used in estimating the temperature distribution of the next iteration,  $j + 1$ .

$$\underline{T}^{j+1}(t_i) = \underline{T}^j(t_i) + \beta (\underline{T}^j(t_i) - \underline{T}^{j-1}(t_i)) \quad (2.61)$$

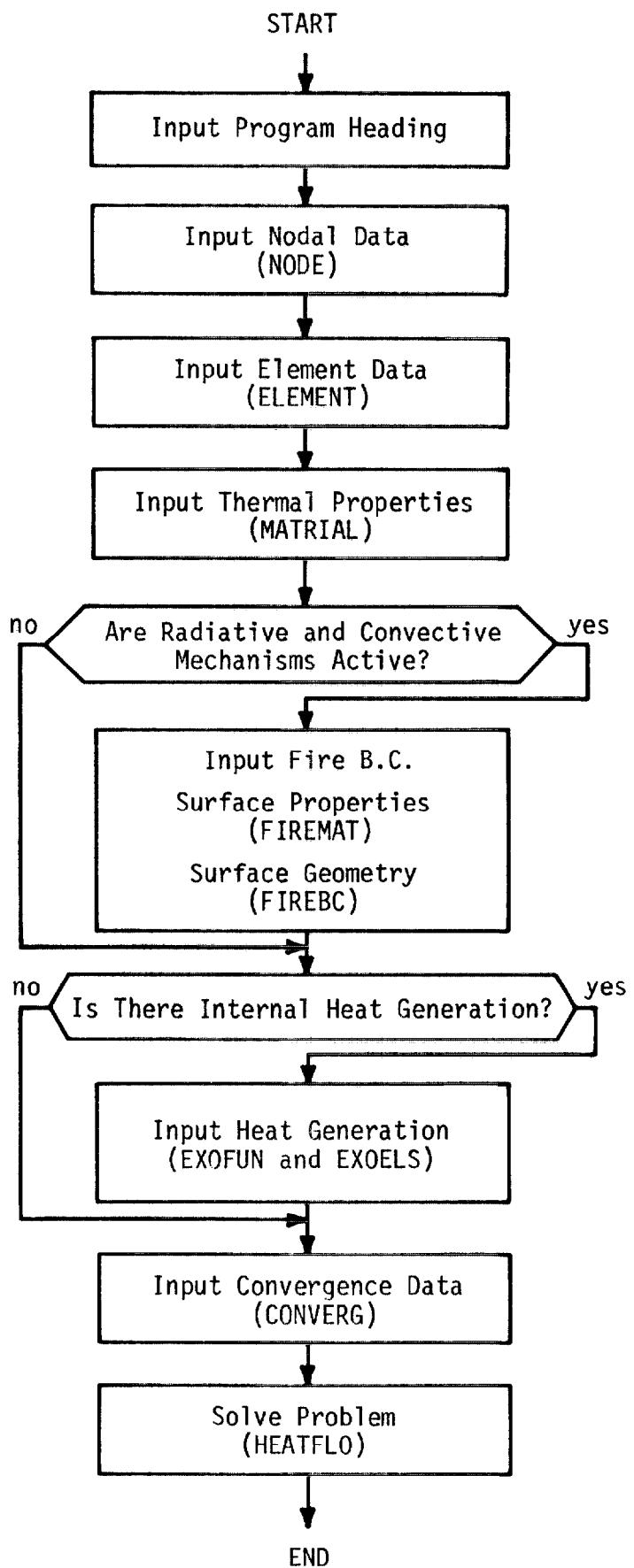
Experience has shown that  $\beta$  should vary from -0.10 to -0.40 in the case of the nonlinear fire condition. Convergence is achieved when the temperature distribution of two successive iterations coincide within a prescribed level of error expressed as

$$\frac{2 \cdot | T^j(t_i) - T^{j-1}(t_i) |}{| T^j(t_i) + T^{j-1}(t_i) |} < \text{permissible error} \quad (2.62)$$

The step-by-step assembly and solution of Eq. (2.60) gradually traces out the temperature history in the structure.

The structure of FIRES-T3 is shown in the following flow charts. Fig. 2.5 shows the organization of the main subroutine (FIRES-T3) which inputs the data describing the problem. The subroutine which controls step-by-step integration (HEATFLO) is presented in Fig. 2.6.

An important feature of FIRES-T3 is that all system variables are



**FIGURE 2.5 FLOW CHART FOR PROGRAM FIRES-T3**

(Titles in Parenthesis are Names of Subroutines)

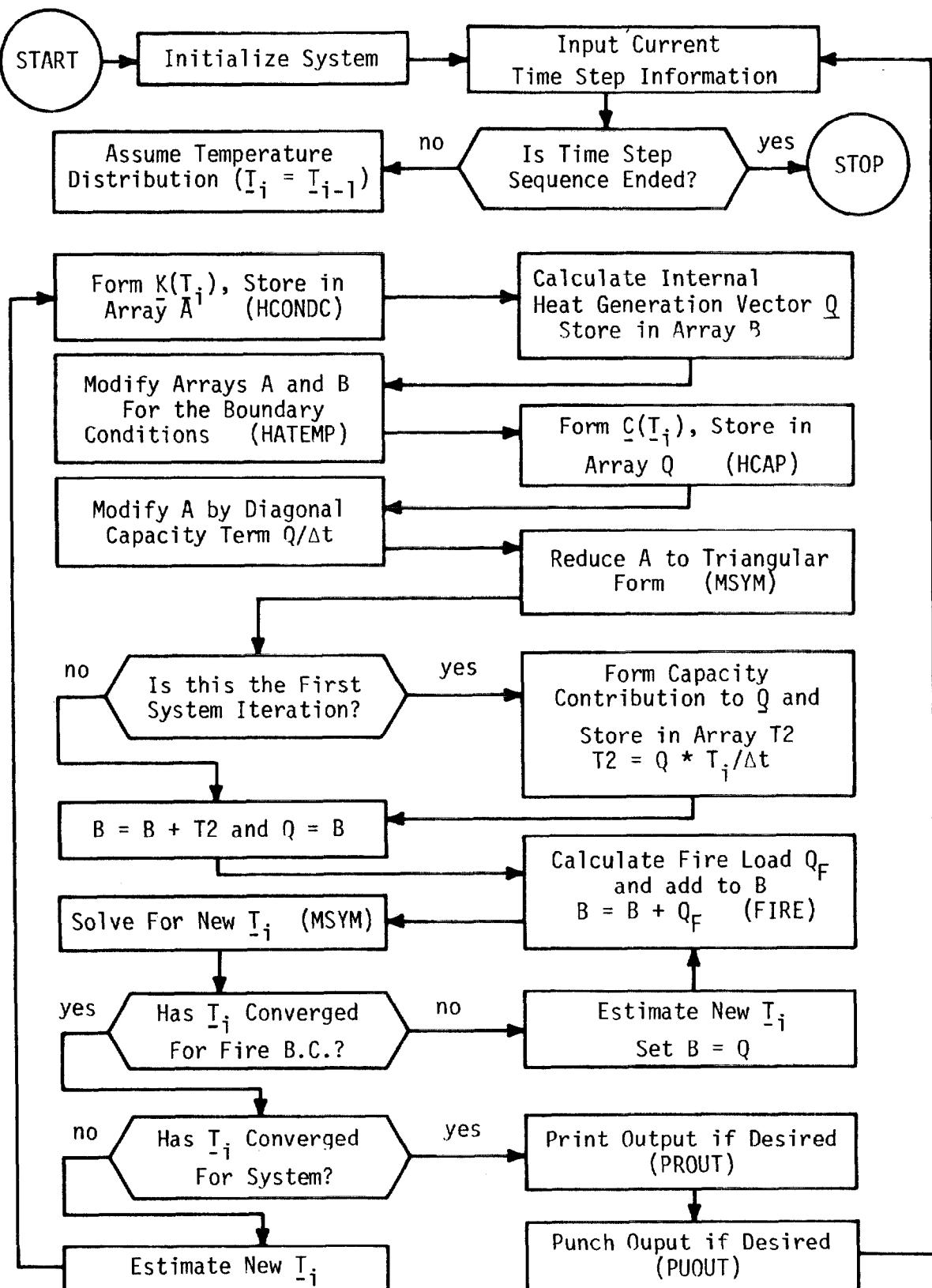


FIGURE 2.6 FLOW CHART FOR SUBROUTINE HEATFLO ASSUMING FIRE BOUNDARY CONDITIONS

(Titles in Parenthesis are Names of Program Subroutines)

dynamically dimensioned. That is, the amount of computer storage space allotted for each variable is specified based on the size of the problem being analyzed. The organization of this dynamic dimensioning scheme is shown in Table 2.1.

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<u>ADDRESS</u>	<u>DIMENSION</u>	<u>ARRAY</u>	<u>CONTENTS</u>
N1=1	• NUMNP	X	X-coordinates of nodes
N2	• NUMNP	Y	Y-coordinates of nodes
N3	• NUMNP	Z	Z-coordinates of nodes
ND0	• NUMNP	KODE	Indicator for type of boundary condition (Flow or Temperature)
ND1	• NUMNP	ID or D	
ND2	• NUMNP	MA	
ND3	• NUMNP	T1	
ND4	• NUMNP	T2	
ND5	• NUMNP	T3	

TABLE 2.1 DYNAMICALLY DIMENSIONED STORAGE ALLOCATION FOR FIRES-T3

<u>ADDRESS</u>	<u>DIMENSION</u>	<u>ARRAY</u>	<u>CONTENTS</u>
N4	2 * NEL1D + 4 * NEL2D + 8 * NEL3D	LM	Nodal points of elements
N5	NUMEL	MMTYPE	Material types of elements
N6	NEL1D	BAREA	Cross-sectional areas of one-dimensional elements
N7	NEL2D	THICK	Thickness of two-dimensional elements
NV	NEL3D	VOLUME	Volumes of three-dimensional elements
N8	6 * NMAT	MATL	Control data for material properties
N9	NSTORE	XYS	Storage for material properties
N10	NBCMAT	MAT	Control data for fire B.C. properties
N11	NSTORE	FXYS	Fire B. C. properties

TABLE 2.1 (cont.)

<u>ADDRESS</u>	<u>DIMENSION</u>	<u>ARRAY</u>	<u>CONTENTS</u>
N12	• NUMFBC	LI	
N13	• NFBC2D + NFBC3D	LJ	
N14	• NFBC3D	LK	
N15	• NFBC3D	LL	
N16	• NUMFBC	LMAT	Fire boundary type
N17	• NUMFBC	LFIRE	Fire number
33			
N18	• NUMFBC	AIJKL	Area of surface segment
N19	• NFBC1D + NFBC2D	LELEM	Element bounding surface segment
N20	• 3 * NQINT	IEXO	Control data for internal heat generation curves

TABLE 2.1 (cont.)

<u>ADDRESS</u>	<u>DIMENSION</u>	<u>ARRAY</u>	<u>CONTENTS</u>
N21	• NSTORE	EXYS	Internal heat generation curves
N22	• NINT	IEL	Elements with internal heating
N23	• NINT	IMAT	Heating curve for each element
N24	• NINT	VEL	Volume of each element
N25	• NUMNP	Q	Heat flow vector
N26	• NUMNP	T	Temperature vector
N27	• NUMNP	B	Loading vector used in analysis
N28	• NUMEL	AT	Element temperatures
N29	• NUMNP * NBAND	A	Effective conductivity matrix
NTOTAL			

TABLE 2.1 (cont.)

### 3. COMMENTARY

FIREST-3 and its predecessor FIREST have been used to solve a wide variety of heat flow problems [4, 7]. Experience gained in these analyses is drawn upon to present here a brief commentary on practical aspects of using the program.

Few difficulties are encountered in thermal analyses governed by specified temperature or heat flow boundary conditions. The principal factors which determine the effective use of the program are the layout of the finite element mesh and the selection of time-step intervals. On the one hand both must be fine enough to properly model the thermal behavior of the structure, and on the other as few elements as possible should be used since computational effort increases roughly with the cube of the number of nodes. Also, nodes should be numbered in such a way as to minimize the bandwidth of the resulting matrix equations. Time-step size can vary during the time integration to most efficiently reflect the expected rate of temperature change.

The heat flow problem is nonlinear whenever conductivity or heat capacity are temperature-dependent. However, with models of thermal properties used here this nonlinearity is negligible within a time step and adequate results are obtained by a linearized analysis (number of iterations = 0), thereby greatly reducing solution cost. When boundary conditions are linear, convergence is effected in a few iterations with little likelihood of instability. However, the fire boundary condition, including both conduction and radiation terms, is highly nonlinear and in order to ensure convergence of an iterative solution the time step size must be kept quite small, since numeric instability in the region of the fire surface can result when fire temperatures are high. A higher-order nonlinear technique (such as Newton-Raphson iteration) would improve stability and convergence rate of the solution and may be incorporated into a future version of FIREST-3. Overconvergence factors also improve the stability of the nonlinear iteration and are recommended when using the nonlinear fire boundary condition. When divergence occurs at some time step in an analysis, it is best to shorten the time step size for the remaining part of the analysis and restart the step-by-step solution from the last converging time step. Therefore,

whenever it is impossible to know a priori the time step sizes needed, nodal temperatures should be punched at each time step in case a restart is necessary.

The fire environment in FIRES-T3 is modeled through a pseudo-fire, defined by a time-temperature curve, and the convective and radiative heat flow terms in the fire boundary condition. The phenomena associated with heat transfer in the turbulent environment of a fire are difficult to model exactly. The possibility of including the temperature dependence of the related parameters may be included, and the detailed temperature and flow fields in a fire compartment may be modeled by subdividing the fire into zones with different characteristics. In the pseudo-fire concept the critical parameters appear to be the emissivity of both the flame and the surface of the structural element. This assumes that at the temperatures normally associated with fires, the radiative heat transfer mechanism predominates. At present, a value of 0.9 is used for the emissivity of concrete. The value of flame emissivity appears to be more uncertain with potential values ranging from 0.3 to 0.9. Current work indicates a possible dependency of the flame emissivity on the particular content of the flame, which may vary greatly from the controlled fire of a test furnace to that of the uncontrolled environment of an actual fire. In addition, radiation sources, compartment enclosure surfaces, may contribute to the overall fire boundary condition problem.

The radiation component of heat transfer at a boundary of an element exposed to fire depends on the spatial configuration of the enclosure, the geometry of the element under consideration, and the emissivities of the fire, the element surface, and the enclosure surface. For concrete structural elements surrounded by fire the effective emissivity  $\epsilon$  falls in the range of 0.3 to 0.9 [9]. The radiation boundary condition in Equation (2.55) is an attempt to represent the influence of the fire and the element surface emissivities on the radiative heat transfer. This approximation is sensitive to the selection of  $\epsilon_f$  and  $\epsilon_s$  values. Unless it can be shown by comparison with experimental data that this approximation give reliable results, it is recommended that for concrete structures exposed to fire both values of  $\epsilon_f$  and  $\epsilon_s$  where  $0.5 < \epsilon < 0.7$  be replaced by effective emissivity  $\epsilon$ .

The capability of modeling internal heat generation in FIRES-T3 introduces no numerical complications or sensitivities. This option could be used to include the effects of internal combustion in a fire analysis [5]. Also, it could solve a variety of other temperature problems such as determining temperature rise in mass concrete due to hydration of cement paste [4].

#### ACKNOWLEDGEMENTS

The research described in this report has been conducted as part of a study on structural response to fire carried out at the University of California, Berkeley, under the National Science Foundation RANN Program, Grant No. ERT70-01080 A05 (formerly Grant No. GI-43). Partial support for publication of this report was provided by National Bureau of Standards Grant No. NBS-G7-9006-10/77. The results of the study and their interpretation have not yet been reviewed by the National Bureau of Standards and represent only the preliminary conclusions of the investigators. The support of the sponsors is gratefully acknowledged.

The work presented in this report is an extension of earlier work by Zuhayr Nizamuddin [7], Jim Becker et al [1] and Hani Bizri [3].

## REFERENCES

1. Becker, J. M., Bizri, H., and Bresler, B., "FIRES-T - A Computer Program for the FIre REsponse of Structures - Thermal," Report No. UCB FRG 74-1, Fire Research Group, Structural Engineering and Structural Mechanics, Department of Civil Engineering, University of California, Berkeley, 1974.
2. Becker, J. and Bresler, B., "FIRES-RC - A Computer Program for the FIre REsponse of Structures - Reinforced Concrete Frames," Report No. UCB FRG 74-3, Fire Research Group, Structural Engineering and Structural Mechanics, Department of Civil Engineering, University of California, Berkeley, 1974.
3. Bizri, H., "Structural Capacity of Reinforced Concrete Columns Subjected to Fire Induced Thermal Gradients," Report No. UC SESM 73-1, Structural Engineering Laboratory, University of California, Berkeley, 1973.
4. Bresler, B., and Iding, R. H., "Effects of Normal and Extreme Environment on Reinforced Concrete Structures," Report No. UC SESM 77-4, Department of Civil Engineering, University of California, Berkeley, 1977.
5. Knudsen, R. M., Performance of Structural Wood Members Exposed to Fire, Ph. D. Dissertation, Department of Civil Engineering, University of California, Berkeley, 1973.
6. Iding, R. H., Bresler, B., and Nizamuddin, Z., "FIRES-RC II - A Computer Program for the FIre REsponse of Structures - Reinforced Concrete Frames - Revised Version," Report No. UCB FRG 77-8, Fire Research Group, Structural Engineering and Structural Mechanics, Department of Civil Engineering, University of California, Berkeley, 1977.
7. Nizamuddin, Z., Thermal and Structural Analysis of Reinforced Concrete Slabs in Fire Environments, Ph. D. Dissertation, Department of Civil Engineering, University of California, Berkeley, 1976.
8. Wickstrom, U., "A Numerical Procedure for Calculating Temperature in Hollow Structures Exposed to Fire," Report No. UCB FRG 77-9, Department of Civil Engineering, University of California, Berkeley, 1977.
9. Sahota, M. S. and Pagni, P. J., "Temperature Fields in Structural Elements Subjected to Fire," Report No. UCB FRG 75-19, Department of Mechanical Engineering, University of California, Berkeley, 1977.
10. Wilson, E. L., and Nickell, R. E., "Application of the Finite Element Method to Heat Conduction Analysis," Nuclear Engineering and Design, Vol. 4, 1966, pp. 276-286.

11. Wilson, E. L., and Farhoomand, I., "Non-Linear Heat Transfer Analysis of Axisymmetric Solids," Report No. UC SESM 71-6, Structural Engineering Laboratory, University of California, Berkeley, April 1971.
12. Zienkiewicz, O. C., The Finite Element Method in Structural and Continuum Mechanics, McGraw-Hill, London, 1967.
13. Zienkiewicz, O. C., "Isoparametric and Allied Numerically Integrated Elements - A Review," Numerical and Computer Methods in Structural Mechanics, S. J. Fenres, et. al. (eds.), Academic Press, N. Y., 1973, pp. 13-42.

APPENDIX A

INPUT INSTRUCTIONS FOR FIRES-T3

APPENDIX A - INPUT INSTRUCTIONS FOR FIRES-T3

Contents

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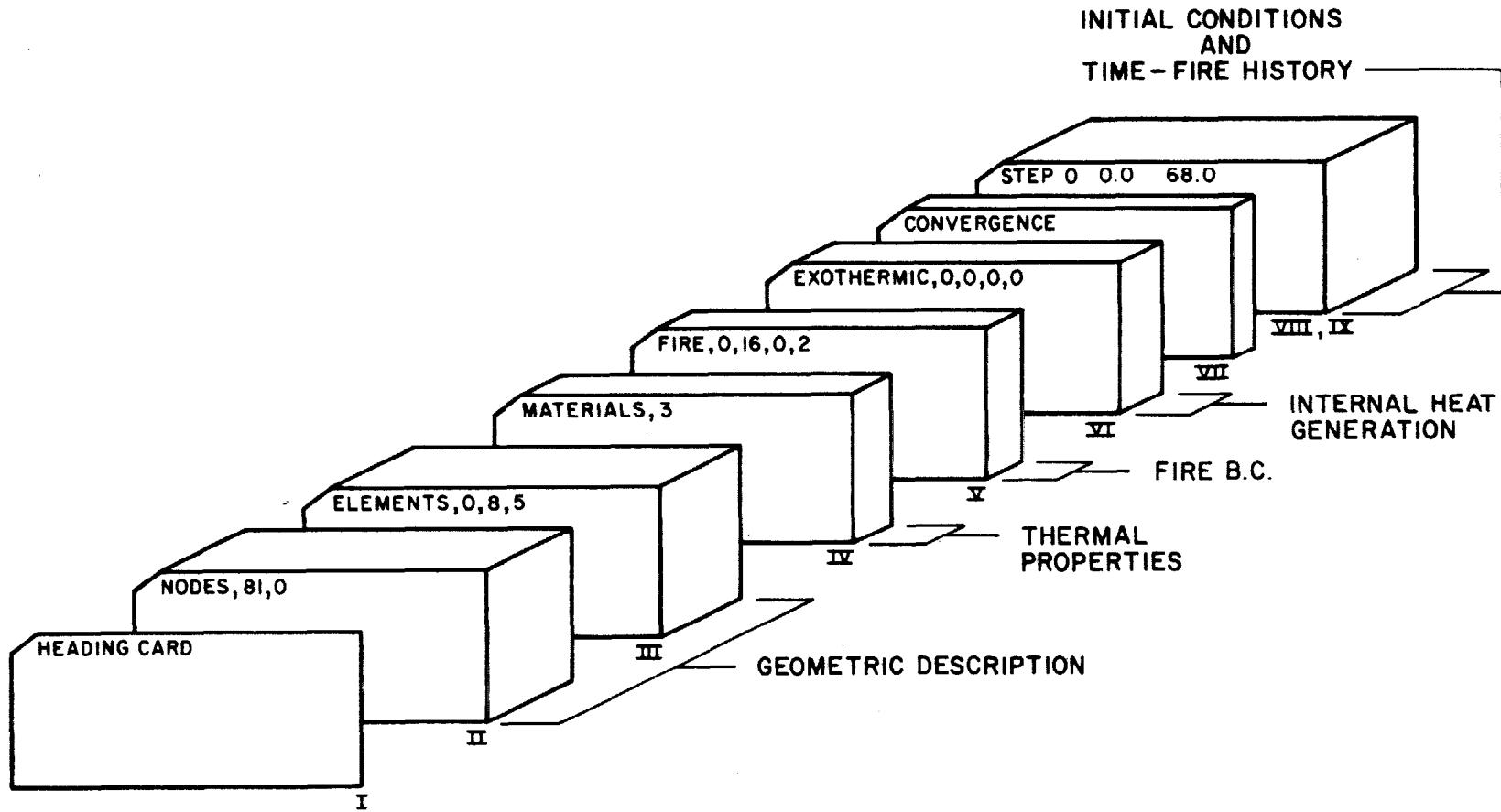


FIGURE A.1 TYPICAL INPUT DECK FOR FIRES-T3

## I. HEADING CARD (6A10)

NOTE/ Begin each new data case with a heading card. To halt the program insert two blank cards instead of a heading card.

## II. NODAL DATA

### A. Control Card (Alphanumeric)

NODES, N1, N2

note	field	variable	entry
(1)	NODES	--	Enter the word "NODES"
	N1	NUMNP	Number of nodal points
	N2	NTBC	Number of nodal points with a specified temperature boundary condition

NOTE/ This is an alphanumeric control card containing both a key word (NODES) to identify the block of input data to follow and control parameters (N1 and N2) for that block of data. Alphanumeric control cards are left-justified with no blanks in the list. Note the examples in Fig. A.1.

### B. Nodal Cards (I5,3E10.0)

note	columns	variable	entry
(1)	1-5	N	Nodal point number
(2)	6-15	X(N)	X-coordinate
	16-25	Y(N)	Y-coordinate
(3)	26-35	Z(N)	Z-coordinate

### NOTES/

- (1) Nodal point cards must be in numerical sequence. If cards are omitted, the missing nodes are generated at equal intervals along a straight line between the bounding nodal points.
- (2) Nodal coordinates may be input using any system of units. However, this same system must be used to define all other input quantities that follow.
- (3) In purely two-dimensional heat flow problems, leave the Z-coordinate blank.

C. Specified Temperature Nodes (1615)

Omit this card if (N2) on the control card is zero

note	columns	variable	entry
(1)	1-5	ID(1)	Node number of first node in which temperature is fixed as a boundary condition
	6-10	ID(2)	Node number of second specified temperature node
	.	.	.
	.	.	.
	.	.	.
	.	.	.
(2)	.	ID(N2)	Continue until all N2 nodes are input

NOTES/

- (1) Three types of boundary conditions are possible in FIRES-T3:
  - 1) Temperature specified
  - 2) Heat flow specified
  - 3) Fire B.C.

If the first type of B.C. is to be used, all nodes with a specified temperature must be identified on this card.  
Actual temperatures are input later in Data Block IX. All boundary nodes not specified here are assumed to have a boundary condition of type 2 or type 3.
- (2) Input 16 temperature nodes per card.

III. ELEMENT DATA

A. Control Card (Alphanumeric)

ELEMENTS, N1, N2, N3

note	field	variable	entry
(1)	ELEMENTS	--	Enter the word "ELEMENTS"

(2)	N1	NELID	Number of one-dimensional elements
(3)	N2	NEL2D	Number of two-dimensional elements
(4)	N3	NEL3D	Number of three-dimensional elements

NOTES/

- (1) This is an alphanumeric control card with key word (ELEMENTS) and control parameters (N1, N2 and N3) as in Fig. A.1.
- (2) Enter the total number of one-dimensional elements to be used. If 1-D elements are not to be used, enter "0" as N1.
- (3) Enter the number of two-dimensional elements. If 2-D elements are not used, enter "0" as N2.
- (4) Enter the number of three-dimensional elements. If 3-D elements are not used, enter "0" as N3.

A single mesh may utilize all three of the above element types.

B. One-Dimensional Element Cards (4I5, F10.0)

Omit these cards if (N1) on the control card above is zero

note	columns	variable	entry
(1)	1-5	NUM	Element number GE.1 and LE. NELID
(2)	6-10	I	Nodal point I
	11-15	J	Nodal point J
(3)	16-20	MTYPE	Material identification number
(4)	21-30	BAREA	Cross-sectional area of one-dimensional element

NOTES/

- (1) Elements must be input in ascending element number order. If element cards are missing, the program generates the missing elements by incrementing NUM and nodal points I and J. A generated element assumes the material type and cross-sectional area of the element immediately preceding it. The last one-dimensional element in the mesh cannot be generated.

- (2) The program uses isoparametric bar elements defined by their end nodes I and J, as shown in Fig. A.2(a). Enter the global node number of each end node. These one-dimensional bar elements may be used in a two- or three-dimensional nodal mesh. Any orientation in space is permissible.
- (3) One or more sets of thermal material properties are input in the next data block, each of them labelled by an identification number. Enter the identification number of the material this element is composed of.
- (4) Enter the cross-sectional area of the one-dimensional element, as shown in Fig. A.2. For a purely one-dimensional analysis, set (BAREA) equal to unity (1.0).

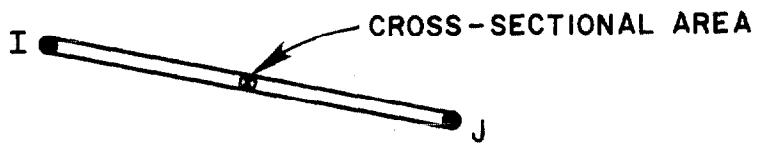
C. Two-Dimensional Element Cards (6I5, F10.0)

Omit these cards if (N2) on the control card above is zero.

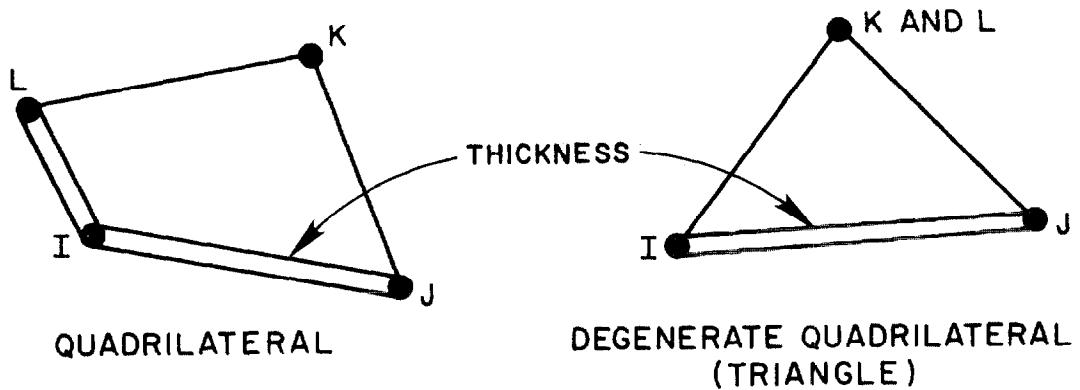
note	columns	variable	entry
(1)	1-5	NUM	Element number GE.1 and LE. NEL2D
(2)	6-10	I	Nodal point I
	11-15	J	Nodal point J
	16-20	K	Nodal point K
	21-25	L	Nodal point L
(3)	26-30	MTYPE	Material identification number
(4)	31-40	THICK	Thickness of two-dimensional element EQ.0.0, default of 1.0 used

NOTES/

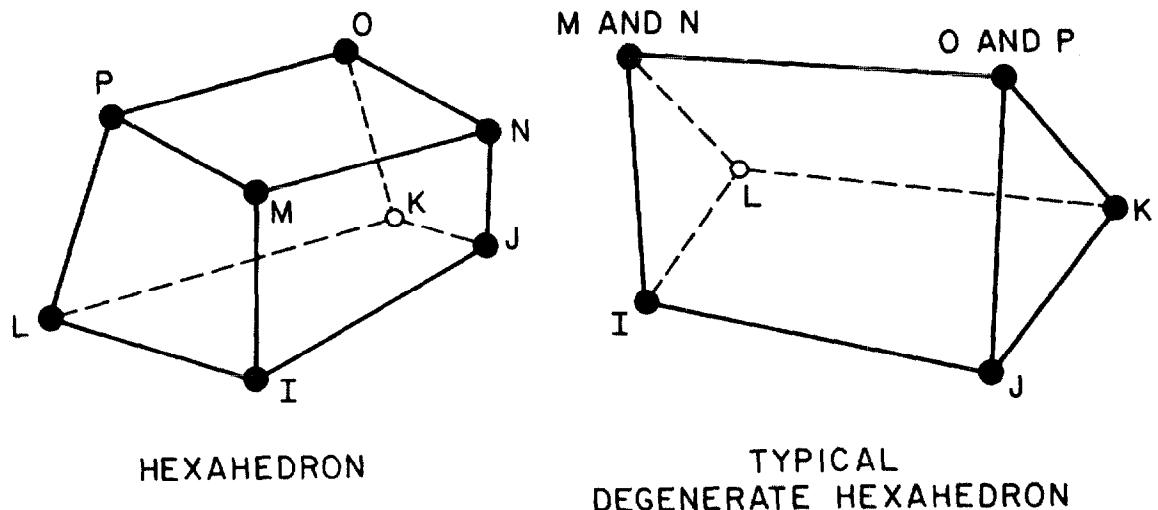
- (1) Two-dimensional elements must be input in ascending element number order. Start with "1" even if one-dimensional elements have also been used. If element cards are missing, the program generates the missing elements by incrementing NUM and nodal points I, J, K, and L. A generated element assumes the material type and thickness of the element immediately preceding it. The last two-dimensional element in the mesh cannot be generated.



(a) ONE-DIMENSIONAL ELEMENT



(b) TWO-DIMENSIONAL ELEMENTS



(c) THREE-DIMENSIONAL ELEMENTS

FIGURE A.2 ELEMENTS AVAILABLE IN FIRES-T3

- (2) The program uses quadrilateral isoparametric elements defined by their corner nodes (I, J, K, L). Enter the global node number corresponding to each of these four corner nodes in counterclockwise order, as shown in Fig. A.2(b). Triangular elements can be formed by specifying a degenerate quadrilateral, i.e., by letting K and L be defined by the same global nodal point (see Fig. A.2(b)).

These two-dimensional elements may be used in three-dimensional analyses. It is most economical to define two-dimensional elements in an (X, Y) plane ( $Z = \text{constant}$ , not necessarily zero). However, it is also possible to define two-dimensional elements in an (X, Z) plane or in a (Y, Z) plane. Two-dimensional flow in skewed planes must be modelled by a layer of three-dimensional elements.

- (3) One or more sets of thermal material properties are input in the next data block, each of them labelled by an identification number. Enter the identification number of the material this element is composed of.
- (4) Enter the thickness of the two-dimensional quadrilateral (or triangle), as shown in Fig. A.2(b). For a purely two-dimensional analysis, set (THICK) equal to unity (1.0).

#### D. Three-Dimensional Element Cards (1015)

Omit these cards if (N3) on the control card above is zero

note	columns	variable	entry
(1)	1-5	NUM	Element number GE.1 and LE. NEL3D
(2)	6-10	I	Nodal point I
	11-15	J	Nodal point J
	16-20	K	Nodal point K
	21-25	L	Nodal point L
	26-30	M	Nodal point M
	31-35	N	Nodal point N
	36-40	O	Nodal point O
	41-45	P	Nodal point P
(3)	46-50	MTYPE	Material identification number

NOTES/

- (1) Three-dimensional elements must be input in ascending element number order. Start with "1" even if one-dimensional or two-dimensional elements have also been used. If element cards are missing, the program generates the missing elements by incrementing NUM and nodal points I, J, K, L, M, N, O, and P. A generated element assumes the material type of the element immediately preceding it. The last element in the mesh cannot be generated.
- (2) The program uses hexahedral isoparametric elements defined by their corner nodes (I, J, K, L, M, N, O, P). Enter the global node number corresponding to each of these eight corner nodes in counterclockwise order, bottom layer first, as shown in Fig. A.2(c). Pentahedral or tetrahedral elements may be formed by specifying a degenerate hexahedron, i.e., by letting certain corner nodes be defined by the same global nodal point (see Fig. A.2(c)). Three-dimensional elements are comparatively expensive to form and lower order elements (one-dimensional or two-dimensional) of appropriate thickness should be used in those portions of a solid where heat flow is one- or two-dimensional.
- (3) One or more set of thermal material properties are input in the next data block, each of them labelled by an identification number. Enter the identification number of the material this element is composed of.

IV. THERMAL MATERIAL PROPERTY DATA

A. Control Card (Alphanumeric)

MATERIALS, N1

note field variable entry

(1) MATERIALS -- Enter the word "MATERIALS"

N1 NMAT Number of different material types

NOTE/ This is an alphanumeric control card with key word (MATERIALS) and main control parameter (N1), as in Fig. A.1.

B. Material Data

Input the following set of cards for each material type:

### 1. Control Card (3I5)

note	columns	variable	entry
(1)	1-5	MK	Number of points used to define heat conductivity function EQ.0, constant conductivity
(1)	6-10	MCP	Number of points used to define specific heat capacity function EQ.0, constant heat capacity
(1)	11-15	MD	Number of points used to define density function EQ.0, constant density

NOTE/

- (1) Each material type is characterized by three material parameters: thermal conductivity, thermal heat capacity, and density. Each of these may be input as a constant or as a tabular function of temperature. At least two points are needed to define each function, and linear interpolation is used between points.

### 2. Heat Conductivity (8E10.0)

note	columns	variable	entry
(1)	1-10	X(1)	Temperature of point 1
	11-20	Y(1)	Value of conductivity function at point 1
	21-30	X(2)	Temperature of point 2
	.	.	.
	.	.	.
	.	.	.
	.	Y(MK)	Continue until all (MK) points are input.

NOTE/

- (1) Input the table that defines the heat conductivity function. Each point is described by an ordered pair ( $X$ ,  $Y$ ), where  $X$  is temperature and  $Y$  is the value of heat conductivity at that temperature, as shown in Fig. A.3. Input 4 such pairs per card and use as many cards as necessary. The table for the function must be defined over the entire temperature range to be considered in the solution process - i.e. extrapolation below the lowest point or beyond the highest point is not permitted.

If conductivity was specified constant with temperature (MK.EQ.0) enter the constant value in columns 1-10.

Units for material properties must be consistent with units used for other input quantities (nodal coordinates, temperature, time step size, etc.).

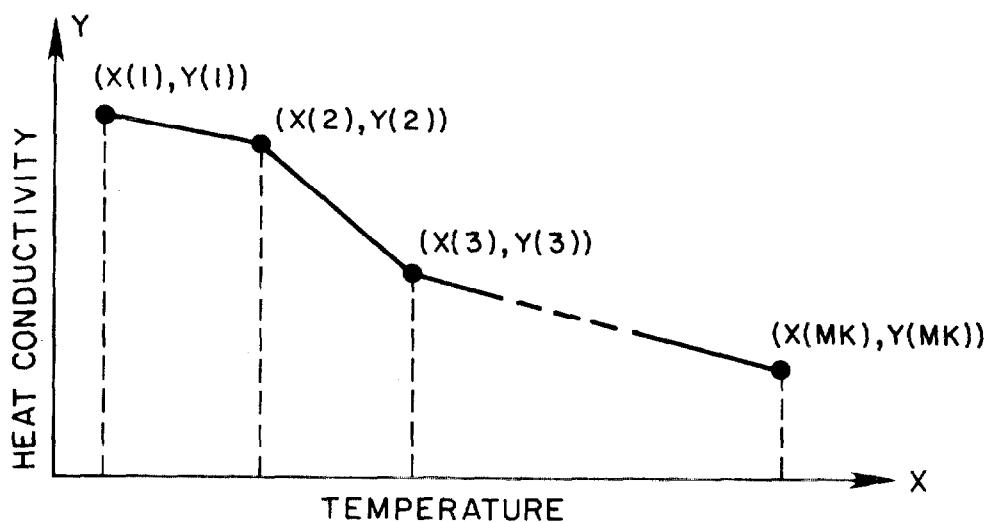


FIGURE A.3 MATERIAL PROPERTY TABLE FORMAT

3. Specific Heat Capacity (8E10.0)  
(same as above)

4. Density (8E10.0)

(same as above)

V. FIRE BOUNDARY CONDITION DATA

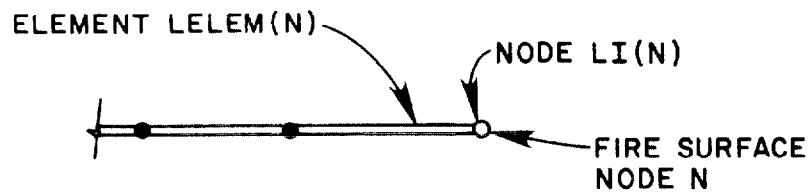
A. Control Card (Alphanumeric)

FIRE, N1, N2, N3, N4

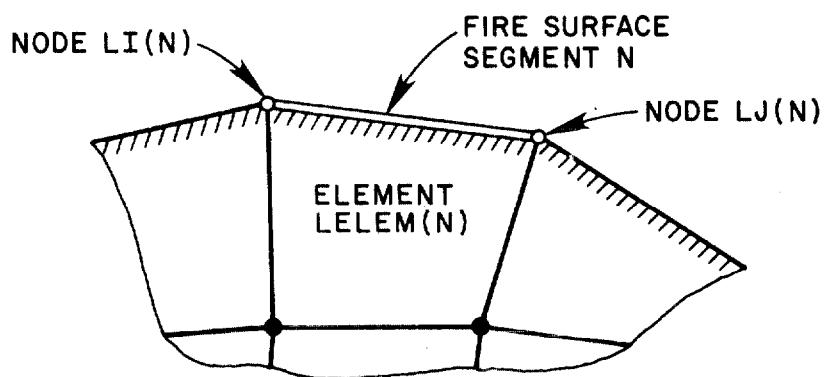
note	field	variable	entry
(1)	FIRE	-	Enter the word "FIRE"
(2)	N1	NFBC1D	Number of one-dimensional surface nodes exposed to fire
(2)	N2	NFBC2D	Number of two-dimensional surface segments exposed to fire
(2)	N3	NFBC3D	Number of three-dimensional surface areas exposed to fire
(3)	N4	NBCMAT	Number of different surface material types

NOTES/

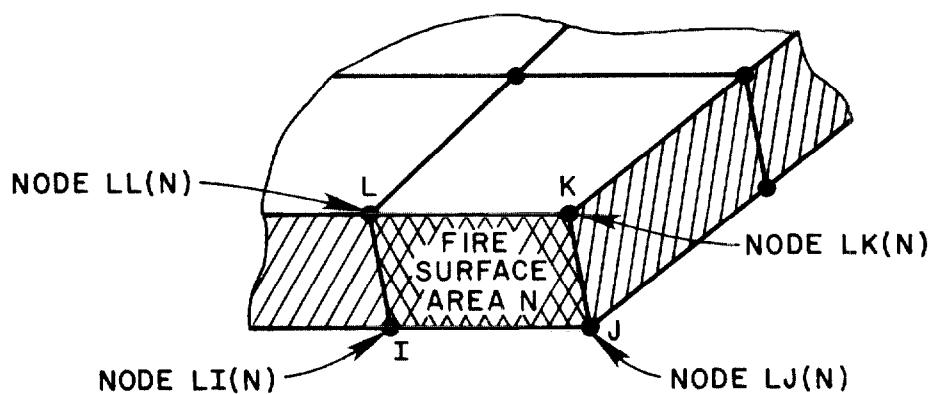
- (1) This is an alphanumeric control card with key word (FIRE) and control parameters (N1, N2, N3, and N4), as shown in Fig. A.1. If fire boundary conditions are not to be used, insert as control card "FIRE, 0, 0, 0, 0" and go on to Data Block VI.
- (2) The fire boundary condition is idealized through the identification of surface segments and their associated thermal properties. For one-dimensional elements, enter (N1), the total number of boundary nodes exposed to fire (see Fig. A.4(a)). For two-dimensional elements, enter (N2), the total number of element edges (surface segments) exposed to fire (see Fig. A.4(b)). For three-dimensional elements, enter (N3), the total number of element surfaces exposed to fire (see Fig. A.4(c)).
- (3) Enter the number of different materials used to define fire surface properties.



(a) TYPICAL FIRE B.C. SURFACE NODE  
FOR 1-D ELEMENTS



(b) TYPICAL FIRE B.C. SURFACE SEGMENT  
FOR 2-D ELEMENTS



(c) TYPICAL FIRE B.C. SURFACE AREA  
FOR 3-D ELEMENTS

FIGURE A.4 DESCRIPTION OF FIRE SURFACE

## B. Linear/Nonlinear Model Card (Alphanumeric)

note      columns      entry

- (1) 1-9 Enter the word "LINEAR" or the word "NONLINEAR" - one or the other.

**NOTE /**

- (1) Either a linear or a nonlinear model may be used for the fire boundary condition. Specify the type on this card.

### C. Material Description

## 1. Linear Material Data

Omit if control card above is "NONLINEAR". Otherwise, input the following set of cards for each material type.

a. Control Card (15)

note    columns    variable    entry

- (1) 1-5 K Number of points used to define linearized  
heat transfer function  
E0, 0, constant function

## **NOTE /**

- (1) Linear surface material properties are input in the same way as conductivity, heat capacity and density in Data Block IV (page A-10)

b. Data Cards (8E10.0)

(same as previous material function input, IV.B.2  
page A-11, 4 pairs per card)

## 2. Nonlinear Material Data

Omit these cards if control card above is "LINEAR":

a. Constant Data Card (2E10-0)

note      columns      variable      entry

1-10 SB Stephan - Boltzman constant

11-20 TSHIFT Shift for absolute temperature

b. Material Data Card (6E10.0)

Input the following card for each material type:

note	columns	variable	entry
(1)	1-10	A	Convection factor
	11-20	P	Power of convection factor
	21-30	V	View factor for radiation term
	31-40	AB	Absorption of surface
	41-50	EF	Emmissivity of flame
	51-60	ES	Emmissivity of surface

NOTE/

See Page 22 for explanation of nonlinear fire boundary condition.

D. Description of Fire Surface

1. Control Card (Alphanumeric)

SURFACE, N1, N2, N3

note	field	variable	entry
(1)	SURFACE	--	Enter the word "SURFACE"
	N1	NS1	Number of one-dimensional surface nodes to be input below LE. NFBC1D
	N2	NS2	Number of two-dimensional surface segments to be input below LE. NFBC2D
	N3	NS3	Number of three-dimensional areas to be input below LE. NFBC3D

NOTE/

- (1) This is an alphanumeric control card with key word (SURFACE) and control parameters (N1, N2, and N3), as shown in Fig. A.1. Enter the number of each type of fire B.C. surface elements that will be input on the cards that follow. The number of each type of element must be less than the maximums defined on Card V.A.

2. Description of One-Dimensional Fire Surface Nodes (16I5)

Omit this card if (N1) on control card above is zero:

note	columns	variable	entry
(1)	1-5	LI(1)	Node number of first boundary surface node
(2)	6-10	LMAT(1)	Material type for this surface node LE. NBCMAT
(3)	11-15	LFIRE(1)	Fire number for this surface node
(4)	16-20	LELEM(1)	Element number of one-dimensional iso-parametric bar element adjacent to this surface node
	21-25	LI(2)	Node number of second boundary surface node.
.	.	.	.
.	.	.	.
.	.	.	.
.	.	.	.
.	.	.	Continue until all (NS1) nodes are input, four nodes per card.

NOTES/

- (1) See Fig. A.4(a)
- (2) Specify which of the material descriptions input on Cards V.C above applies to this boundary node.
- (3) A surface element can be subjected to one of four fires to be input later in the Time-Fire History Data Block.
- (4) The area of the surface exposed to fire is taken to be the cross-sectional area of the one-dimensional bar element connected to that surface node, as shown in Fig. A.4(a). One-dimensional fire surface nodes can only be used in conjunction with one-dimensional bar elements.

3. Description of Two-Dimensional Fire Surface Segments (15I5)

Omit this card if (N2) on control card above is zero.

note	columns	variable	entry
(1)	1-5	LI(1)	Node number of segment end I for first surface segment
(1)	6-10	LJ(1)	Node number of segment end J for first surface segment
(2)	11-15	LMAT(1)	Material type for this surface segment LE. NBCMAT
(3)	16-20	LFIRE(1)	Fire number for this surface segment
(4)	20-25	LELEM(1)	Element number of two-dimensional quadrilateral or triangular element adjacent to this surface segment
	26-30	LI(2)	Node of number of segment end I for second surface segment
.	.	.	.
.	.	.	.
.	.	.	.
.	.	.	Continue until all (NS2) segments are input, three segments per card

NOTES/

- (1) Surface segment runs from I to J, as shown in Fig. A.4(b)
- (2) Specify which of the material descriptions input on Cards V.C above applies to this surface segment.
- (3) A surface segment can be subjected to one of four fires to be input later in the Time-Fire History Data Block
- (4) The edge of one of the quadrilateral elements lies between nodes I and J. Enter the number of that element so that its thickness may be used in determining the area of the surface exposed to fire. Two-dimensional fire surface segments can only be used in conjunction with two-dimensional isoparametric elements.

4. Description of Three-Dimensional Fire Surface Areas (12I5)

Omit this card if (N3) on control card above is zero.

note	columns	variable	entry
(1)	1-5	LI(1)	Node number of corner I for first surface element
(1)	6-10	LJ(1)	Node number of corner J for first surface element
(1)	11-15	LK(1)	Node number of corner K for first surface element
(1)	16-20	LL(1)	Node number of corner L for first surface element
(2)	21-25	LMAT(1)	Material type for this surface element LE. NBCMAT
(3)	26-30	LFIRE(1)	Fire number for this surface element
	31-35	LI(2)	Node number of corner I for second surface element
.	.	.	.
.	.	.	.
.	.	.	.
.	.	.	Continue until all (NS3) surface elements are input, two elements per card

#### NOTES/

- (1) A surface area is defined by the four nodes, I, J, K, and L, as shown in Fig. A.4(c). These must be input in counter-clockwise order and lie in the same plane.
- (2) Specify which of the material descriptions input on Cards V.C above applies to this surface area.
- (3) A surface area can be subjected to one of four fires to be input later in the Time-Fire History Data Block

## VI. INTERNAL HEAT GENERATION DATA

### A. Control Card (Alphanumeric)

EXOTHERMIC, N1, N2, N3, N4

note	field	variable	entry
(1)	EXOTHERMIC	--	Enter the word "EXOTHERMIC"
(2)	N1	NINT1D	Number of one-dimensional elements with internal heat generation
(2)	N2	NINT2D	Number of two-dimensional elements with internal heat generation
(2)	N3	NINT3D	Number of three-dimensional elements with internal heat generation
(3)	N4	NQINT	Number of different heating functions

#### NOTES/

- (1) This is an alphanumeric control card with key word (EXOTHERMIC) and control parameters (N1, N2, N3, and N4), as shown in Fig. A.1. If internal heat generation is not being considered, insert as control card "EXOTHERMIC, 0, 0, 0, 0" and go to Data Block VII.
- (2) Enter how many of each type of element undergoes internal heating from processes such as hydration, combustion, etc.
- (3) Enter the number of heat-rate vs. time functions to be considered.

#### B. Internal Heat Generation Functions

Input the following set of cards for each heating function:

##### 1. Control Card (2I5)

note	columns	variable	entry
(1)	1-5	MK	Number of points used to define heating function GE.2
(2)	6-10	MT	Type of heating curve input EQ.0, heat flow per unit volume EQ.1, heat flow per unit mass

#### NOTES/

- (1) Internal heating rate is input as a tabular function of time. At least two points are needed to define each function and linear interpolation is used between points.

- (2) Heating rate may be defined as heat per unit time per unit volume or as heat per unit time per unit mass. Units must be compatible with other input data.

## 2. Heat Function (3E10.0)

note	columns	variable	entry
(1)	1-10	X(1)	Time of point 1
	11-20	Y(1)	Value of heat rate at point 1
	21-30	X(2)	Time of point 2
	.	.	.
	.	.	.
	.	.	.
	.	Y(MK)	Continue until all (MK) points are input

NOTE/

- (1) Input the table that defines internal heating rate in the same way as conductivity, heat capacity and density in IV.B.2, 4 points per card.

## C. Data for One-Dimensional Elements (16I5)

Omit this card if (N1) on control card is zero.

note	columns	variable	entry
	1-5	IEL(1)	Element number of first one-dimensional bar element undergoing internal heating.
	6-10	IMAT(1)	Heat function number for first element
	11-15	IEL(2)	Element number of second one-dimensional element undergoing internal heating
	16-20	IMAT(2)	Heat function number for second element
	.	.	.
	.	.	.
	.	.	.
	.	IMAT(NINT1D)	Continue until all (NINT1D) elements are input, 8 elements per card.

D. Data for Two-Dimensional Elements (16I5)

Omit this card if (N2) on Control Card is zero.

Input element number and heat function number for all (NINT2D) two-dimensional elements, 8 per card, in the same way as above.

E. Data for Three-Dimensional Elements (16I5)

Omit this card if (N3) on Control Card is zero.

Input element number and heat function number for all (NINT3D) three-dimensional elements, 8 per card, in the same way as above.

VII. CONVERGENCE CRITERIA

A. Control Card (Alphanumeric)

CONVERGENCE

note field variable entry

(1) CONVERGENCE -- Enter the word "CONVERGENCE"

NOTE/

- (1) This is an alphanumeric control card with key word (CONVERGENCE) and no control parameters, as in Fig. A.1.

B. Convergence Criteria Card (I5,2F10.0,I5,2F10.0)

note columns variable entry

(1) 1-5 NCONV Maximum number of iterations permitted for fire B.C. solution

(1) 6-15 CONV Permissible relative error for fire B.C. iteration

(2) 16-25 BETA Overconvergence factor for fire B.C. iteration

(3) 26-30 NCONU Maximum number of iterations permitted for system solution in each time step EQ.0, linear solution - no iteration

note	columns	variable	entry
(3)	31-40	CONU	Permissible relative error for system iteration
(2)	41-50	ALPHA	Overconvergence factor for system iteration

#### NOTES/

- (1) The fire boundary conditions are temperature-dependent and require an iterative solution process, controlled by variables (NCONV) and (CONV) as defined on page 27. If there are no fire B.C.s or if no iteration is desired, leave columns 1-25 blank.
- (2) See page 27 . To ignore overconvergence factors, leave blank.
- (3) When material properties (heat conductivity, etc.) are temperature-dependent, an iterative process is needed to solve the nonlinear heat flow problem. However, usually it suffices to forgo iteration when material properties change little during a time step, thereby reducing greatly computation costs. In this case (or for linear heat flow problems) leave columns 26-50 blank.

## VIII. INITIAL CONDITIONS

### A. Initial Time Step Control Card (A4,I6,2F10.0,2X,A3)

note	columns	variable	entry
	1-4	IA	Enter the word "STEP"
(1)	5-10	MDT	Initial number in sequencing of time steps
(1)	11-20	TIME	Initial time (i.e., base time)
(2)	21-30	TEMP	Uniform initial temperature (If initial temperature is nonuniform, leave cols. 21-30 blank and input nodal temperatures on the data card below)
	31-32	-	blank
(3)	33-34	JP	3-symbol alphanumeric code that will appear in columns 74-76 of punched output.

NOTES/

- (1) The usual starting point for time step sequencing is "0" and the usual initial time is "0.0". However, nonzero values may be specified if desired.
- (2) All nodal temperatures are initialized to this value unless its value is "0.0", in which case all initial nodal temperatures must be input on Card XIII.B below.
- (3) If these columns are left blank the code "NODE" will appear on punched nodal temperatures and the code "ELEM" on punched element temperatures.

B. Initial Temperature Distribution Data (7(4X,F6.1))

Omit this card if (TEMP) on control card above is nonzero.

note	columns	variable	entry
(1)	1-4	--	blank
	5-10	T(1)	Initial temperature at node 1
	11-14	--	blank
	15-20	T(2)	Initial temperature at node 2
	.	.	.
	.	.	.
	.	.	.
	.	T(NUMNP)	Continue until all nodal points are input.

NOTE/

- (1) Enter the initial nodal temperatures, seven (7) nodes per data card, using as many cards as necessary. This format is compatible with punched output from the program; hence, this option can be used to restart a previous analysis that must be continued for a longer time period. For a restarted analysis change the initial time (TIME) on the control card to the time at which the previous analysis ended.

## IX. TIME-FIRE HISTORY

For each time step input the following block of data:

### A. Time Step Control Card (A4,I6,F10.0,I5,4F10.0,4I3)

note	columns	variable	entry
	1-4	IA	Enter the word "STEP"
(1)	5-10	NDT	Time step number
(2)	11-20	DT	Time step interval
(3)	21-25	ITOF	Number of non-zero flow or temperature boundary conditions
(4)	26-35	TFIRE(1)	Temperature of fire 1
	36-45	TFIRE(2)	Temperature of fire 2
	46-55	TFIRE(3)	Temperature of fire 3
	56-65	TFIRE(4)	Temperature of fire 4
	66-68	I1	Printed output desired for this time step EQ.0, no output EQ.1, nodal temperatures EQ.2, element temperatures EQ.3, both nodal and element temperatures
	69-71	I2	Punched output desired for this time step EQ.0, no punched output EQ.1, nodal temperatures EQ.2, element temperatures EQ.3, both nodal and element temperatures
(5)	72-74	I6	Intermediate printed output for debugging purposes EQ.0, no printout EQ.1, debugging printout
(6)	75-77	I7	Fire boundary condition flag EQ.0, continue with same fire B.C. surfaces previously defined EQ.1, input new fire B.C. surface in Data Block IX.C below

NOTES/

- (1) Time step cards must be input in ascending sequence with no time steps omitted.
- (2) Input the length of the time step in the same units used to define other input quantities. To end a data case input a negative time step size in these columns and then proceed to the next data case (Heading Card).
- (3) Enter the total number of nonzero fixed heat flow or fixed temperature boundary condition nodes. The value of flow or temperature for each of these nodes is entered on the next data card.
- (4) Up to four fire histories can be defined for use in conjunction with the fire boundary conditions input in Data Block V. Enter the temperature of each fire at the end of the time step.
- (5) If convergence difficulties are encountered in any particular analysis, this option may be called to print nodal temperatures during each iteration. This option can also be used to determine an efficient overconvergence factor.
- (6) A fire boundary surface is defined in Data Block V and can be used for all time steps in the fire history. However, it is possible to change the fire surface during any time step by setting (I7) equal to "1" and inputting the new surface below. The new surface replaces the one previously input and is used for all subsequent time steps.

B. Nonzero Boundary Condition Data (5(I5,F10.0))

Omit this card if (ITOF) on Control Card above is zero.

note	columns	variable	entry
(1)	1-5	J(1)	Node number
	6-15	FT(1)	Specified temperature or flow at that node
	16-20	J(2)	Node number

note	columns	variable	entry
21-30	FT(2)		Specified temperature or flow at that node
.	.		.
.	.		.
.	.		.
.	FT(ITOF)		Continue until all (ITOF) boundary nodes are input.

**NOTE/**

- (1) Input the global node number and the specified temperature boundary condition or specified heat flow boundary condition for each of the (ITOF) boundary nodes. Enter five (5) nodes per data card and use as many cards as necessary. This data must be input for each time step - even if the boundary conditions do not change from time step to time step. Fixed flow nodes are differentiated from fixed temperature nodes using the information input previously in Data Block II.C.

**C. Description of New Fire Surface**

Omit these cards if (I7) on the control card above is zero.

**1. Control Card (Alphanumeric)**

SURFACE, N1, N2, N3

(Same format as Card V.D.1, page A-16)

**2. Description of One-Dimensional Fire Surface Nodes (16I5)**

Omit this card if (N1) is zero

(Same format as Card V.D.2, page A-17)

**3. Description of Two-Dimensional Fire Surface Segments (15I5)**

Omit this card if (N2) is zero

(Same format as Card V.D.3, page A-18)

4. Description of Three-Dimensional Fire Surface Areas (12I5)

Omit this card if (N3) is zero

(Same format as Card V.D.4, page A-19)

APPENDIX B

SAMPLE PROBLEMS

### SAMPLE PROBLEM 1 - COLUMN CROSS-SECTION WITH UNIFORM FIRE EXPOSURE

The first sample problem is a two-dimensional thermal analysis of a square column (Fig. B.1) from the basement level of a typical reinforced concrete frame structure. It is necessary only to consider a quadrant of the column in the thermal analysis because of its cross-sectional symmetry and the uniformity of the intended fire exposure. The column is thus idealized by the finite element grid in Fig. B.2, where planes of symmetry are modeled as insulated ( $Q_p = 0$ ) surfaces. The thermal properties used in both sample problems are those given by Bizri [3]. The choice of the grid was based on the results of previous studies. A fine grid is employed in the vicinity of the fire boundary since a steep temperature gradient is to be expected there. A coarser grid is used in the center of the column since the gradient there is expected to be lower. The steel bars have been idealized as rectangles based on an area equivalence ( $S = 0.89D$ ). The flexibility of steel placement is apparent from Fig. B.2, allowing convenient modeling of any cross-section regardless of steel arrangement. The column is analyzed for two time-temperature curves (pseudo-fires):

- 1) A long duration moderate intensity fire (ASTM E-119), and
- 2) A short duration high intensity fire (SDHI).

These two pseudo-fire curves are shown in Figs. B.3 and B.4, respectively.

The fire boundary condition is simulated with the nonlinear model using a concrete emissivity of 0.9 and a flame emissivity of 0.3. Input data for this problem is given in Table B.1 (for the first few time steps), and typical output is given in Table B.2. Figs. B.3 and B.4 are graphs showing the results of the analysis as the variation of temperature with time for certain elements, the exterior corner element (78) and the interior corner element (1), two steel elements (60 and 8), and a side element (10). The ASTM fire was also considered using a flame emissivity of 0.9 (Fig. B.5). The importance of flame emissivity can be seen by comparing Figs. B.3 and B.5. The central processing time required for each case is 49 seconds using the University of California CDC 6400 computer.

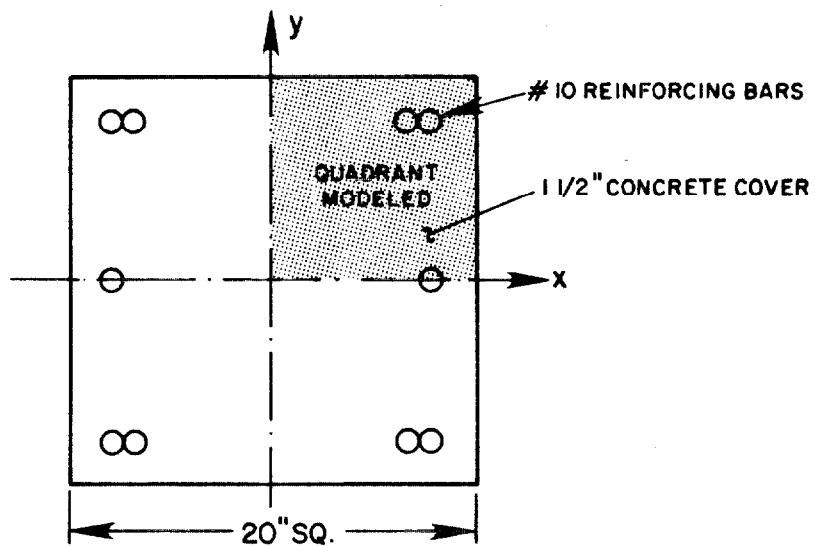


FIGURE B.1 COLUMN CROSS-SECTION

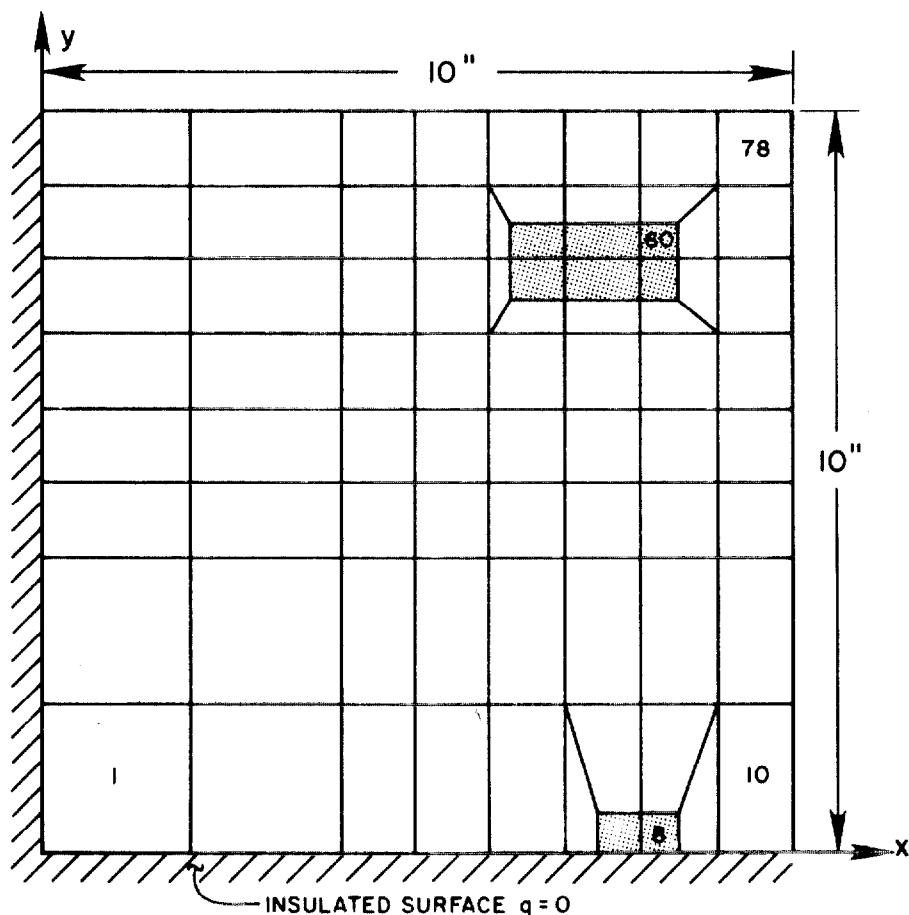


FIGURE B.2 FINITE ELEMENT MESH FOR COLUMN QUADRANT  
(TWO-DIMENSIONAL QUADRILATERAL ELEMENTS)

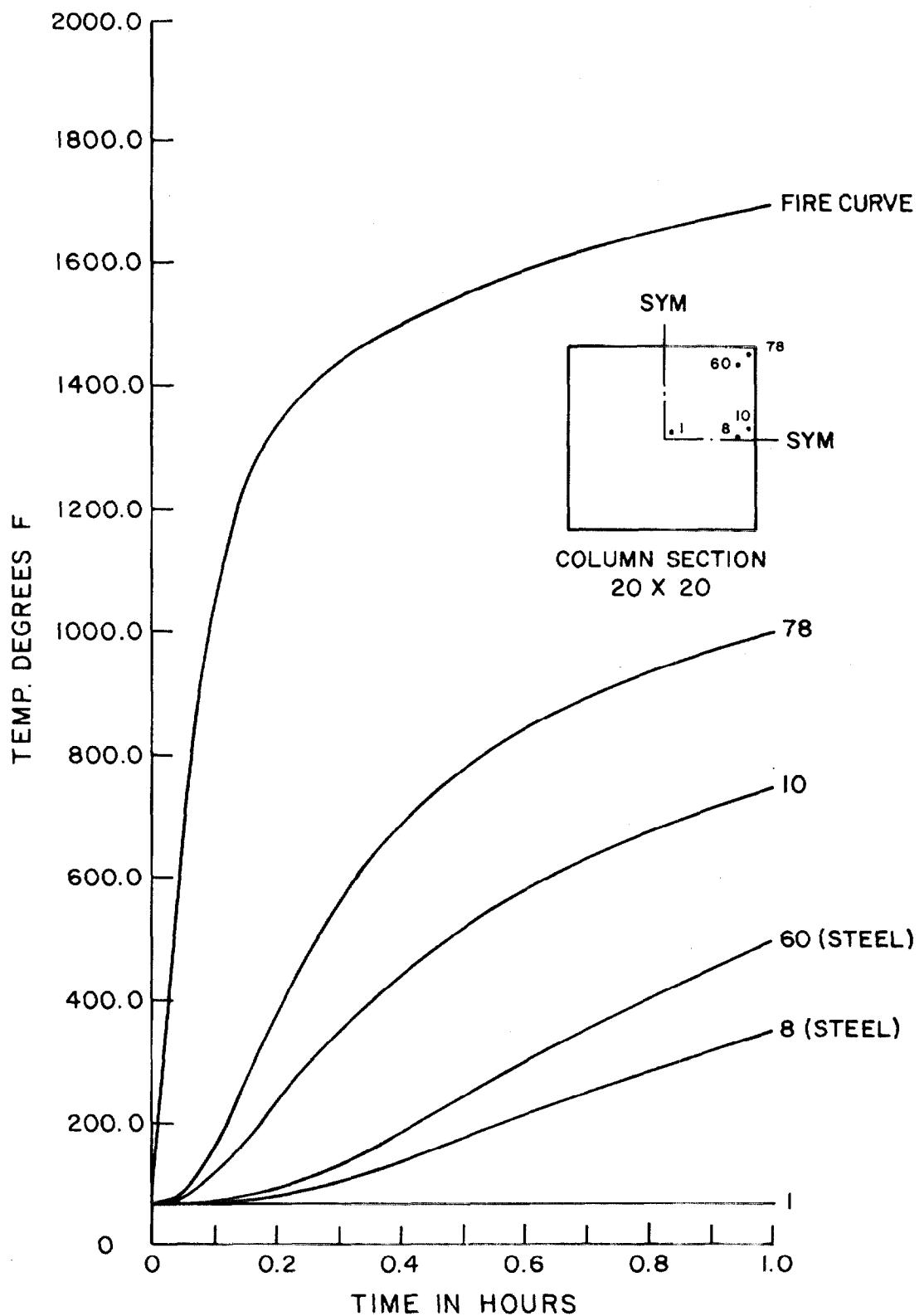


FIGURE B.3 TEMPERATURE HISTORIES FOR SELECT ELEMENTS  
IN COLUMN UNIFORMLY EXPOSED TO ASTM FIRE  
( $\epsilon_f = 0.3$ )

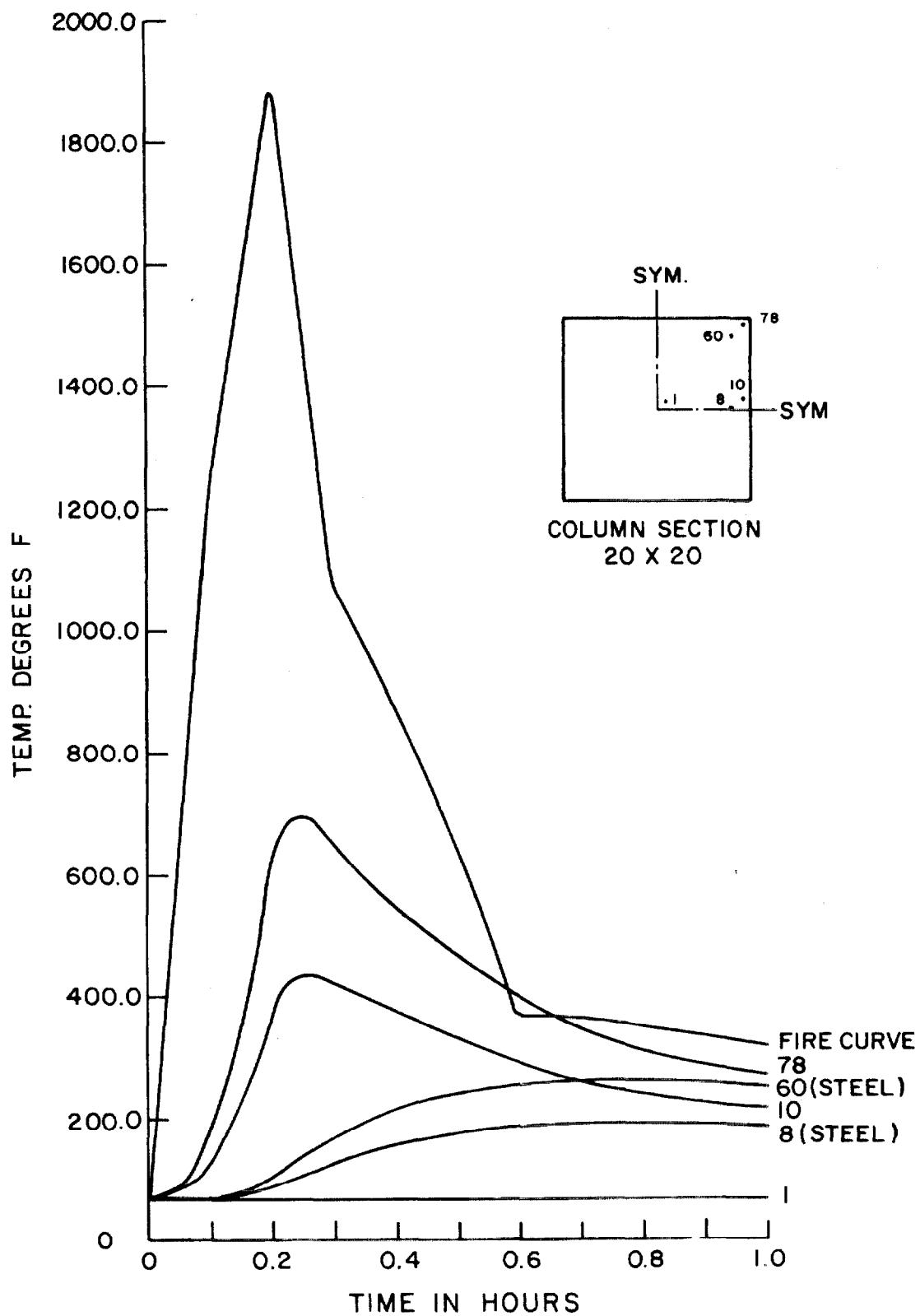


FIGURE B.4 TEMPERATURE HISTORIES FOR SELECT ELEMENTS  
IN COLUMN UNIFORMLY EXPOSED TO SDHI FIRE  
( $\epsilon_f = 0.3$ )

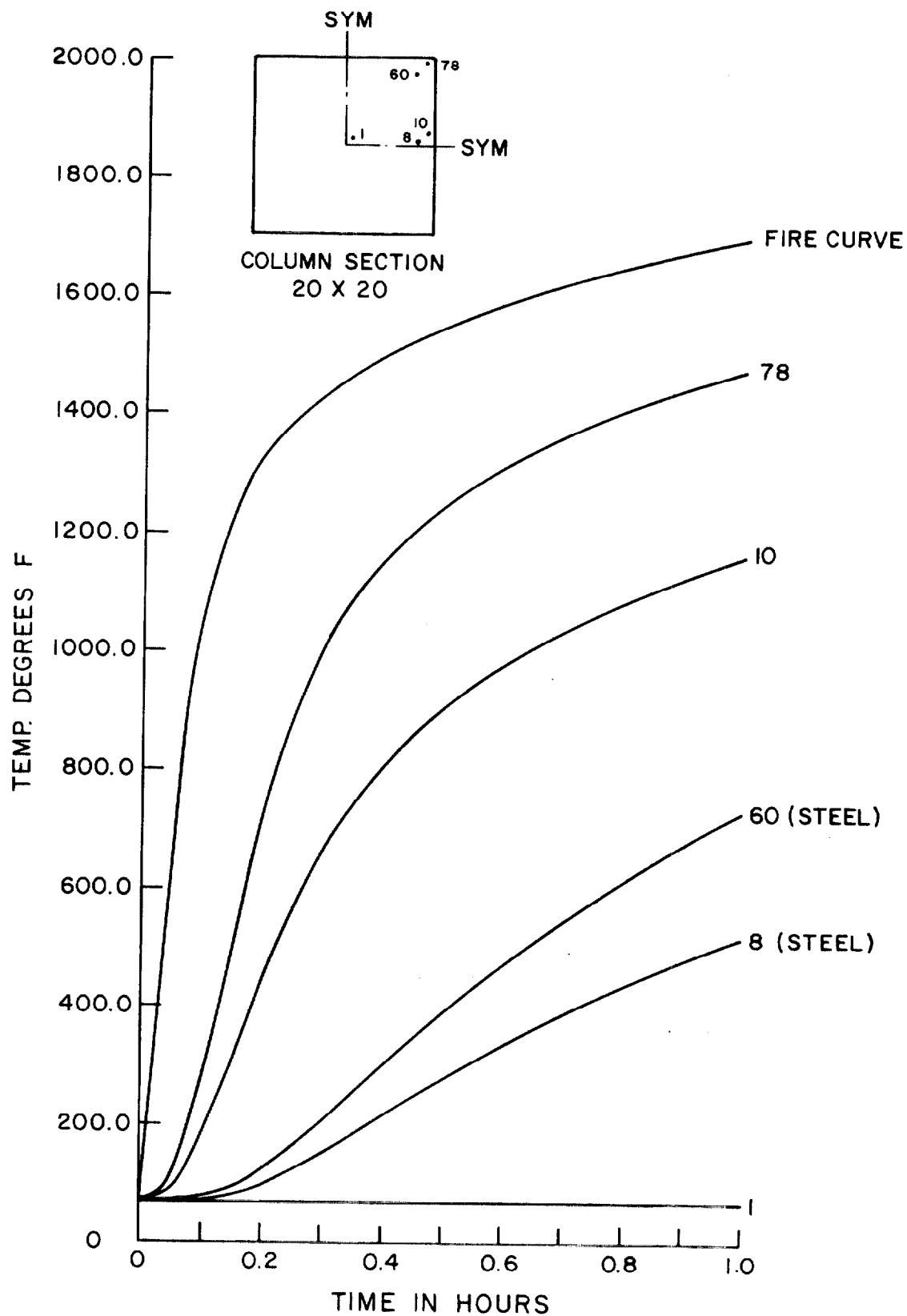


FIGURE B.5 TEMPERATURE HISTORIES FOR SELECT ELEMENTS  
IN COLUMN UNIFORMLY EXPOSED TO ASTM FIRE  
( $\epsilon_f = 0.9$ )

	0	20	30	40	50	60
<b>*** INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE ***</b>						
NODES, 96, 0						
1	0.		0.			
3	.333333		0.			
6	.583333		0.			
7	.614154		0.			
8	.666667		0.			
9	.708333		0.			
10	.750000		0.			
11	.833333		0.			
12	.614154	.047086				
13	.666667	.047086				
14	.708333	.047086				
15	0.	.166667				
17	.333333	.166667				
23	.833333	.166667				
24	0.	.333333				
26	.333333	.333333				
32	.833333	.333333				
33	0.	.416667				
35	.333333	.416667				
41	.833333	.416667				
42	0.	.500000				
44	.333333	.500000				
50	.833333	.500000				
51	0.	.583333				
53	.333333	.583333				
55	.500000	.583333				
56	.520834	.614154				
57	.583333	.583333				
58	.583333	.614154				
59	.666667	.583333				
60	.666667	.614154				
61	.708333	.614154				
62	.750000	.583333				
63	.833333	.583333				
64	0.	.666667				
66	.333333	.666667				
68	.500000	.666667				
69	.520834	.666667				
70	.583333	.666667				
71	.666667	.666667				
72	.708333	.666667				
73	.750000	.666667				
74	.833333	.666667				
75	0.	.750000				
77	.333333	.750000				
79	.500000	.750000				
80	.520834	.708333				
81	.583333	.750000				
82	.583333	.708333				
83	.666667	.750000				
84	.666667	.708333				
85	.708333	.708333				
86	.750000	.750000				

TABLE B.1 INPUT FOR SAMPLE PROBLEM ONE

	10	20	30	40	50	60	70	80
87	.833333	.750000						
88	0.	.833333						
90	.333333	.833333						
96	.833333	.833333						
<b>ELEMENTS, 0,78,0</b>								
1	1	2	16	15	1	1.0		
6	6	7	12	20	1	1.0		
7	7	8	13	12	2	1.0		
9	9	10	22	14	1	1.0		
10	10	11	23	22	1	1.0		
11	12	13	21	20	1	1.0		
12	13	14	22	21	1	1.0		
13	15	16	25	24	1	1.0		
21	24	25	34	33	1	1.0		
29	33	34	43	42	1	1.0		
37	42	43	52	51	1	1.0		
41	46	47	57	55	1	1.0		
42	47	48	59	57	1	1.0		
43	48	49	62	59	1	1.0		
44	49	50	63	62	1	1.0		
45	51	52	65	64	1	1.0		
50	55	57	58	56	1	1.0		
51	57	59	60	58	1	1.0		
52	59	62	61	60	1	1.0		
53	61	62	73	72	1	1.0		
54	62	63	74	73	1	1.0		
55	56	58	70	69	2	1.0		
56	58	60	71	70	2	1.0		
57	60	61	72	71	2	1.0		
58	69	70	82	80	2	1.0		
59	70	71	84	82	2	1.0		
60	71	72	85	84	2	1.0		
61	64	65	76	75	1	1.0		
66	80	82	81	79	1	1.0		
67	82	84	83	81	1	1.0		
68	84	85	86	83	1	1.0		
69	72	73	86	85	1	1.0		
71	75	76	89	88	1	1.0		
75	79	81	93	92	1	1.0		
76	81	83	94	93	1	1.0		
77	83	86	95	94	1	1.0		
78	86	87	96	95	1	1.0		
<b>MATERIALS, 2</b>								
4	0	0						
	0.0	1.01	390.0	1.01	1650.0	0.506	3000.0	0.506
	0.272							
	150.0							
3	4	0						
	0.0	30.00	1100.0	19.90	3000.0	19.90		
	0.0	0.107	750.0	0.144	1100.0	0.172	3000.0	0.172
	480.0							
<b>FIRE, 0,16,0,1</b>								
<b>NONLINEAR</b>								
	1.7E-9	460.0						
	.27	1.25	1.0	0.9	0.3	0.9		
<b>SURFACE, 0,16,0</b>								
	10	20	30	40	50	60	70	80

	10	20	30	40	50	60	70	80
11	23	1	1	10	23	32	1	1
41	50	1	1	36	50	63	1	1
74	87	1	1	70	87	96	1	1
95	94	1	1	77	94	93	1	1
92	91	1	1	74	91	90	1	1
89	88	1	1	71				
EXOTHERMIC, 0,0,0,0								
CONVERGENCE								
15	.005		- .25					
STEP 0		0.0	68.0					
STEP 1		.025	347.600				3	2
STEP 2		.025	627.200				3	2
STEP 3		.025	906.800				3	2
STEP 4		.025	1060.000				3	2
STEP 5		.025	1150.000				3	2

\*\*\*\*\*

FFFFF	I	RRRRR	EEEE	SSSSS		TTTTT	33333
F	I	R R	E	S		T	3
F	I	R R	E	S		T	3
F	I	R R	E	S		T	3
FFF	I	RRRRR	EEE	SSSSS	=====	T	3333
F	I	RR	E	S		T	3
F	I	R R	E	S		T	3
F	I	R R	E	S		T	3
F	I	R R	EEEE	SSSSS		T	33333

A THERMAL ANALYZER FOR THREE-DIMENSIONAL SYSTEMS,  
WITH TEMPERATURE-DEPENDENT THERMAL PROPERTIES,  
SUBJECTED TO A FIRE ENVIRONMENT

\*\*\*\*\*

- - - TITLE OF RUN - - -

\*\*\* INTERIOR BASEMENT COLUMN - BLOCK STUDY - ASTM FIRE \*\*\*

\*\*\*\*\*

TABLE B.2 OUTPUT FOR SAMPLE PROBLEM ONE

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

GEOMETRIC DESCRIPTION OF SYSTEM TO BE ANALYZED

\*\*\*\*\*

\* \* \* THERE ARE 96 NODAL POINTS \* \* \*

NODAL POINT	X	COORDINATES	Z	BOUNDARY CONDITION
	X	Y	Z	
1	0.	0.	-0.	FLCW
2	.1667	0.	0.	FLOW
3	.3333	0.	-0.	FLOW
4	.4167	0.	0.	FLOW
5	.5000	0.	0.	FLOW
6	.5833	0.	-0.	FLCW
7	.6142	0.	-0.	FLOW
8	.6667	0.	-0.	FLOW
9	.7083	0.	-0.	FLCW
10	.7500	0.	-0.	FLOW
11	.8333	0.	-0.	FLCW
12	.6142	.0471	-0.	FLOW
13	.6667	.0471	-0.	FLOW
14	.7083	.0471	-0.	FLOW
15	0.	.1667	-0.	FLOW
16	.1667	.1667	0.	FLCW
17	.3333	.1667	-0.	FLOW
18	.4167	.1667	0.	FLOW
19	.5000	.1667	0.	FLCW
20	.5833	.1667	0.	FLOW
21	.6667	.1667	0.	FLCW
22	.7500	.1667	0.	FLOW
23	.8333	.1667	-0.	FLOW
24	0.	.3333	-0.	FLCW
25	.1667	.3333	0.	FLOW
26	.3333	.3333	-0.	FLCW
27	.4167	.3333	0.	FLOW
28	.5000	.3333	0.	FLOW
29	.5833	.3333	0.	FLCW
30	.6667	.3333	0.	FLOW
31	.7500	.3333	0.	FLCW
32	.8333	.3333	-0.	FLOW
33	0.	.4167	-0.	FLOW
34	.1667	.4167	0.	FLCW
35	.3333	.4167	-0.	FLOW
36	.4167	.4167	0.	FLCW
37	.5000	.4167	0.	FLCW
38	.5833	.4167	0.	FLOW

39	.6667	.4167	0.	FLOW
40	.7500	.4167	0.	FLOW
41	.8333	.4167	-0.	FLOW
42	0.	.5000	-0.	FLOW
43	.1667	.5000	0.	FLCW
44	.3333	.5000	-0.	FLOW
45	.4167	.5000	0.	FLOW
46	.5000	.5000	0.	FLCW
47	.5833	.5000	0.	FLOW
48	.6667	.5000	0.	FLCW
49	.7500	.5000	0.	FLOW
50	.8333	.5000	-0.	FLOW
51	0.	.5833	-0.	FLOW
52	.1667	.5833	0.	FLOW
53	.3333	.5833	-0.	FLCW
54	.4167	.5833	0.	FLOW
55	.5000	.5833	-0.	FLOW
56	.5208	.6142	-0.	FLOW
57	.5833	.5833	-0.	FLOW
58	.5833	.6142	-0.	FLOW
59	.6667	.5833	-0.	FLOW
60	.6667	.6142	-0.	FLOW
61	.7083	.6142	-0.	FLOW
62	.7500	.5833	-0.	FLOW
63	.8333	.5833	-0.	FLOW
64	0.	.6667	-0.	FLOW
65	.1667	.6667	0.	FLOW
66	.3333	.6667	-0.	FLOW
67	.4167	.6667	0.	FLOW
68	.5000	.6667	-0.	FLCW
69	.5208	.6667	-0.	FLOW
70	.5833	.6667	-0.	FLOW
71	.6667	.6667	-0.	FLOW
72	.7083	.6667	-0.	FLOW
73	.7500	.6667	-0.	FLCW
74	.8333	.6667	-0.	FLOW
75	0.	.7500	-0.	FLOW
76	.1667	.7500	0.	FLOW
77	.3333	.7500	-0.	FLOW
78	.4167	.7500	0.	FLCW
79	.5000	.7500	-0.	FLCW
80	.5208	.7083	-0.	FLOW
81	.5833	.7500	-0.	FLCW
82	.5833	.7083	-0.	FLOW
83	.6667	.7500	-0.	FLCW
84	.6667	.7083	-0.	FLCW
85	.7083	.7083	-0.	FLOW
86	.7500	.7500	-0.	FLCW
87	.8333	.7500	-0.	FLOW
88	0.	.8333	-0.	FLCW
89	.1667	.8333	0.	FLCW
90	.3333	.8333	-0.	FLOW
91	.4167	.8333	0.	FLCW
92	.5000	.8333	0.	FLOW
93	.5833	.8333	0.	FLOW
94	.6667	.8333	0.	FLCW

95	.7500	.8333	0.	FLCW
96	.8333	.8333	-C.	FLCW

\* \* \* \* THERE ARE 78 2-D ELEMENTS \* \* \*

ELMT	I	J	K	L	MAT	THICKNESS
1	1	2	16	15	1	1.00000
2	2	3	17	16	1	1.00000
3	3	4	18	17	1	1.00000
4	4	5	19	18	1	1.00000
5	5	6	20	19	1	1.00000
6	6	7	12	20	1	1.00000
7	7	8	13	12	2	1.00000
8	8	9	14	13	2	1.00000
9	9	10	22	14	1	1.00000
10	10	11	23	22	1	1.00000
11	12	13	21	20	1	1.00000
12	13	14	22	21	1	1.00000
13	15	16	25	24	1	1.00000
14	16	17	26	25	1	1.00000
15	17	18	27	26	1	1.00000
16	18	19	28	27	1	1.00000
17	19	20	29	28	1	1.00000
18	20	21	30	29	1	1.00000
19	21	22	31	30	1	1.00000
20	22	23	32	31	1	1.00000
21	24	25	34	33	1	1.00000
22	25	26	35	34	1	1.00000
23	26	27	36	35	1	1.00000
24	27	28	37	36	1	1.00000
25	28	29	38	37	1	1.00000
26	29	30	39	38	1	1.00000
27	30	31	40	39	1	1.00000
28	31	32	41	40	1	1.00000
29	33	34	43	42	1	1.00000
30	34	35	44	43	1	1.00000
31	35	36	45	44	1	1.00000
32	36	37	46	45	1	1.00000
33	37	38	47	46	1	1.00000
34	38	39	48	47	1	1.00000
35	39	40	49	48	1	1.00000
36	40	41	50	49	1	1.00000
37	42	43	52	51	1	1.00000
38	43	44	53	52	1	1.00000
39	44	45	54	53	1	1.00000
40	45	46	55	54	1	1.00000
41	46	47	57	55	1	1.00000
42	47	48	59	57	1	1.00000
43	48	49	62	59	1	1.00000
44	49	50	63	62	1	1.00000
45	51	52	65	64	1	1.00000
46	52	53	66	65	1	1.00000

47	53	54	67	66	1	1.00000
48	54	55	68	67	1	1.00000
49	55	56	69	68	1	1.00000
50	55	57	58	56	1	1.00000
51	57	59	60	58	1	1.00000
52	59	62	61	60	1	1.00000
53	61	62	73	72	1	1.00000
54	62	63	74	73	1	1.00000
55	56	58	70	69	2	1.00000
56	58	60	71	70	2	1.00000
57	60	61	72	71	2	1.00000
58	69	70	82	80	2	1.00000
59	70	71	84	82	2	1.00000
60	71	72	85	84	2	1.00000
61	64	65	76	75	1	1.00000
62	65	66	77	76	1	1.00000
63	66	67	78	77	1	1.00000
64	67	68	79	78	1	1.00000
65	68	69	80	79	1	1.00000
66	80	82	81	79	1	1.00000
67	82	84	83	81	1	1.00000
68	84	85	86	83	1	1.00000
69	72	73	86	85	1	1.00000
70	73	74	87	86	1	1.00000
71	75	76	89	88	1	1.00000
72	76	77	90	89	1	1.00000
73	77	78	91	90	1	1.00000
74	78	79	92	91	1	1.00000
75	79	81	93	92	1	1.00000
76	81	83	94	93	1	1.00000
77	83	86	95	94	1	1.00000
78	86	87	96	95	1	1.00000

• • • MAXIMUM BANDWIDTH IS 16 • • •

\*\*\*\*\*

FIRES-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

THERMAL PROPERTIES OF SYSTEM TO BE ANALYZED

THERE ARE 2 DIFFERENT MATERIALS

\*\*\*\*\*

• • • MATERIAL NUMBER 1 • • •

• • • CONDUCTIVITY • • •

NODE	TEMPERATURE	VALUE	SLOPE
1	0.	1.010	0.
2	390.0	1.010	-4.00E-03
3	1650.0	.506	0.
4	3000.0	.506	

• • • SPECIFIC HEAT • • •

MATERIAL PARAMETER OF CONSTANT VALUE .272

• • • DENSITY • • •

MATERIAL PARAMETER OF CONSTANT VALUE 150.000

• • • MATERIAL NUMBER 2 • • •

• • • CONDUCTIVITY • • •

NODE	TEMPERATURE	VALUE	SLOPE
1	0.	30.000	-9.18E-02
2	1100.0	19.900	0.
3	3000.0	19.900	

\* \* \* SPECIFIC HEAT \* \* \*

NODE	TEMPERATURE	VALUE	SLOPE
1	0.	.107	
2	750.0	.144	.493E-04
3	1100.0	.172	.800E-04
4	3000.0	.172	0.

\* \* \* DENSITY \* \* \*

MATERIAL PARAMETER OF CONSTANT VALUE 480.000

\*\*\*\*\*

FIRES-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

NON-LINEAR FIRE BOUNDARY CONDITION

\*\*\*\*\*

$$Q = A * (TF - TS)^N + SB * V * (AB * EF * (TF + TSHIFT) + 4 - ES * (TS + TSHIFT) + 4)$$

WHERE

TF - PSUEDO FIRE TEMPERATURE

TS - SURFACE TEMPERATURE

SB - STEFAN BOLTZMANN CONSTANT = .1700E-08

TSHIFT - SHIFT TO ABSOLUTE TEMPERATURE SCALE = 460.0

AND

MAT NUM	CONVECT FACTOR (A)	CONVECT POWER (N)	VIEW FACTOR (V)	ABSCRET (AE)	FIRE EMISSIV (EF)	SURFACE EMISSIV (ES)
1	.270	1.250	1.000	.900	.300	.900

• • • THERE ARE 16 2-D SURFACE ELEMENTS EXPOSED TO FIRE • •

DESCRIPTION OF SURFACE DIRECTLY EXPOSED TO FIRE

FIREBC SURFACE	NODE I	NODE J	MAT TYPE	FIRE TYPE	AREA
1	11	23	1	1	.167
2	23	32	1	1	.167
3	32	41	1	1	.083
4	41	50	1	1	.083
5	50	63	1	1	.083
6	63	74	1	1	.083
7	74	87	1	1	.083
8	87	96	1	1	.083
9	96	95	1	1	.083
10	95	94	1	1	.083
11	94	93	1	1	.083
12	93	92	1	1	.083
13	92	91	1	1	.083
14	91	90	1	1	.083
15	90	89	1	1	.167
16	89	88	1	1	.167

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

INFORMATION RELEVANT TO THE ANALYSIS PROCEDURE

\*\*\*\*\*

\* \* \* \* CONVERGENCE CRITERIA \* \* \* \*

CONVERGENCE CRITERIA FOR BOUNDARY CONDITIONS

PERMISSIBLE ERROR = .00500  
MAXIMUM NUMBER OF ITERATIONS = 15      BETA = -.2500

\* \* \* \* STORAGE REQUIREMENT FOR BLANK COMMON \* \* \* \*

SIZE BLANK COMMON = 3390 (DECIMAL)  
= 0006476 (CCTAL)

\*\*\*\*\*

FIREST3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

INITIAL SEQUENCE NUMBER IS 0 AND INITIAL TIME IS 0.

\*\*\*\*\*

----- NCAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.00	7	68.00	8	68.00
9	68.00	10	68.00	11	68.00	12	68.00
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.00	40	68.00
41	68.00	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.00	48	68.00
49	68.00	50	68.00	51	68.00	52	68.00
53	68.00	54	68.00	55	68.00	56	68.00
57	68.00	58	68.00	59	68.00	60	68.00
61	68.00	62	68.00	63	68.00	64	68.00
65	68.00	66	68.00	67	68.00	68	68.00
69	68.00	70	68.00	71	68.00	72	68.00
73	68.00	74	68.00	75	68.00	76	68.00
77	68.00	78	68.00	79	68.00	80	68.00
81	68.00	82	68.00	83	68.00	84	68.00
85	68.00	86	68.00	87	68.00	88	68.00
89	68.00	90	68.00	91	68.00	92	68.00
93	68.00	94	68.00	95	68.00	96	68.00

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

TIME STEP NUMBER 1 - TIME .025 - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS -0

FIRE BOUNDARY CONDITION

FIRE(1) = 347.600  
FIRE(2) = -0.  
FIRE(3) = -0.  
FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.01	7	68.03	8	68.03
9	68.04	10	68.47	11	72.74	12	68.02
13	68.03	14	68.05	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.03	22	68.37	23	72.73	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.03	31	68.36	32	72.73
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.03	40	68.36
41	72.73	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.00	48	68.03
49	68.36	50	72.73	51	68.00	52	68.00
53	68.00	54	68.00	55	68.01	56	68.04
57	68.01	58	68.04	59	68.03	60	68.06
61	68.08	62	68.37	63	72.73	64	68.03
65	68.03	66	68.03	67	68.03	68	68.02
69	68.05	70	68.05	71	68.07	72	68.07
73	68.47	74	72.76	75	68.36	76	68.36
77	68.36	78	68.36	79	68.37	80	68.07
81	68.43	82	68.06	83	68.45	84	68.07
85	68.10	86	68.73	87	73.05	88	72.73
89	72.73	90	72.73	91	72.73	92	72.73
93	72.74	94	72.76	95	73.05	96	77.32

----- TEMPERATURE OF 2-D ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.01	7	68.03	8	68.04

9	68.23	10	70.58	11	68.02	12	68.12
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.01	19	68.20	20	70.55
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.01	27	68.19	28	70.54
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.01	35	68.19	36	70.54
37	68.00	38	68.00	39	68.00	40	68.00
41	68.01	42	68.02	43	68.20	44	70.55
45	68.01	46	68.01	47	68.01	48	68.01
49	68.03	50	68.03	51	68.04	52	68.13
53	68.25	54	70.58	55	68.05	56	68.06
57	68.07	58	68.06	59	68.06	60	68.08
61	68.19	62	68.19	63	68.19	64	68.20
65	68.13	66	68.23	67	68.25	68	68.34
69	68.34	70	70.75	71	70.54	72	70.54
73	70.54	74	70.55	75	70.57	76	70.60
77	70.75	78	73.04				

\* \* \* PUNCHING ELEMENT DATA \* \* \*

0 SYSTEM ITERATIONS WERE PERFORMED  
 3 B. C. ITERATIONS WERE PERFORMED

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

TIME STEP NUMBER 2 - TIME .0EC - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS -0

FIRE BOUNDARY CONDITION

FIRE(1) = 627.20C  
FIRE(2) = -0.  
FIRE(3) = -0.  
FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.04	7	68.15	8	68.20
9	68.24	10	70.27	11	87.40	12	68.14
13	68.19	14	68.29	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.02
21	68.16	22	69.81	23	87.37	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.01	30	68.16	31	69.78	32	87.35
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.01	39	68.16	40	69.78
41	87.35	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.02	48	68.16
49	69.78	50	87.35	51	68.01	52	68.01
53	68.01	54	68.01	55	68.05	56	68.23
57	68.08	58	68.26	59	68.15	60	68.34
61	68.42	62	69.85	63	87.38	64	68.16
65	68.16	66	68.16	67	68.16	68	68.13
69	68.28	70	68.30	71	68.37	72	68.42
73	70.28	74	87.51	75	69.78	76	69.78
77	69.78	78	69.78	79	69.83	80	68.36
81	70.11	82	68.33	83	70.22	84	68.41
85	68.55	86	71.61	87	88.92	88	87.35
89	87.35	90	87.35	91	87.35	92	87.37
93	87.40	94	87.52	95	88.91	96	105.94

----- TEMPERATURE OF 2-D ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.02	6	68.09	7	68.17	8	68.23

9	69.15	10	78.71	11	68.12	12	68.61
13	68.00	14	68.00	15	68.00	16	68.00
17	68.01	18	68.09	19	68.98	20	78.58
21	68.00	22	68.00	23	68.00	24	68.00
25	68.01	26	68.09	27	68.97	28	78.56
29	68.00	30	68.00	31	68.00	32	68.00
33	68.01	34	68.09	35	68.97	36	78.57
37	68.01	38	68.01	39	68.01	40	68.02
41	68.04	42	68.10	43	68.99	44	78.59
45	68.09	46	68.09	47	68.09	48	68.09
49	68.17	50	68.15	51	68.21	52	68.69
53	69.24	54	78.75	55	68.27	56	68.32
57	68.39	58	68.32	59	68.35	60	68.44
61	68.97	62	68.97	63	68.97	64	68.98
65	68.65	66	69.16	67	69.27	68	69.70
69	69.71	70	79.58	71	78.56	72	78.56
73	78.56	74	78.58	75	78.68	76	78.81
77	79.57	78	88.84				

\* \* \* \* PUNCHING ELEMENT DATA \* \* \*

0 SYSTEM ITERATIONS WERE PERFORMED  
3 B. C. ITERATIONS WERE PERFORMED

\*\*\*\*\*

FIRES-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

TIME STEP NUMBER 3 - TIME .075 - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS -0

FIRE BOUNDARY CONDITION

FIRE(1) = 506.800  
FIRE(2) = -0.  
FIRE(3) = -0.  
FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.01	6	68.16	7	68.52	8	68.66
9	68.79	10	74.58	11	117.37	12	68.47
13	68.63	14	68.95	15	68.00	16	68.00
17	68.00	18	68.00	19	68.01	20	68.06
21	68.53	22	73.37	23	117.27	24	68.00
25	68.00	26	68.00	27	68.00	28	68.01
29	68.05	30	68.54	31	73.27	32	117.21
33	68.00	34	68.00	35	68.00	36	68.00
37	68.01	38	68.05	39	68.54	40	73.28
41	117.21	42	68.01	43	68.01	44	68.01
45	68.01	46	68.02	47	68.07	48	68.55
49	73.28	50	117.22	51	68.05	52	68.05
53	68.05	54	68.05	55	68.18	56	68.78
57	68.29	58	68.90	59	68.54	60	69.13
61	69.37	62	73.45	63	117.32	64	68.54
65	69.54	66	68.54	67	68.54	68	68.46
69	68.94	70	69.00	71	69.24	72	69.37
73	74.66	74	117.73	75	73.27	76	73.27
77	73.27	78	73.28	79	73.39	80	69.19
81	74.15	82	69.10	83	74.52	84	69.35
85	69.75	86	78.60	87	121.75	88	117.21
89	117.21	90	117.21	91	117.22	92	117.28
93	117.36	94	117.77	95	121.75	96	163.90

----- TEMPERATURE OF 2-C ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.01
5	68.06	6	68.30	7	68.57	8	68.76

9	71.42	10	95.65	11	68.42	12	69.87
13	68.00	14	68.00	15	68.00	16	68.00
17	68.03	18	68.30	19	70.93	20	95.28
21	68.00	22	68.00	23	68.00	24	68.00
25	68.03	26	68.30	27	70.91	28	95.24
29	68.00	30	68.00	31	68.00	32	68.01
33	68.04	34	68.30	35	70.91	36	95.25
37	68.03	38	68.03	39	68.03	40	68.06
41	68.14	42	68.36	43	70.96	44	95.32
45	68.30	46	68.30	47	68.30	48	68.31
49	68.59	50	68.54	51	68.72	52	70.12
53	71.71	54	95.79	55	68.90	56	69.07
57	69.28	58	69.06	59	69.17	60	69.42
61	70.91	62	70.91	63	70.91	64	70.92
65	70.00	66	71.46	67	71.78	68	73.05
69	73.09	70	98.19	71	95.24	72	95.24
73	95.25	74	95.29	75	95.55	76	95.95
77	98.16	78	121.50				

\* \* \* \* PUNCHING ELEMENT DATA \* \* \*

0 SYSTEM ITERATIONS WERE PERFORMED  
 3 B. C. ITERATIONS WERE PERFORMED

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

TIME STEP NUMBER 4 - TIME .100 - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS -0

FIRE BOUNDARY CONDITION

FIRE(1) = 1060.000  
FIRE(2) = -0.  
FIRE(3) = -0.  
FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.03	6	68.43	7	69.29	8	69.59
9	69.86	10	81.66	11	156.38	12	69.18
13	69.54	14	70.21	15	68.00	16	68.00
17	68.00	18	68.00	19	68.02	20	68.17
21	69.29	22	79.41	23	156.16	24	68.00
25	68.00	26	68.00	27	68.00	28	68.02
29	68.15	30	69.32	31	79.23	32	156.02
33	68.00	34	68.00	35	68.00	36	68.00
37	68.02	38	68.15	39	69.32	40	79.24
41	156.02	42	68.02	43	68.02	44	68.02
45	68.03	46	68.06	47	68.19	48	69.35
49	79.26	50	156.05	51	68.15	52	68.15
53	68.15	54	68.15	55	68.48	56	69.95
57	68.78	58	70.22	59	69.38	60	70.73
61	71.23	62	79.56	63	156.27	64	69.32
65	69.32	66	69.32	67	69.33	68	69.21
69	70.28	70	70.43	71	70.96	72	71.23
73	82.00	74	157.24	75	79.23	76	79.23
77	79.23	78	79.24	79	79.40	80	70.81
81	80.89	82	70.64	83	81.71	84	71.20
85	72.03	86	90.37	87	165.46	88	156.02
89	156.02	90	156.02	91	156.03	92	156.16
93	156.36	94	157.32	95	165.44	96	238.09

----- TEMPERATURE OF 2-D ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.02
5	68.16	6	68.76	7	69.40	8	69.80

9	75.29	10	118.40	11	69.04	12	72.11
13	68.00	14	68.00	15	68.00	16	68.01
17	68.09	18	68.73	19	74.31	20	117.70
21	68.00	22	68.00	23	68.00	24	68.01
25	68.08	26	68.74	27	74.28	28	117.63
29	68.01	30	68.01	31	68.01	32	68.03
33	68.10	34	68.75	35	74.29	36	117.64
37	68.08	38	68.08	39	68.09	40	68.18
41	68.38	42	68.92	43	74.39	44	117.78
45	68.73	46	68.73	47	68.74	48	68.79
49	69.48	50	69.36	51	69.78	52	72.73
53	76.01	54	118.77	55	70.22	56	70.58
57	71.04	58	70.54	59	70.80	60	71.35
61	74.28	62	74.28	63	74.28	64	74.30
65	72.43	66	75.44	67	76.11	68	78.83
69	78.91	70	123.77	71	117.62	72	117.62
73	117.63	74	117.71	75	118.20	76	119.07
77	123.71	78	164.84				

\* \* \* \* PUNCHING ELEMENT DATA \* \* \*

0 SYSTEM ITERATIONS WERE PERFORMED  
3 B. C. ITERATIONS WERE PERFORMED

\*\*\*\*\*

FIRES-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\*\* INTERIOR BASEMENT COLUMN - BLDG STUDY - ASTM FIRE \*\*\*

TIME STEP NUMBER 5 - TIME .125 - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS -0

FIRE BOUNDARY CONCITION

FIRE(1) = 1150.000

FIRE(2) = -0.

FIRE(3) = -0.

FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.01
5	68.08	6	68.94	7	70.59	8	71.13
9	71.59	10	91.21	11	199.03	12	70.38
13	71.04	14	72.19	15	68.00	16	68.00
17	68.00	18	68.01	19	68.06	20	68.37
21	70.58	22	87.87	23	198.64	24	68.00
25	68.00	26	68.00	27	68.01	28	68.04
29	68.32	30	70.64	31	87.60	32	198.41
33	68.00	34	68.00	35	68.01	36	68.01
37	68.05	38	68.34	39	70.63	40	87.60
41	198.41	42	68.04	43	68.04	44	68.04
45	68.07	46	68.14	47	68.43	48	70.71
49	87.65	50	198.46	51	68.32	52	68.33
53	68.33	54	68.35	55	69.06	56	71.93
57	69.69	58	72.43	59	70.85	60	73.35
61	74.21	62	88.10	63	198.87	64	70.62
65	70.62	66	70.63	67	70.66	68	70.56
69	72.53	70	72.80	71	73.75	72	74.23
73	92.09	74	200.71	75	87.60	76	87.60
77	87.60	78	87.61	79	87.74	80	73.43
81	90.11	82	73.16	83	91.64	84	74.16
85	75.59	86	106.61	87	214.40	88	198.40
89	198.40	90	198.40	91	198.43	92	198.65
93	199.02	94	200.84	95	214.37	96	317.37

----- TEMPERATURE OF 2-D ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.04
5	68.36	6	69.57	7	70.78	8	71.48

9	80.71	10	144.19	11	70.09	12	75.42
13	68.00	14	68.00	15	68.00	16	68.03
17	68.20	18	69.48	19	79.17	20	143.13
21	68.00	22	68.00	23	68.01	24	68.03
25	68.19	26	69.48	27	79.12	28	143.00
29	68.02	30	68.02	31	68.03	32	68.07
33	68.24	34	69.53	35	79.15	36	143.03
37	68.18	38	68.18	39	68.20	40	68.41
41	68.83	42	69.92	43	79.33	44	143.27
45	69.47	46	69.48	47	69.49	48	69.66
49	71.02	50	70.78	51	71.58	52	76.63
53	82.16	54	144.94	55	72.42	56	73.08
57	73.88	58	72.98	59	73.46	60	74.43
61	79.11	62	79.11	63	79.12	64	79.14
65	76.06	66	81.11	67	82.27	68	87.00
69	87.13	70	153.45	71	143.00	72	143.00
73	143.01	74	143.11	75	143.88	76	145.40
77	153.36	78	213.19				

\* \* \* \* PUNCHING ELEMENT DATA \* \* \*

0 SYSTEM ITERATIONS WERE PERFORMED  
 3 B. C. ITERATIONS WERE PERFORMED

## SAMPLE PROBLEM 2 - COLUMN WITH SPIRAL

The second sample problem is a circular column with longitudinal and spiral reinforcing which is uniformly exposed to fire. The dimensions and configuration of the column are shown in Fig. B.6. The column is subjected to each of the two fire curves discussed in the previous sample problem, the ASTM E-119 long-duration, moderate intensity fire (Fig. B.3) and the SDHI short-duration, high intensity fire (Fig. B.4). The fire boundary is modeled nonlinearly as in the previous sample problem. Both material properties and the boundary properties are given in the sample output in Table B.4.

For each fire curve two analyses are made:

1. A three-dimensional thermal analysis which is capable of modeling the effect spiral reinforcing has on heat flow into the column's interior. Due to symmetry only the wedge-shaped portion shown in Fig. B.6 need be discretized into elements. The mesh chosen is shown in Fig. B.7. Both the input data (Table B.3) and the output (Table B.4) is given for the first three time steps of the SDHI fire analysis. The temperature distribution within the column after one hour of fire exposure is shown in Fig. B.8.
2. A two-dimensional thermal analysis which ignores the effect of the spiral steel on heat flow into the column's interior. A mesh of two-dimensional quadrilateral elements which is equivalent to the three-dimensional mesh is defined by the top surface of the wedge-shaped mesh in Fig. B.7. The temperature distribution within the column after one hour of fire exposure is shown in Fig. B.9.

The time-temperature history of the longitudinal reinforcing bars is shown in Fig. B.10 for each of the four analyses conducted. Note that the steel spiral does considerably affect heat conduction into the interior of the column.

The SDHI fire analyses required 80 time steps for two hours duration. The three-dimensional analysis needed 902 seconds of central processing time (CDC 6400) and the two-dimensional analysis 62 seconds. For the ASTM fire, 120 time steps were needed for a two hour analysis. The three-dimensional case consumed 1520 seconds of central processing time and the two-dimensional case 105 seconds.

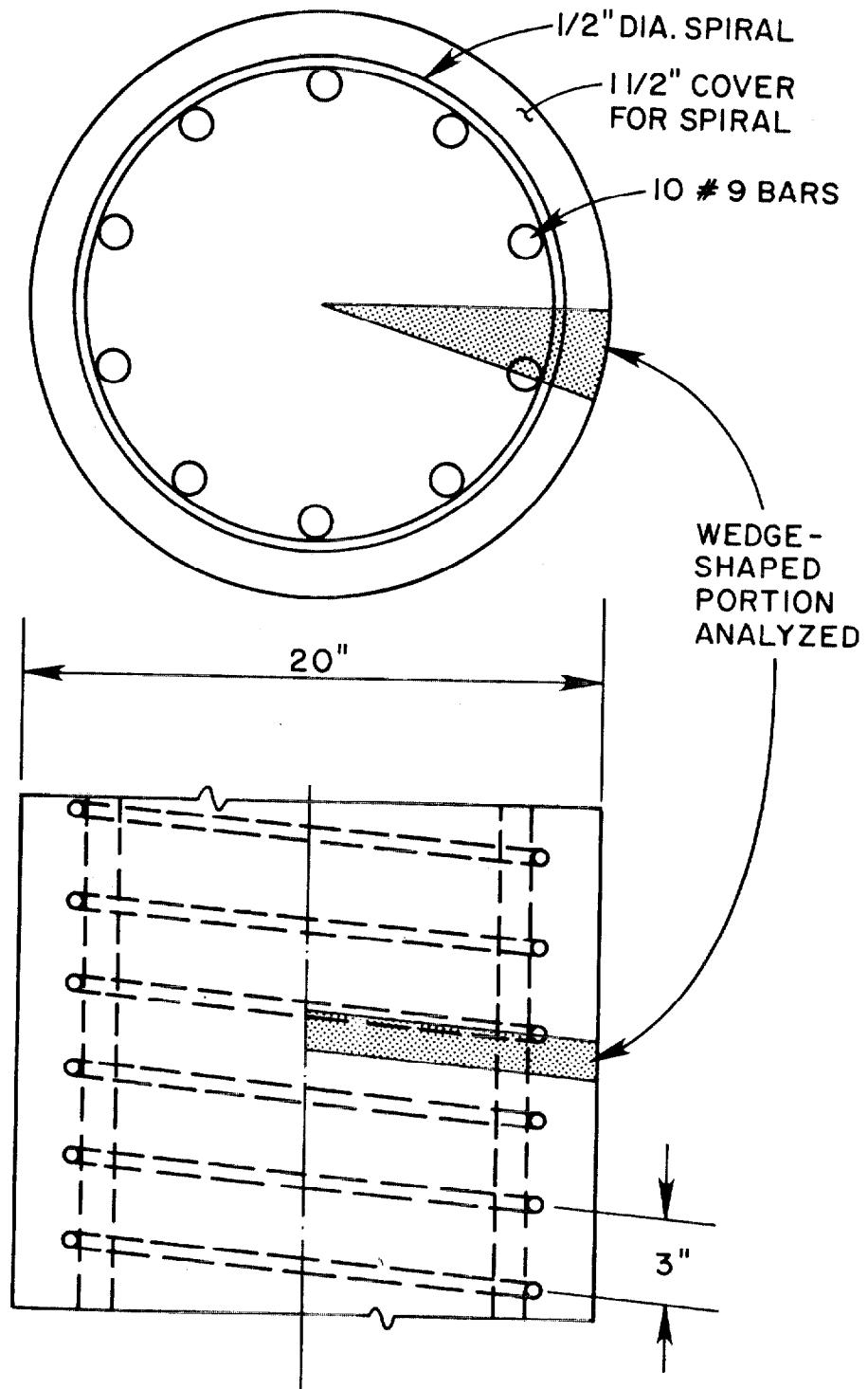


FIGURE B.6 CROSS-SECTIONS OF CIRCULAR COLUMN

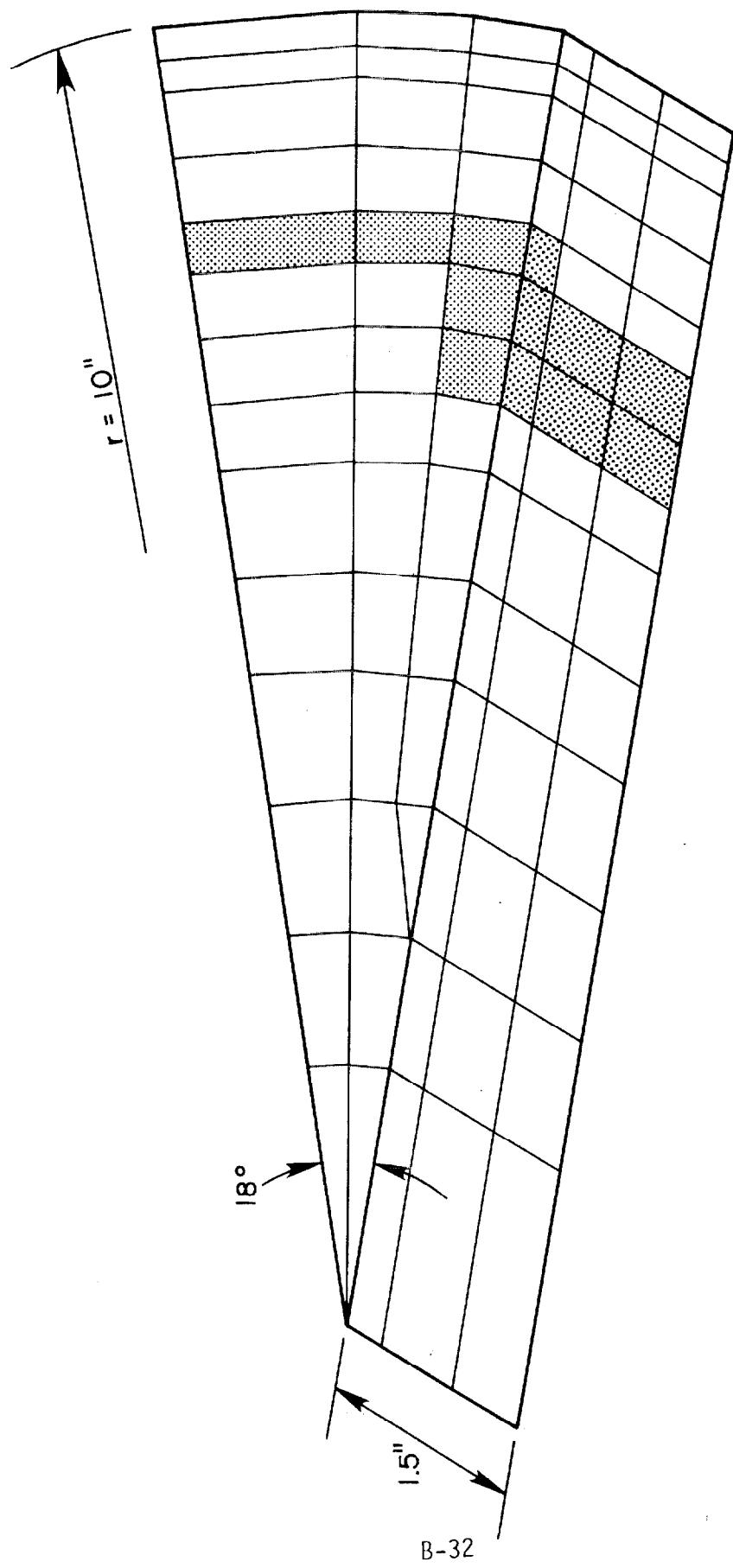


FIGURE B.7 THREE-DIMENSIONAL MESH FOR CIRCULAR COLUMN (WEDGE)

(ISOTHERMS PLOTTED FOR CROSS-SECTION THROUGH SPIRAL)

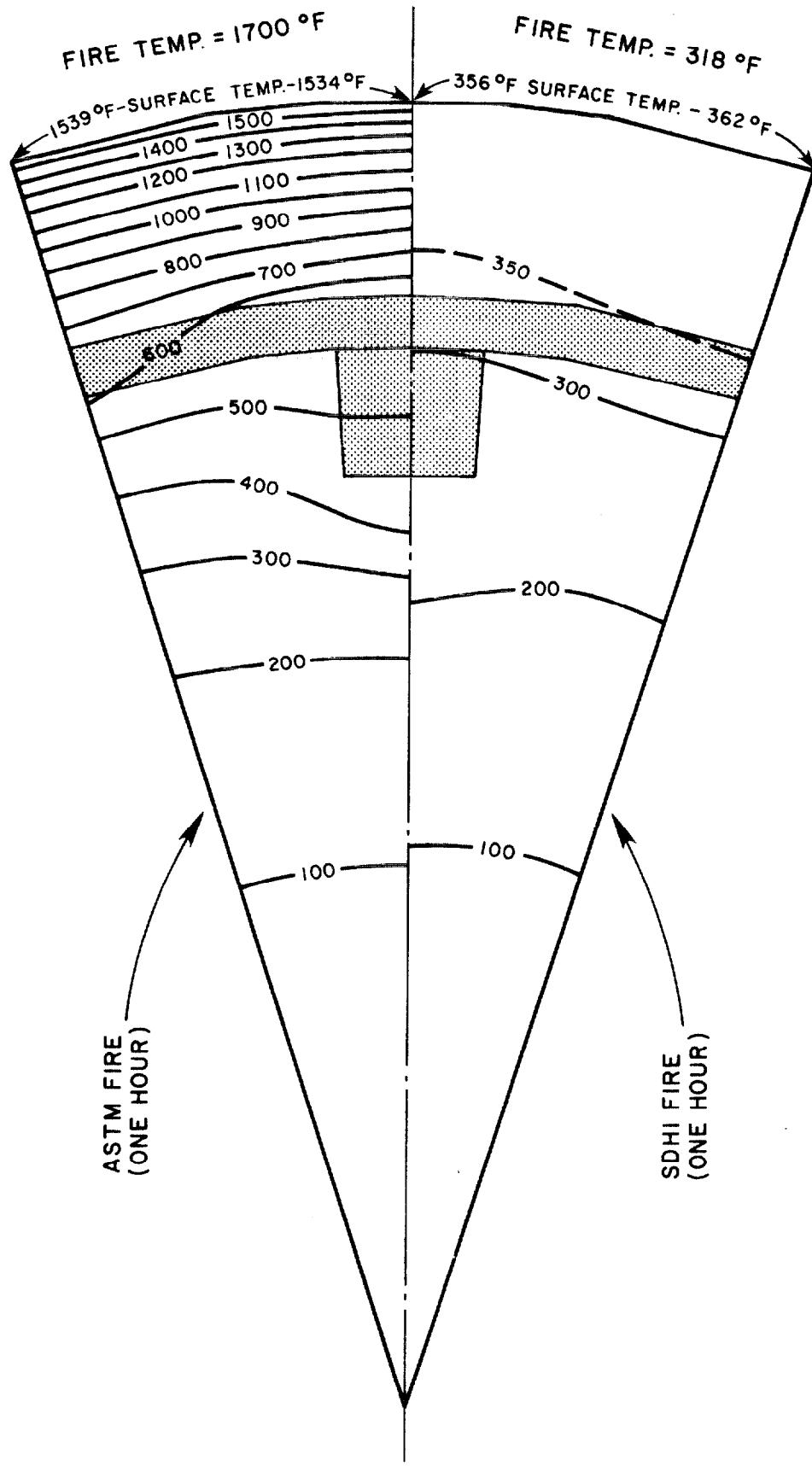


FIGURE B.8 ISOTHERMS IN CROSS-SECTION - THREE-DIMENSIONAL ANALYSIS WHICH INCLUDES EFFECT OF SPIRAL REINFORCING

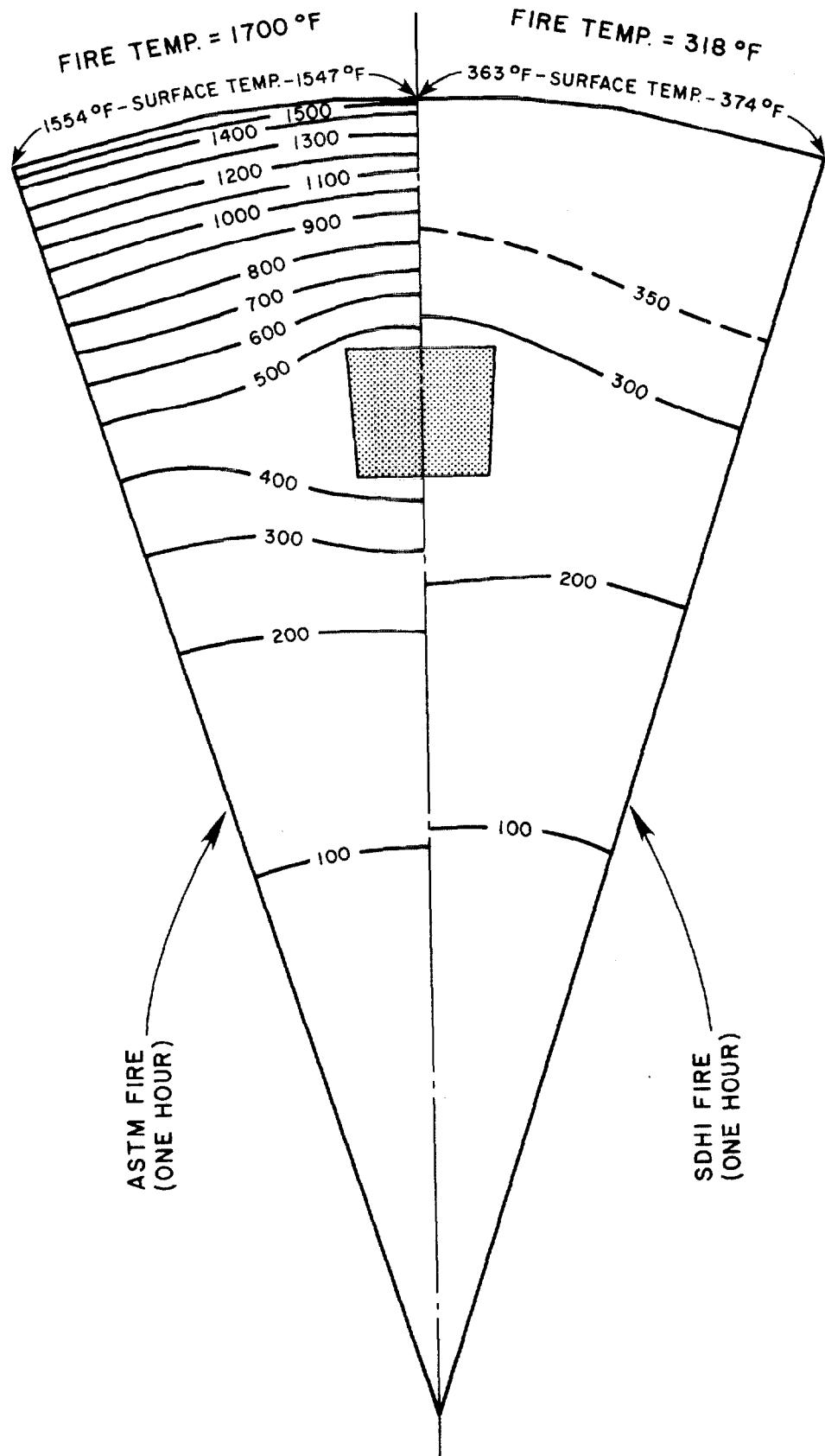


FIGURE B.9 ISOETHERMS IN CROSS-SECTION - TWO-DIMENSIONAL ANALYSIS WHICH IGNORES EFFECT OF SPIRAL REINFORCING

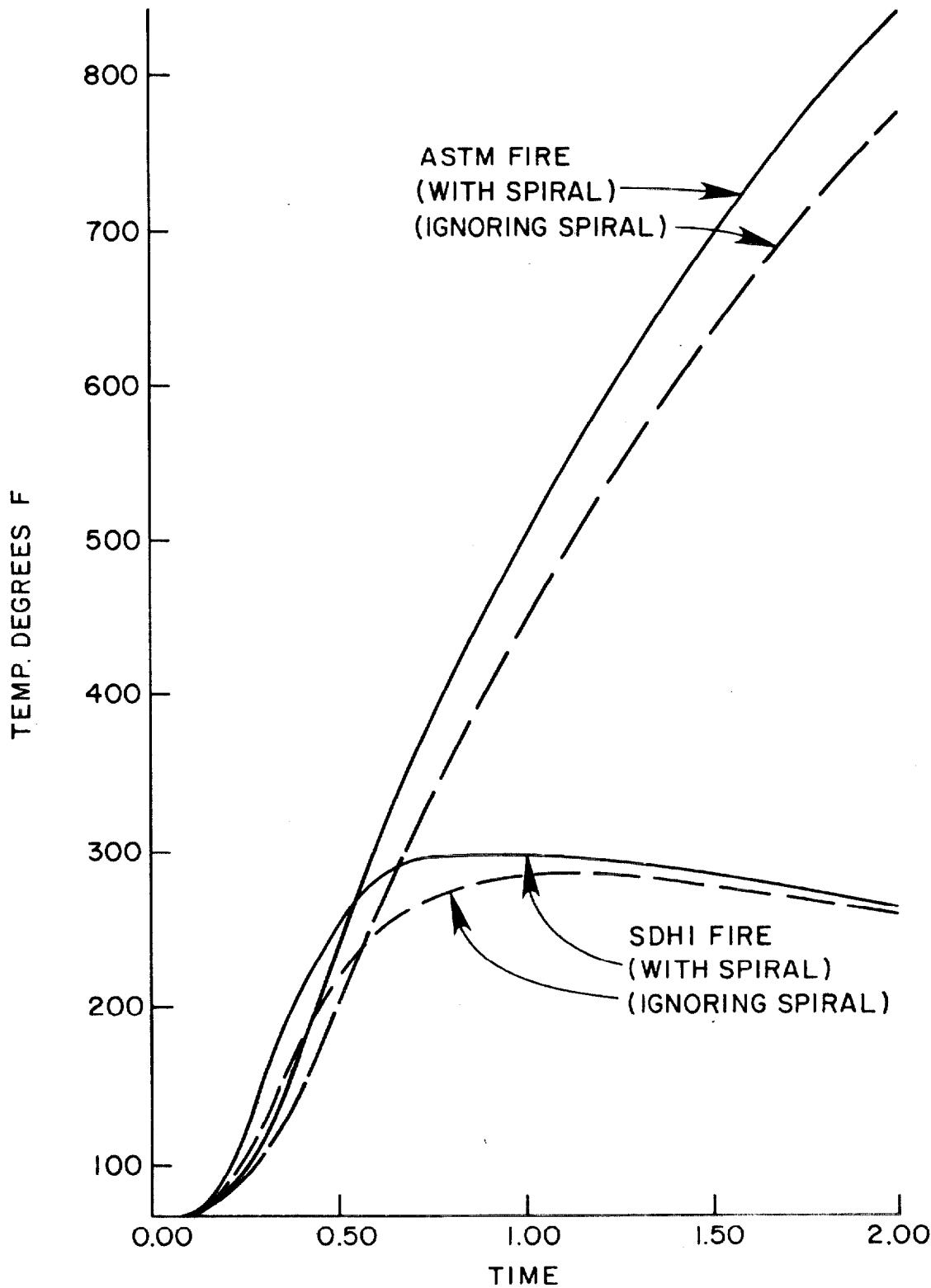


FIGURE B.10 TEMPERATURE-TIME HISTORY IN REINFORCING BARS  
(AT CENTER OF BAR AT LEVEL OF SPIRAL)

	0	20	30	40
** CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS **				
NODES,220,0				
1	0.	0.	0.	
3	0.	0.	.1042	
4	0.	0.	.1250	
5	.1646	-.0261	0.	
7	.1646	-.0261	.1042	
8	.1646	-.0261	.1250	
9	.1667	0.	0.	
11	.1667	0.	.1042	
12	.1667	0.	.1250	
13	.1646	.0261	0.	
15	.1646	.0261	.1042	
16	.1646	.0261	.1250	
17	.2469	-.0391	0.	
19	.2469	-.0391	.1042	
20	.2469	-.0391	.1250	
21	.2500	0.	0.	
23	.2500	0.	.1042	
24	.2500	0.	.1250	
25	.2469	.0391	0.	
27	.2469	.0391	.1042	
28	.2469	.0391	.1250	
29	.3292	-.0522	0.	
31	.3292	-.0522	.1042	
32	.3292	-.0522	.1250	
33	.3320	-.0302	0.	
35	.3320	-.0302	.1042	
36	.3320	-.0302	.1250	
37	.3333	0.	0.	
39	.3333	0.	.1042	
40	.3333	0.	.1250	
41	.3292	.0522	0.	
43	.3292	.0522	.1042	
44	.3292	.0522	.1250	
45	.4115	-.0652	0.	
47	.4115	-.0652	.1042	
48	.4115	-.0652	.1250	
49	.4149	-.0377	0.	
51	.4149	-.0377	.1042	
52	.4149	-.0377	.1250	
53	.4167	0.	0.	
55	.4167	0.	.1042	
56	.4167	0.	.1250	
57	.4115	.0652	0.	
59	.4115	.0652	.1042	
60	.4115	.0652	.1250	
61	.4733	-.0749	0.	
63	.4733	-.0749	.1042	
64	.4733	-.0749	.1250	
65	.4772	-.0434	0.	
67	.4772	-.0434	.1042	
68	.4772	-.0434	.1250	
69	.4792	0.	0.	
71	.4792	0.	.1042	

TABLE B.3 INPUT FOR SAMPLE PROBLEM TWO

	10	20	30	40
72	.4792	0.	.1250	
73	.4733	.0749	0.	
75	.4733	.0749	.1042	
76	.4733	.0749	.1250	
77	.5350	-.0848	0.	
79	.5350	-.0848	.1042	
80	.5350	-.0848	.1250	
81	.5394	-.0492	0.	
83	.5394	-.0492	.1042	
84	.5394	-.0492	.1250	
85	.5417	0.	0.	
87	.5417	0.	.1042	
88	.5417	0.	.1250	
89	.5350	.0848	0.	
91	.5350	.0848	.1042	
92	.5350	.0848	.1250	
93	.5850	-.0927	0.	
95	.5850	-.0927	.1042	
96	.5850	-.0927	.1250	
97	.5899	-.0537	0.	
99	.5899	-.0537	.1042	
100	.5899	-.0537	.1250	
101	.5923	0.	0.	
103	.5923	0.	.1042	
104	.5923	0.	.1250	
105	.5850	-.0927	0.	
107	.5850	.0927	.1042	
108	.5850	.0927	.1250	
109	.6262	-.0992	0.	
111	.6262	-.0992	.1042	
112	.6262	-.0992	.1250	
113	.6314	-.0575	0.	
115	.6314	-.0575	.1042	
116	.6314	-.0575	.1250	
117	.6340	0.	0.	
119	.6340	0.	.1042	
120	.6340	0.	.1250	
121	.6262	.0992	0.	
123	.6262	.0992	.1042	
124	.6262	.0992	.1250	
125	.6673	-.1057	0.	
127	.6673	-.1057	.1042	
128	.6673	-.1057	.1250	
129	.6729	-.0613	0.	
131	.6729	-.0613	.1042	
132	.6729	-.0613	.1250	
133	.6757	0.	0.	
135	.6757	0.	.1042	
136	.6757	0.	.1250	
137	.6673	.1057	0.	
139	.6673	.1057	.1042	
140	.6673	.1057	.1250	
141	.6996	-.1108	0.	
143	.6996	-.1108	.1042	
144	.6996	-.1108	.1250	
145	.7054	-.0642	0.	

	10	20	30	40
147	.7054	-.0642	.1042	
148	.7054	-.0642	.1250	
149	.7083	0.	0.	
151	.7083	0.	.1042	
152	.7083	0.	.1250	
153	.6996	.1108	0.	
155	.6996	.1108	.1042	
156	.6996	.1108	.1250	
157	.7408	-.1173	0.	
159	.7408	-.1173	.1042	
160	.7408	-.1173	.1250	
161	.7469	-.0680	0.	
163	.7469	-.0680	.1042	
164	.7469	-.0680	.1250	
165	.7500	0.	0.	
167	.7500	0.	.1042	
168	.7500	0.	.1250	
169	.7408	.1173	0.	
171	.7408	.1173	.1042	
172	.7408	.1173	.1250	
173	.7819	-.1238	0.	
175	.7819	-.1238	.1042	
176	.7819	-.1238	.1250	
177	.7884	-.0718	0.	
179	.7884	-.0718	.1042	
180	.7884	-.0718	.1250	
181	.7917	0.	0.	
183	.7917	0.	.1042	
184	.7917	0.	.1250	
185	.7819	.1238	0.	
187	.7819	.1238	.1042	
188	.7819	.1238	.1250	
189	.8025	-.1271	0.	
191	.8025	-.1271	.1042	
192	.8025	-.1271	.1250	
193	.8092	-.0737	0.	
195	.8092	-.0737	.1042	
196	.8092	-.0737	.1250	
197	.8125	0.	0.	
199	.8125	0.	.1042	
200	.8125	0.	.1250	
201	.8025	.1271	0.	
203	.8025	.1271	.1042	
204	.8025	.1271	.1250	
205	.8231	-.1303	0.	
207	.8231	-.1303	.1042	
208	.8231	-.1303	.1250	
209	.8299	-.0756	0.	
211	.8299	-.0756	.1042	
212	.8299	-.0756	.1250	
213	.8333	0.	0.	
215	.8333	0.	.1042	
216	.8333	0.	.1250	
217	.8231	.1303	0.	
219	.8231	.1303	.1042	
220	.8231	.1303	.1250	

	10	20	30	40	50	60	70	80
ELEMENTS, 0, 0, 120								
1	1	5	9	9	2	6	10	1
4	1	9	13	13	2	10	14	1
7	5	17	21	9	6	18	22	10
10	9	21	25	13	10	22	26	14
13	17	29	33	33	18	30	34	34
16	17	33	37	21	18	34	38	22
19	21	37	41	25	22	38	42	26
22	29	45	49	33	30	46	50	34
25	33	49	53	37	34	50	54	38
28	37	53	57	41	38	54	58	42
31	45	61	65	49	46	62	66	50
34	49	65	69	53	50	66	70	54
37	53	69	73	57	54	70	74	58
40	61	77	81	65	62	78	82	66
43	65	81	85	69	66	82	86	70
46	69	85	89	73	70	86	90	74
49	77	93	97	81	78	94	98	82
52	81	97	101	85	82	98	102	86
55	85	101	105	89	86	102	106	90
58	93	109	113	97	94	110	114	98
61	97	113	117	101	98	114	118	102
64	101	117	121	105	102	118	122	106
67	109	125	129	113	110	126	130	114
70	113	129	133	117	114	130	134	118
73	117	133	137	121	118	134	138	122
76	125	141	145	129	126	142	146	130
78	127	143	147	131	128	144	148	132
79	129	145	149	133	130	146	150	134
81	131	147	151	135	132	148	152	136
82	133	149	153	137	134	150	154	138
84	135	151	155	139	136	152	156	140
85	141	157	161	145	142	158	162	146
88	145	161	165	149	146	162	166	150
91	149	165	169	153	150	166	170	154
94	157	173	177	161	158	174	178	162
97	161	177	181	165	162	178	182	166
100	165	181	185	169	166	182	186	170
103	173	189	193	177	174	190	194	178
106	177	193	197	181	178	194	198	182
109	181	197	201	185	182	198	202	186
112	189	205	209	193	190	206	210	194
115	193	209	213	197	194	210	214	198
118	197	213	217	201	198	214	218	202
120	199	215	219	203	200	216	220	204
MATERIALS, 2								
4	0	0						
	0.0	0.90	390.0		0.90	1650.0	0.506	3000.0
	0.272							
	150.0							
3	4	0						
	0.0	30.00	1100.0		19.90	3000.0	19.90	
	0.0	0.107	750.0		0.144	1100.0	0.172	3000.0
	480.0							
FIRE, 0, 0, 9, 1								
NONLINEAR								

<b>1.7E-9</b>	<b>460.0</b>										
<b>.27</b>	<b>1.25</b>	<b>1.0</b>	<b>0.9</b>	<b>0.9</b>	<b>0.9</b>						
<b>SURFACE,0,0,9</b>											
205	209	210	206	1	1	206	210	211	207	1	1
207	211	212	208	1	1	209	213	214	210	1	1
210	214	215	211	1	1	211	215	216	212	1	1
213	217	218	214	1	1	214	218	219	215	1	1
215	219	220	216	1	1						
<b>EXOTHERMIC,0,0,0,0</b>											
<b>CONVERGENCE</b>											
30	<b>.005</b>	<b>-.25</b>									
STEP	0	<b>0.0</b>	<b>68.0</b>								
STEP	1	<b>.025</b>	<b>347.600</b>								<b>3</b>
STEP	2	<b>.025</b>	<b>627.200</b>								<b>3</b>
STEP	3	<b>.025</b>	<b>906.800</b>								<b>3</b>
STEP	4	<b>.025</b>	<b>1060.000</b>								<b>3</b>
STEP	5	<b>.025</b>	<b>1150.000</b>								<b>3</b>

\*\*\*\*\*

FFFFF	I	RRRRR	EEEEE	SSSSS	TTTTT	23333
F	I	R R	E	S	T	3
F	I	R P	F	S	T	3
F	I	R R	E	S	T	3
FFF	I	PRRRR	EEE	SSSSS	=====	T 3333
F	I	RR	E	S	T	3
F	I	R R	E	S	T	3
F	I	R P	E	S	T	3
F	I	R R	EEEEE	SSSSS	T	33333

A THERMAL ANALYZER FOR THREE-DIMENSIONAL SYSTEMS,  
WITH TEMPERATURE-DEPENDENT THERMAL PROPERTIES,  
SUBJECTED TO A FIRE ENVIRONMENT

\*\*\*\*\*

- - - TITLE OF RUN - - -

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

\*\*\*\*\*

TABLE B.4 OUTPUT FOR SAMPLE PROBLEM TWO

\*\*\*\*\*

FIREST-3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

GEOMETRIC DESCRIPTION OF SYSTEM TO BE ANALYZED

\*\*\*\*\*

\* \* \* \* THERE ARE 220 NODAL POINTS \* \* \*

NODAL POINT	X	COORDINATES Y	Z	BOUNDARY CONDITION
1	0.	0.	0.	FLOW
2	0.	0.	.0521	FLOW
3	0.	0.	.1042	FLOW
4	0.	0.	.1250	FLOW
5	.1646	-.0261	0.	FLOW
6	.1646	-.0261	.0521	FLOW
7	.1646	-.0261	.1042	FLOW
8	.1646	-.0261	.1250	FLOW
9	.1667	0.	0.	FLOW
10	.1667	0.	.0521	FLOW
11	.1667	0.	.1042	FLOW
12	.1667	0.	.1250	FLOW
13	.1646	.0261	0.	FLOW
14	.1646	.0261	.0521	FLOW
15	.1646	.0261	.1042	FLOW
16	.1646	.0261	.1250	FLOW
17	.2469	-.0391	0.	FLOW
18	.2469	-.0391	.0521	FLOW
19	.2469	-.0391	.1042	FLOW
20	.2469	-.0391	.1250	FLOW
21	.2500	0.	0.	FLOW
22	.2500	0.	.0521	FLOW
23	.2500	0.	.1042	FLOW
24	.2500	0.	.1250	FLOW
25	.2469	.0391	0.	FLOW
26	.2469	.0391	.0521	FLOW
27	.2469	.0391	.1042	FLOW
28	.2469	.0391	.1250	FLOW
29	.3292	-.0522	0.	FLOW
30	.3292	-.0522	.0521	FLOW
31	.3292	-.0522	.1042	FLOW
32	.3292	-.0522	.1250	FLOW
33	.3320	-.0302	0.	FLOW
34	.3320	-.0302	.0521	FLOW
35	.3320	-.0302	.1042	FLOW
36	.3320	-.0302	.1250	FLOW
37	.3333	0.	0.	FLOW
38	.3333	0.	.0521	FLOW

39	.3333	0.	.1042	FLOW
40	.3333	0.	.1250	FLOW
41	.3292	.0522	0.	FLOW
42	.3292	.0522	.0521	FLOW
43	.3292	.0522	.1042	FLOW
44	.3292	.0522	.1250	FLOW
45	.4115	-.0652	0.	FLOW
46	.4115	-.0652	.0521	FLOW
47	.4115	-.0652	.1042	FLOW
48	.4115	-.0652	.1250	FLOW
49	.4149	-.0377	0.	FLOW
50	.4149	-.0377	.0521	FLOW
51	.4149	-.0377	.1042	FLOW
52	.4149	-.0377	.1250	FLOW
53	.4167	0.	0.	FLOW
54	.4167	0.	.0521	FLOW
55	.4167	0.	.1042	FLOW
56	.4167	0.	.1250	FLOW
57	.4115	.0652	0.	FLOW
58	.4115	.0652	.0521	FLOW
59	.4115	.0652	.1042	FLOW
60	.4115	.0652	.1250	FLOW
61	.4733	-.0749	0.	FLOW
62	.4733	-.0749	.0521	FLOW
63	.4733	-.0749	.1042	FLOW
64	.4733	-.0749	.1250	FLOW
65	.4772	-.0434	0.	FLOW
66	.4772	-.0434	.0521	FLOW
67	.4772	-.0434	.1042	FLOW
68	.4772	-.0434	.1250	FLOW
69	.4792	0.	0.	FLOW
70	.4792	0.	.0521	FLOW
71	.4792	0.	.1042	FLOW
72	.4792	0.	.1250	FLOW
73	.4733	.0749	0.	FLOW
74	.4733	.0749	.0521	FLOW
75	.4733	.0749	.1042	FLOW
76	.4733	.0749	.1250	FLOW
77	.5350	-.0848	0.	FLOW
78	.5350	-.0848	.0521	FLOW
79	.5350	-.0848	.1042	FLOW
80	.5350	-.0848	.1250	FLOW
81	.5394	-.0492	0.	FLOW
82	.5394	-.0492	.0521	FLOW
83	.5394	-.0492	.1042	FLOW
84	.5394	-.0492	.1250	FLOW
85	.5417	0.	0.	FLOW
86	.5417	0.	.0521	FLOW
87	.5417	0.	.1042	FLOW
88	.5417	0.	.1250	FLOW
89	.5350	.0848	0.	FLOW
90	.5350	.0848	.0521	FLOW
91	.5350	.0848	.1042	FLOW
92	.5350	.0848	.1250	FLOW
93	.5350	-.0927	0.	FLOW
94	.5850	-.0927	.0521	FLOW

95	.5850	-.0927	.1042	FLOW
96	.5850	-.0927	.1250	FLOW
97	.5899	-.0537	0.	FLOW
98	.5899	-.0537	.0521	FLOW
99	.5899	-.0537	.1042	FLOW
100	.5899	-.0537	.1250	FLOW
101	.5923	0.	0.	FLOW
102	.5923	0.	.0521	FLOW
103	.5923	0.	.1042	FLOW
104	.5923	0.	.1250	FLOW
105	.5850	.0927	0.	FLOW
106	.5850	.0927	.0521	FLOW
107	.5850	.0927	.1042	FLOW
108	.5850	.0927	.1250	FLOW
109	.6262	-.0992	0.	FLOW
110	.6262	-.0992	.0521	FLOW
111	.6262	-.0992	.1042	FLOW
112	.6262	-.0992	.1250	FLOW
113	.6314	-.0575	0.	FLOW
114	.6314	-.0575	.0521	FLOW
115	.6314	-.0575	.1042	FLOW
116	.6314	-.0575	.1250	FLOW
117	.6340	0.	0.	FLOW
118	.6340	0.	.0521	FLOW
119	.6340	0.	.1042	FLOW
120	.6340	0.	.1250	FLOW
121	.6262	.0992	0.	FLOW
122	.6262	.0992	.0521	FLOW
123	.6262	.0992	.1042	FLOW
124	.6262	.0992	.1250	FLOW
125	.6673	-.1057	0.	FLOW
126	.6673	-.1057	.0521	FLOW
127	.6673	-.1057	.1042	FLOW
128	.6673	-.1057	.1250	FLOW
129	.6729	-.0613	0.	FLOW
130	.6729	-.0613	.0521	FLOW
131	.6729	-.0613	.1042	FLOW
132	.6729	-.0613	.1250	FLOW
133	.6757	0.	0.	FLOW
134	.6757	0.	.0521	FLOW
135	.6757	0.	.1042	FLOW
136	.6757	0.	.1250	FLOW
137	.6673	.1057	0.	FLOW
138	.6673	.1057	.0521	FLOW
139	.6673	.1057	.1042	FLOW
140	.6673	.1057	.1250	FLOW
141	.6996	-.1108	0.	FLOW
142	.6996	-.1108	.0521	FLOW
143	.6996	-.1108	.1042	FLOW
144	.6996	-.1108	.1250	FLOW
145	.7054	-.0642	0.	FLOW
146	.7054	-.0642	.0521	FLOW
147	.7054	-.0642	.1042	FLOW
148	.7054	-.0642	.1250	FLOW
149	.7083	0.	0.	FLOW
150	.7083	0.	.0521	FLOW

151	.7083	0.	.1042	FLOW
152	.7083	0.	.1250	FLOW
153	.6996	.1108	0.	FLCW
154	.6996	.1108	.0521	FLOW
155	.6996	.1108	.1042	FLOW
156	.6996	.1108	.1250	FLOW
157	.7408	-.1173	0.	FLOW
158	.7408	-.1173	.0521	FLCW
159	.7408	-.1173	.1042	FLOW
160	.7408	-.1173	.1250	FLCW
161	.7469	-.0680	0.	FLOW
162	.7469	-.0680	.0521	FLOW
163	.7469	-.0680	.1042	FLCW
164	.7469	-.0680	.1250	FLOW
165	.7500	0.	0.	FLCW
166	.7500	0.	.0521	FLOW
167	.7500	0.	.1042	FLOW
168	.7500	0.	.1250	FLCW
169	.7408	.1173	0.	FLOW
170	.7408	.1173	.0521	FLCW
171	.7408	.1173	.1042	FLOW
172	.7408	.1173	.1250	FLOW
173	.7819	-.1238	0.	FLCW
174	.7819	-.1238	.0521	FLOW
175	.7819	-.1238	.1042	FLCW
176	.7819	-.1238	.1250	FLOW
177	.7884	-.0718	0.	FLOW
178	.7884	-.0718	.0521	FLCW
179	.7884	-.0718	.1042	FLOW
180	.7884	-.0718	.1250	FLCW
181	.7917	0.	0.	FLOW
182	.7917	0.	.0521	FLCW
183	.7917	0.	.1042	FLOW
184	.7917	0.	.1250	FLCW
185	.7819	.1238	0.	FLCW
186	.7819	.1238	.0521	FLOW
187	.7819	.1238	.1042	FLCW
188	.7819	.1238	.1250	FLOW
189	.8025	-.1271	0.	FLCW
190	.8025	-.1271	.0521	FLOW
191	.8025	-.1271	.1042	FLCW
192	.8025	-.1271	.1250	FLOW
193	.8092	-.0737	0.	FLOW
194	.8092	-.0737	.0521	FLCW
195	.8092	-.0737	.1042	FLOW
196	.8092	-.0737	.1250	FLCW
197	.8125	0.	0.	FLOW
198	.8125	0.	.0521	FLCW
199	.8125	0.	.1042	FLOW
200	.8125	0.	.1250	FLCW
201	.8025	.1271	0.	FLCW
202	.8025	.1271	.0521	FLOW
203	.8025	.1271	.1042	FLCW
204	.8025	.1271	.1250	FLOW
205	.8231	-.1303	0.	FLOW
206	.8231	-.1303	.0521	FLCW

207	.8231		-.1303		.1042	FLOW
208	.8231		-.1303		.1250	FLOW
209	.8299		-.0756		0.	FLOW
210	.8299		-.0756		.0521	FLOW
211	.8299		-.0756		.1042	FLOW
212	.8299		-.0756		.1250	FLOW
213	.8333		0.		0.	FLOW
214	.8333		0.		.0521	FLOW
215	.8333		0.		.1042	FLOW
216	.8333		0.		.1250	FLOW
217	.8231		.1303		0.	FLOW
218	.8231		.1303		.0521	FLOW
219	.8231		.1303		.1042	FLOW
220	.8231		.1303		.1250	FLOW

\* \* \* \* THERE ARE 120 3-D ELEMENTS \* \* \*

ELMT	I	J	K	L	M	N	O	P	MAT
1	1	5	9	9	2	6	10	10	1
2	2	6	10	10	3	7	11	11	1
3	3	7	11	11	4	8	12	12	1
4	1	9	13	13	2	10	14	14	1
5	2	10	14	14	3	11	15	15	1
6	3	11	15	15	4	12	16	16	1
7	5	17	21	9	6	18	22	10	1
8	6	18	22	10	7	19	23	11	1
9	7	19	23	11	8	20	24	12	1
10	9	21	25	13	10	22	26	14	1
11	10	22	26	14	11	23	27	15	1
12	11	23	27	15	12	24	28	16	1
13	17	29	33	33	18	30	34	34	1
14	18	30	34	34	19	31	35	35	1
15	19	31	35	35	20	32	36	36	1
16	17	33	37	21	18	34	38	22	1
17	18	34	38	22	19	35	39	23	1
18	19	35	39	23	20	36	40	24	1
19	21	37	41	25	22	38	42	26	1
20	22	38	42	26	23	39	43	27	1
21	23	39	43	27	24	40	44	28	1
22	29	45	49	33	30	46	50	34	1
23	30	46	50	34	31	47	51	35	1
24	31	47	51	35	32	48	52	36	1
25	33	49	53	37	34	50	54	38	1
26	34	50	54	38	35	51	55	39	1
27	35	51	55	39	36	52	56	40	1
28	37	53	57	41	38	54	58	42	1
29	38	54	58	42	39	55	59	43	1
30	39	55	59	43	40	56	60	44	1
31	45	61	65	49	46	62	66	50	1
32	46	62	66	50	47	63	67	51	1
33	47	63	67	51	48	64	68	52	1
34	49	65	69	53	50	66	70	54	1

35	50	66	70	54	51	67	71	55	1
36	51	67	71	55	52	68	72	56	1
37	53	69	73	57	54	70	74	58	1
38	54	70	74	58	55	71	75	59	1
39	55	71	75	59	56	72	76	60	1
40	61	77	81	65	62	78	82	66	1
41	62	78	82	66	63	79	83	67	1
42	63	79	83	67	64	80	84	68	1
43	65	81	85	69	66	82	86	70	1
44	66	82	86	70	67	83	87	71	1
45	67	83	87	71	68	84	88	72	1
46	69	85	89	73	70	86	90	74	1
47	70	86	90	74	71	87	91	75	1
48	71	87	91	75	72	88	92	76	1
49	77	93	97	81	78	94	98	82	1
50	78	94	98	82	79	95	99	83	1
51	79	95	99	83	80	96	100	84	1
52	81	97	101	85	82	98	102	86	1
53	82	98	102	86	83	99	103	87	1
54	83	99	103	87	84	100	104	88	1
55	85	101	105	89	86	102	106	90	1
56	86	102	106	90	87	103	107	91	1
57	87	103	107	91	88	104	108	92	1
58	93	109	113	97	94	110	114	98	2
59	94	110	114	98	95	111	115	99	2
60	95	111	115	99	96	112	116	100	2
61	97	113	117	101	98	114	118	102	1
62	98	114	118	102	99	115	119	103	1
63	99	115	119	103	100	116	120	104	1
64	101	117	121	105	102	118	122	106	1
65	102	118	122	106	103	119	123	107	1
66	103	119	123	107	104	120	124	108	1
67	109	125	129	113	110	126	130	114	2
68	110	126	130	114	111	127	131	115	2
69	111	127	131	115	112	128	132	116	2
70	113	129	133	117	114	130	134	118	1
71	114	130	134	118	115	131	135	119	1
72	115	131	135	119	116	132	136	120	1
73	117	133	137	121	118	134	138	122	1
74	118	134	138	122	119	135	139	123	1
75	119	135	139	123	120	136	140	124	1
76	125	141	145	129	126	142	146	130	1
77	126	142	146	130	127	143	147	131	1
78	127	143	147	131	128	144	148	132	2
79	129	145	149	133	130	146	150	134	1
80	130	146	150	134	131	147	151	135	1
81	131	147	151	135	132	148	152	136	2
82	133	149	153	137	134	150	154	138	1
83	134	150	154	138	135	151	155	139	1
84	135	151	155	139	136	152	156	140	2
85	141	157	161	145	142	158	162	146	1
86	142	158	162	146	143	159	163	147	1
87	143	159	163	147	144	160	164	148	1
88	145	161	165	149	146	162	166	150	1
89	146	162	166	150	147	163	167	151	1
90	147	163	167	151	148	164	168	152	1

91	149	165	169	153	150	166	170	154	1
92	150	166	170	154	151	167	171	155	1
93	151	167	171	155	152	168	172	156	1
94	157	173	177	161	158	174	178	162	1
95	158	174	178	162	159	175	179	163	1
96	159	175	179	163	160	176	180	164	1
97	161	177	181	165	162	178	182	166	1
98	162	178	182	166	163	179	183	167	1
99	163	179	183	167	164	180	184	168	1
100	165	181	185	169	166	182	186	170	1
101	166	182	186	170	167	183	187	171	1
102	167	183	187	171	168	184	188	172	1
103	173	189	193	177	174	190	194	178	1
104	174	190	194	178	175	191	195	179	1
105	175	191	195	179	176	192	196	180	1
106	177	193	197	181	178	194	198	182	1
107	178	194	198	182	179	195	199	183	1
108	179	195	199	183	180	196	200	184	1
109	181	197	201	185	182	198	202	186	1
110	182	198	202	186	183	199	203	187	1
111	183	199	203	187	184	200	204	188	1
112	185	205	209	193	190	206	210	194	1
113	190	206	210	194	191	207	211	195	1
114	191	207	211	195	192	208	212	196	1
115	193	209	213	197	194	210	214	198	1
116	194	210	214	198	195	211	215	199	1
117	195	211	215	199	196	212	216	200	1
118	197	213	217	201	198	214	218	202	1
119	198	214	218	202	199	215	219	203	1
120	199	215	219	203	200	216	220	204	1

\* \* \* MAXIMUM BANDWIDTH IS 22 \* \* \*

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

THERMAL PROPERTIES OF SYSTEM TO BE ANALYZED

THERE ARE 2 DIFFERENT MATERIALS

\*\*\*\*\*

• • • MATERIAL NUMBER 1 • • •

• • • CONDUCTIVITY • • •

NODE	TEMPERATURE	VALUE	SLOPE
1	0.	.900	
2	390.0	.900	C.
3	1650.0	.506	-.313E-03
4	3000.0	.506	0.

• • • SPECIFIC HEAT • • •

MATERIAL PARAMETER OF CONSTANT VALUE .272

• • • DENSITY • • •

MATERIAL PARAMETER OF CONSTANT VALUE 150.000

• • • MATERIAL NUMBER 2 • • •

• • • CONDUCTIVITY • • •

NODE	TEMPERATURE	VALUE	SLOPE
1	0.	30.000	
2	1100.0	19.900	-.918E-02
3	3000.0	19.900	C.

\* \* \* SPECIFIC HEAT \* \* \*

NODE	TEMPERATURE	VALUE	SLOPE
1	0.	.107	.493E-04
2	750.0	.144	.800E-04
3	1100.0	.172	C.
4	3000.0	.172	

\* \* \* DENSITY \* \* \*

MATERIAL PARAMETER OF CONSTANT VALUE 480.000

\*\*\*\*\*

FIREST-3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

NON-LINEAR FIRE BOUNDARY CONDITION

\*\*\*\*\*

C=A\*(TF-TS)\*\*N+SB\*V\*(AB\*EF\*(TF+TSHIFT)\*\*4-ES\*(TS+TSHIFT)\*\*4)

WHERE

TF - PSEUDO FIRE TEMPERATURE

TS - SURFACE TEMPERATURE

SB - STEFAN BOLTZMANN CONSTANT = .1700E-08

TSHIFT - SHIFT TO ABSOLUTE TEMPERATURE SCALE = 460.0

AND

MAT NUM	CONVECT FACTOR (A)	CONVECT POWER (N)	VIEW FACTOR (V)	ABSORBT (AB)	FIRE EMISSIV (FF)	SURFACE EMISSIV (ES)
1	.270	1.250	1.000	.900	.900	.900

\* \* \* THERE ARE 9 3-D SURFACE ELEMENTS EXPOSED TO FIRE \* \*

DESCRIPTION OF SURFACE DIRECTLY EXPOSED TO FIRE

FIRESURFACE	NODE I	NODE J	NODE K	NODE L	MAT TYPE	FIRE TYPE	AREA
1	205	209	210	206	1	1	.003
2	206	210	211	207	1	1	.003
3	207	211	212	208	1	1	.001
4	209	213	214	210	1	1	.004
5	210	214	215	211	1	1	.004
6	211	215	216	212	1	1	.002
7	213	217	218	214	1	1	.007
8	214	218	219	215	1	1	.007
9	215	219	220	216	1	1	.003

\*\*\*\*\*

FIREST-3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

INFORMATION RELEVANT TO THE ANALYSIS PROCEDURE

\*\*\*\*\*

\* \* \* \* CONVERGENCE CRITERIA \* \* \*

CONVERGENCE CRITERIA FOR BOUNDARY CONDITIONS

PERMISSIBLE ERROR = .00500

MAXIMUM NUMBER OF ITERATIONS = 30      BETA = -.2500

\* \* \* \* STORAGE REQUIREMENT FOR BLANK COMMON \* \* \*

SIZE BLANK COMMON = 8923 (DECIMAL)  
= 0021333 (OCTAL)

\*\*\*\*\*

FIREST-3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

INITIAL SEQUENCE NUMBER IS 0 AND INITIAL TIME IS 0.

\*\*\*\*\*

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.00	7	68.00	8	68.00
9	68.00	10	68.00	11	68.00	12	68.00
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.00	40	68.00
41	68.00	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.00	48	68.00
49	68.00	50	68.00	51	68.00	52	68.00
53	68.00	54	68.00	55	68.00	56	68.00
57	68.00	58	68.00	59	68.00	60	68.00
61	68.00	62	68.00	63	68.00	64	68.00
65	68.00	66	68.00	67	68.00	68	68.00
69	68.00	70	68.00	71	68.00	72	68.00
73	68.00	74	68.00	75	68.00	76	68.00
77	68.00	78	68.00	79	68.00	80	68.00
81	68.00	82	68.00	83	68.00	84	68.00
85	68.00	86	68.00	87	68.00	88	68.00
89	68.00	90	68.00	91	68.00	92	68.00
93	68.00	94	68.00	95	68.00	96	68.00
97	68.00	98	68.00	99	68.00	100	68.00
101	68.00	102	68.00	103	68.00	104	68.00
105	68.00	106	68.00	107	68.00	108	68.00
109	68.00	110	68.00	111	68.00	112	68.00
113	68.00	114	68.00	115	68.00	116	68.00
117	68.00	118	68.00	119	68.00	120	68.00
121	68.00	122	68.00	123	68.00	124	68.00
125	68.00	126	68.00	127	68.00	128	68.00
129	68.00	130	68.00	131	68.00	132	68.00
133	68.00	134	68.00	135	68.00	136	68.00
137	68.00	138	68.00	139	68.00	140	68.00
141	68.00	142	68.00	143	68.00	144	68.00
145	68.00	146	68.00	147	68.00	148	68.00
149	68.00	150	68.00	151	68.00	152	68.00
153	68.00	154	68.00	155	68.00	156	68.00
157	68.00	158	68.00	159	68.00	160	68.00

161	68.00	162	68.00	163	68.00	164	68.00
165	68.00	166	68.00	167	68.00	168	68.00
169	68.00	170	68.00	171	68.00	172	68.00
173	68.00	174	68.00	175	68.00	176	68.00
177	68.00	178	68.00	179	68.00	180	68.00
181	68.00	182	68.00	183	68.00	184	68.00
185	68.00	186	68.00	187	68.00	188	68.00
189	68.00	190	68.00	191	68.00	192	68.00
193	68.00	194	68.00	195	68.00	196	68.00
197	68.00	198	68.00	199	68.00	200	68.00
201	68.00	202	68.00	203	68.00	204	68.00
205	68.00	206	68.00	207	68.00	208	68.00
209	68.00	210	68.00	211	68.00	212	68.00
213	68.00	214	68.00	215	68.00	216	68.00
217	68.00	218	68.00	219	68.00	220	68.00

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

TIME STEP NUMBER 1 - TIME .025 - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS - 0

FIRE BOUNDARY CONDITION

FIRE(1) = 347.600  
FIRE(2) = -0.  
FIRE(3) = -0.  
FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.00	7	68.00	8	68.00
9	68.00	10	68.00	11	68.00	12	68.00
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.00	40	68.00
41	68.00	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.00	48	68.00
49	68.00	50	68.00	51	68.00	52	68.00
53	68.00	54	68.00	55	68.00	56	68.00
57	68.00	58	68.00	59	68.00	60	68.00
61	68.00	62	68.00	63	68.00	64	68.00
65	68.00	66	68.00	67	68.00	68	68.00
69	68.00	70	68.00	71	68.00	72	68.00
73	68.00	74	68.00	75	68.00	76	68.00
77	68.00	78	68.00	79	68.00	80	68.00
81	68.00	82	68.00	83	68.00	84	68.00
85	68.00	86	68.00	87	68.00	88	68.00
89	68.00	90	68.00	91	68.00	92	68.00
93	68.02	94	68.02	95	68.02	96	68.02
97	68.02	98	68.02	99	68.02	100	68.02
101	68.00	102	68.00	103	68.00	104	68.00
105	68.00	106	68.00	107	68.00	108	68.00
109	68.02	110	68.02	111	68.02	112	68.02
113	68.02	114	68.02	115	68.03	116	68.03
117	68.01	118	68.01	119	68.01	120	68.01
121	68.01	122	68.01	123	68.01	124	68.01
125	68.02	126	68.02	127	68.04	128	68.04

129	68.03	130	68.03	131	68.04	132	68.05
133	68.04	134	68.03	135	68.07	136	68.07
137	68.04	138	68.03	139	68.07	140	68.08
141	68.13	142	68.13	143	68.07	144	68.06
145	68.13	146	68.13	147	68.07	148	68.06
149	68.13	150	68.13	151	68.09	152	68.08
153	68.13	154	68.14	155	68.09	156	68.09
157	68.62	158	68.61	159	68.61	160	68.61
161	68.62	162	68.61	163	68.61	164	68.61
165	68.62	166	68.62	167	68.61	168	68.61
169	68.62	170	68.62	171	68.62	172	68.62
173	70.96	174	70.96	175	70.96	176	70.96
177	70.97	178	70.97	179	70.97	180	70.97
181	70.98	182	70.98	183	70.98	184	70.98
185	70.98	186	70.98	187	70.98	188	70.98
189	75.49	190	75.49	191	75.49	192	75.49
193	75.51	194	75.51	195	75.51	196	75.51
197	75.52	198	75.52	199	75.52	200	75.52
201	75.52	202	75.52	203	75.52	204	75.52
205	85.73	206	85.73	207	85.73	208	85.73
209	85.74	210	85.74	211	85.74	212	85.74
213	85.74	214	85.74	215	85.74	216	85.74
217	85.75	218	85.75	219	85.75	220	85.75

----- TEMPERATURE OF 3-D ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.00	7	68.00	8	68.00
9	68.00	10	68.00	11	68.00	12	68.00
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.00	40	68.00
41	68.00	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.00	48	68.00
49	68.01	50	68.01	51	68.01	52	68.01
53	68.01	54	68.01	55	68.00	56	68.00
57	68.00	58	68.02	59	68.02	60	68.02
61	68.01	62	68.01	63	68.02	64	68.01
65	68.01	66	68.01	67	68.02	68	68.03
69	68.03	70	68.02	71	68.03	72	68.04
73	68.02	74	68.03	75	68.04	76	68.08
77	68.07	78	68.05	79	68.08	80	68.07
81	68.07	82	68.09	83	68.08	84	68.08
85	68.37	86	68.36	87	68.34	88	68.38
89	68.36	90	68.34	91	68.38	92	68.36
93	68.35	94	69.79	95	69.79	96	69.79
97	69.80	98	69.79	99	69.79	100	69.80
101	69.80	102	69.80	103	73.23	104	73.23
105	73.23	106	73.24	107	73.24	108	73.24

109	73.25	110	73.25	111	73.25	112	80.62
113	80.62	114	80.62	115	80.63	116	80.63
117	80.63	118	80.63	119	80.63	120	80.63

0 SYSTEM ITERATIONS WERE PERFORMED  
3 E. C. ITERATIONS WERE PERFORMED

\*\*\*\*\*

FIRE-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL

\*\* CIRCULAR COLUMN WITH 1/2 INCH SPIRAL - 3-D ANALYSIS \*\*

TIME STEP NUMBER 2 - TIME .050 - TIME STEP .025

\*\*\*\*\*

NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS -C

FIRE BOUNDARY CONDITION

FIRE(1) = 627.200  
FIRE(2) = -0.  
FIRE(3) = -0.  
FIRE(4) = -0.

----- NODAL POINT TEMPERATURES -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.00	7	68.00	8	68.00
9	68.00	10	68.00	11	68.00	12	68.00
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.00	40	68.00
41	68.00	42	68.00	43	68.00	44	68.00
45	68.00	46	68.00	47	68.00	48	68.00
49	68.00	50	68.00	51	68.00	52	68.00
53	68.00	54	68.00	55	68.00	56	68.00
57	68.00	58	68.00	59	68.00	60	68.00
61	68.00	62	68.00	63	68.00	64	68.00
65	68.00	66	68.00	67	68.00	68	68.00
69	68.00	70	68.00	71	68.00	72	68.00
73	68.00	74	68.00	75	68.00	76	68.00
77	68.02	78	68.02	79	68.02	80	68.02
81	68.01	82	68.01	83	68.02	84	68.02
85	68.01	86	68.01	87	68.01	88	68.01
89	68.00	90	68.00	91	68.00	92	68.00
93	68.12	94	68.13	95	68.14	96	68.14
97	68.11	98	68.12	99	68.13	100	68.13
101	68.02	102	68.02	103	68.02	104	68.02
105	68.01	106	68.02	107	68.02	108	68.02
109	68.14	110	68.15	111	68.17	112	68.17
113	68.13	114	68.14	115	68.16	116	68.16
117	68.06	118	68.07	119	68.08	120	68.09
121	68.05	122	68.07	123	68.09	124	68.09
125	68.15	126	68.16	127	68.23	128	68.25

129	68.17	130	68.17	131	68.27	132	68.30
133	68.24	134	68.22	135	68.42	136	68.44
137	68.24	138	68.22	139	68.44	140	68.47
141	68.78	142	68.78	143	68.39	144	68.35
145	68.78	146	68.79	147	68.42	148	68.38
149	68.80	150	68.80	151	68.51	152	68.48
153	68.80	154	68.81	155	68.55	156	68.52
157	71.30	158	71.28	159	71.24	160	71.23
161	71.30	162	71.28	163	71.25	164	71.24
165	71.32	166	71.30	167	71.27	168	71.26
169	71.32	170	71.31	171	71.29	172	71.28
173	81.86	174	81.85	175	81.85	176	81.85
177	81.88	178	81.87	179	81.87	180	81.87
181	81.93	182	81.93	183	81.92	184	81.92
185	81.93	186	81.93	187	81.92	188	81.92
189	99.46	190	99.46	191	99.45	192	99.45
193	99.54	194	99.54	195	99.54	196	99.54
197	99.55	198	99.55	199	99.55	200	99.55
201	99.55	202	99.55	203	99.54	204	99.54
205	135.23	206	135.23	207	135.22	208	135.22
209	135.28	210	135.28	211	135.28	212	135.28
213	135.27	214	135.27	215	135.27	216	135.27
217	135.28	218	135.28	219	135.28	220	135.28

----- TEMPERATURE OF 3-C ELEMENTS -----

N	TEMP.	N	TEMP.	N	TEMP.	N	TEMP.
1	68.00	2	68.00	3	68.00	4	68.00
5	68.00	6	68.00	7	68.00	8	68.00
9	68.00	10	68.00	11	68.00	12	68.00
13	68.00	14	68.00	15	68.00	16	68.00
17	68.00	18	68.00	19	68.00	20	68.00
21	68.00	22	68.00	23	68.00	24	68.00
25	68.00	26	68.00	27	68.00	28	68.00
29	68.00	30	68.00	31	68.00	32	68.00
33	68.00	34	68.00	35	68.00	36	68.00
37	68.00	38	68.00	39	68.00	40	68.01
41	68.01	42	68.01	43	68.01	44	68.01
45	68.01	46	68.00	47	68.00	48	68.00
49	68.07	50	68.07	51	68.08	52	68.04
53	68.04	54	68.04	55	68.01	56	68.01
57	68.01	58	68.13	59	68.14	60	68.15
61	68.08	62	68.09	63	68.10	64	68.04
65	68.05	66	68.05	67	68.15	68	68.1F
69	68.22	70	68.15	71	68.19	72	68.24
73	68.15	74	68.20	75	68.27	76	68.47
77	68.40	78	68.32	79	68.50	80	68.45
81	68.40	82	68.52	83	68.50	84	68.48
85	70.03	86	69.93	87	69.81	88	70.04
89	69.95	90	69.85	91	70.06	92	69.98
93	69.89	94	76.58	95	76.56	96	76.55
97	76.60	98	76.59	99	76.58	100	76.62
101	76.61	102	76.60	103	90.68	104	90.68
105	90.68	106	90.72	107	90.72	108	90.72

109	90.74	110	90.74	111	90.73	112	117.37
113	117.37	114	117.37	115	117.41	116	117.41
117	117.41	118	117.41	119	117.41	120	117.41

0 SYSTEM ITERATIONS WERE PERFORMED  
3 E. C. ITERATIONS WERE PERFORMED

PROBLEM COMPLETED

APPENDIX C

LISTING OF COMPUTER PROGRAM FIRES-T3

The following listing is the operational version of FIRES-T3 as of September 1977. Although the program has been tested, no warranty is made regarding its accuracy or reliability, and no responsibility is assumed in this respect.



```

FIRE5-T3      1      PROGRAM FIREST3(INPUT,OUTPUT,PUNCH,TAPE1=INPUT,TAPE2=OUTPUT,TAPE3=
FIRE5-T3      2      IPUNCH,TAPE6)
FIRE5-T3      3
FIRE5-T3      4
FIRE5-T3      5
FIRE5-T3      6
FIRE5-T3      7
FIRE5-T3      8
FIRE5-T3      9
FIRE5-T3     10
FIRE5-T3     11
FIRE5-T3     12
FIRE5-T3     13
FIRE5-T3     14
FIRE5-T3     15
FIRE5-T3     16
FIRE5-T3     17
FIRE5-T3     18
FIRE5-T3     19
FIRE5-T3     20
FIRE5-T3     21
FIRE5-T3     22
FIRE5-T3     23
FIRE5-T3     24
FIRE5-T3     25
FIRE5-T3     26
FIRE5-T3     27
FIRE5-T3     28
FIRE5-T3     29
FIRE5-T3     30
FIRE5-T3     31
FIRE5-T3     32
FIRE5-T3     33
FIRE5-T3     34
FIRE5-T3     35
FIRE5-T3     36
FIRE5-T3     37
FIRE5-T3     38
FIRE5-T3     39
FIRE5-T3     40
FIRE5-T3     41
FIRE5-T3     42
FIRE5-T3     43
FIRE5-T3     44
FIRE5-T3     45
FIRE5-T3     46
FIRE5-T3     47
FIRE5-T3     48
FIRE5-T3     49
FIRE5-T3     50
FIRE5-T3     51
FIRE5-T3     52
FIRE5-T3     53
FIRE5-T3     54
FIRE5-T3     55
FIRE5-T3     56
FIRE5-T3     57
FIRE5-T3     58
FIRE5-T3     59
FIRE5-T3     60
FIRE5-T3     61
FIRE5-T3     62

C ***** ****
C *FIRE5-T3* IS A PROGRAM FOR TRANSIENT, NONLINEAR, CNE-, TNC-, *
C AND 3-DIMENSIONAL HEAT FLOW PROBLEMS IN WHICH THERMAL PROPER-
C TIES OF THE MATERIAL CAN BE FUNCTIONS OF TEMPERATURE. A FIRE *
C ENVIRONMENT CAN BE SIMULATED WITH OPTIONAL LINEAR AND NONLINEAR *
C BOUNDARY CONDITIONS. THE ANALYTICAL APPROACH USED IS THAT OF A *
C FINITE ELEMENT TECHNIQUE COUPLED WITH A TIME STEP INTEGRATION. *
C THE PROGRAM IS DYNAMICALLY DIMENSIONED TO ALLOW FOR MINIMUM *
C CENTRAL MEMORY STORAGE.
C
C ORIGINAL VERSION          AUGUST 1977
C
C PROGRAMMED BY R. EDING AND E. BRESLER
C
C UNIVERSITY OF CALIFORNIA      BERKELEY
C ***** ****
C
COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPUNCH,NUPNP,NELID,N
IEL2D,NEL3D,NUMEL,MRAND,NMAT,NFBC1D,NFBC2D,NFBC3D,NBCMAT,NRCTYP
COMMON /EXOTH/ NINTID,NINT2D,NINT3D,NINT,NINCT
COMMON C(10000)

C
NIN=1
NOUT=2
NPUNCH=3
NALLOW=10000

C INPUT TITLE
10 READ (NIN,100) ITITLE
C INPUT OF PROBLEM DESCRIPTION DONE ON BASIS OF ALPHA-NUMERIC
C CONTROL CARDS IN WHICH THE FIRST LETTER OF THE CONTROL
C WORD IS CHECKED FOR PROPER INPUT SEQUENCE
C INPUT CONTROL CARD * NODES *
C
READ (NIN,160) IREAD
IF (IREAD(1).EQ.55B) STOP
IF (IREAD(1).NE.16B) GO TO 90
N=1
C OUTPUT TITLE PAGE
51 C
52 C
53 C
54 C
55 C
56 C
57 C
58 C
59 C
60 C
61 C
62 C
WRITE (NOUT,110)
WRITE (NOUT,120)
WRITE (NOUT,130)
WHITE (NOUT,140)
WHITE (NOUT,120)
WHITE (NOUT,150) ITITLE
WHITE (NOUT,120)

C INPUT OF NODAL DATA

```

```

FIRES-T3 63      C      NUMNP - NUMBER OF NODAL POINTS
FIRES-T3 64      C      NTBC - NUMBER OF KNOWN TEMPERATURE BOUNDARY CONDITIONS
FIRES-T3 65      C      X(1)=C(N1) - X COORDINATE OF NODAL POINT
FIRES-T3 66      C      Y(1)=C(N2) - Y COORDINATE OF NODAL POINT
FIRES-T3 67      C      Z(1)=C(N3) - Z COORDINATE OF NODAL POINT
FIRES-T3 68      C      KODE(I)=C(NDO) - INDICATES TYPE OF BOUNDARY CONDITION
FIRES-T3 69      C      FLOW - HEAT FLOW IS KNOWN
FIRES-T3 70      C      TEMP - TEMPERATURE IS KNOWN
FIRES-T3 71      C      DUMT(I)=C(NDI) - DUMMY VARIABLES REQUIRED IN CALCULATION
FIRES-T3 72      C      I VARIES FROM 1 TO 5
FIRES-T3 73      C

FIRES-T3 74      C      NUMNP=NUMBER(N)
FIRES-T3 75      C      IF (NUMNP.LE.0) GO TO 90
FIRES-T3 76      C      NTBC=NUMBER(N)
FIRES-T3 77      C      IF (NTBC.GT.NUMNP) GO TO 90
N1=1
N2=N1+NUMNP
N3=N2+NUMNP
NDO=N3+NUMNP
ND1=NDO+NUMNP
ND2=ND1+NUMNP
ND3=ND2+NUMNP
ND4=ND3+NUMNP
ND5=ND4+NUMNP
N4=ND5+NUMNP
N5=N4+NUMNP
N6=N5+NUMNP
N7=N6+NUMNP
N8=N7+NUMNP
N9=N8+NUMNP
N10=N9+NUMNP
N11=N10+NUMNP
N12=N11+NUMNP
N13=N12+NUMNP
N14=N13+NUMNP
N15=N14+NUMNP
N16=N15+NUMNP
N17=N16+NUMNP
N18=N17+NUMNP
N19=N18+NUMNP
N20=N19+NUMNP
N21=N20+NUMNP
N22=N21+NUMNP
N23=N22+NUMNP
N24=N23+NUMNP
N25=N24+NUMNP
N26=N25+NUMNP
N27=N26+NUMNP
N28=N27+NUMNP
N29=N28+NUMNP
N30=N29+NUMNP
N31=N30+NUMNP
N32=N31+NUMNP
N33=N32+NUMNP
N34=N33+NUMNP
N35=N34+NUMNP
N36=N35+NUMNP
N37=N36+NUMNP
N38=N37+NUMNP
N39=N38+NUMNP
N40=N39+NUMNP
N41=N40+NUMNP
N42=N41+NUMNP
N43=N42+NUMNP
N44=N43+NUMNP
N45=N44+NUMNP
N46=N45+NUMNP
N47=N46+NUMNP
N48=N47+NUMNP
N49=N48+NUMNP
N50=N49+NUMNP
N51=N50+NUMNP
N52=N51+NUMNP
N53=N52+NUMNP
N54=N53+NUMNP
N55=N54+NUMNP
N56=N55+NUMNP
N57=N56+NUMNP
N58=N57+NUMNP
N59=N58+NUMNP
N60=N59+NUMNP
N61=N60+NUMNP
N62=N61+NUMNP
N63=N62+NUMNP
N64=N63+NUMNP
N65=N64+NUMNP
N66=N65+NUMNP
N67=N66+NUMNP
N68=N67+NUMNP
N69=N68+NUMNP
N70=N69+NUMNP
N71=N70+NUMNP
N72=N71+NUMNP
N73=N72+NUMNP
N74=N73+NUMNP
N75=N74+NUMNP
N76=N75+NUMNP
N77=N76+NUMNP
N78=N77+NUMNP
N79=N78+NUMNP
N80=N79+NUMNP
N81=N80+NUMNP
N82=N81+NUMNP
N83=N82+NUMNP
N84=N83+NUMNP
N85=N84+NUMNP
N86=N85+NUMNP
N87=N86+NUMNP
N88=N87+NUMNP
N89=N88+NUMNP
N90=N89+NUMNP
N91=N90+NUMNP
N92=N91+NUMNP
N93=N92+NUMNP
N94=N93+NUMNP
N95=N94+NUMNP
N96=N95+NUMNP
N97=N96+NUMNP
N98=N97+NUMNP
N99=N98+NUMNP
N100=N99+NUMNP
N101=N100+NUMNP
N102=N101+NUMNP
N103=N102+NUMNP
N104=N103+NUMNP
N105=N104+NUMNP
N106=N105+NUMNP
N107=N106+NUMNP
N108=N107+NUMNP
N109=N108+NUMNP
N110=N109+NUMNP
N111=N110+NUMNP
N112=N111+NUMNP
N113=N112+NUMNP
N114=N113+NUMNP
N115=N114+NUMNP
N116=N115+NUMNP
N117=N116+NUMNP
N118=N117+NUMNP
N119=N118+NUMNP
N120=N119+NUMNP
N121=N120+NUMNP
N122=N121+NUMNP
N123=N122+NUMNP
N124=N123+NUMNP

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FIRES-T3 125
FIRES-T3 126
FIRES-T3 127
FIRES-T3 128
FIRES-T3 129
FIRES-T3 130
FIRES-T3 131
FIRES-T3 132
FIRES-T3 133
FIRES-T3 134
FIRES-T3 135
FIRES-T3 136
FIRES-T3 137
FIRES-T3 138
FIRES-T3 139
FIRES-T3 140
FIRES-T3 141
FIRES-T3 142
FIRES-T3 143
FIRES-T3 144
FIRES-T3 145
FIRES-T3 146
FIRES-T3 147
FIRES-T3 148
FIRES-T3 149
FIRES-T3 150
FIRES-T3 151
FIRES-T3 152
FIRES-T3 153
FIRES-T3 154
FIRES-T3 155
FIRES-T3 156
FIRES-T3 157
FIRES-T3 158
FIRES-T3 159
FIRES-T3 160
FIRES-T3 161
FIRES-T3 162
FIRES-T3 163
FIRES-T3 164
FIRES-T3 165
FIRES-T3 166
FIRES-T3 167
FIRES-T3 168
FIRES-T3 169
FIRES-T3 170
FIRES-T3 171
FIRES-T3 172
FIRES-T3 173
FIRES-T3 174
FIRES-T3 175
FIRES-T3 176
FIRES-T3 177
FIRES-T3 178
FIRES-T3 179
FIRES-T3 180
FIRES-T3 181
FIRES-T3 182
FIRES-T3 183
FIRES-T3 184
FIRES-T3 185
FIRES-T3 186

      READ (NIN,160) IREAD
      IF (IREAD(1).NE.+160) GO TO 90
      N=1
      C
      C INPUT OF THERMAL PROPERTIES FOR DIFFERENT MATERIALS
      C
      C NMAT = NUMBER OF DIFFERENT MATERIALS
      C MATL(1)=C(NB) - CONTROL DATA REQUIRED FOR CALCULATION OF
      C THERMAL PROPERTIES
      C XYS(1)=C(N9) - FUNCTION VALUES FOR THERMAL PROPERTIES
      C CONTAINS X COORDINATE = TEMPERATURE
      C Y COORDINATE = FUNCTION VALUE
      C S = SLOPE OF LINES CONNECTING X,Y PAIR
      C
      NMAT=NUMBER(N)
      IF (NMAT.EQ.0) GO TO 90
      NB=N+6#NMAT
      C
      CALL MATERIAL (C(NB),C(N9),N)
      NIO=N9+N
      C
      C INPUT CONTROL CARD * FIRE * 
      C
      READ (NIN,160) IREAD
      N=1
      IF (IREAD(1).NE.+6) GO TO 90
      C
      C INPUT OF FIRE BOUNDARY CONDITION DATA
      C
      C NEBC10 = NUMBER OF FIRE B.C. SURFACE NODES (FOR 1-D ELEMENTS)
      C NEBC20 = NUMBER OF FIRE B.C. SURFACE ELMTS (FOR 2-D ELEMENTS)
      C NEBC30 = NUMBER OF FIRE B.C. SURFACE ELMTS (FOR 3-D ELEMENTS)
      C { THESE ARE MAXIMUMS IN ORDER TO ALLOT STORAGE }
      C NMCMAT = NUMBER OF DIFFERENT MATERIALS FOR FIRE B.C.
      C NBCTYP = TYPE OF FIRE B.C.
      C MATL(1)=C(NIO) - CONTROL DATA REQUIRED FOR CALCULATION OF FIRE
      C BOUNDARY CONDITION PROPERTIES
      C XYSC(1)=C(N11) - CONTAINS FUNCTIONS OF FIRE B.C. PROPERTIES
      C STRUCTURED SAME AS XYS
      C
      NEBC10=NUMBER(N)
      NEBC20=NUMBER(N)
      NEBC30=NUMBER(N)
      NMCMAT=NEBC10+NEBC20+NEBC30
      IF (NMCMAT.EQ.0) GO TO 40
      NMCMAT=NUMBER(N)
      READ (NIN,160) IREAD
      IF (IREAD(1).NE.+160) GO TO 20
      C
      C LINEAR FIRE BOUNDARY CONDITION
      C
      C D = HCT*(TF-TS) WHERE
      C HCT = SURFACE HEAT TRANSFER COEFFICIENT
      C TF = PSUEUDO-FIRE TEMPERATURE (INPUT DATA)
      C TS = SURFACE TEMPERATURE OF SYSTEM
      C T=(TF+TS)/2
      C
      NBCTYP=LINEAR
      NIO=NEBC30+NMCMAT
      GO TO 50
      C
      C D = TF - (TF-ACCL*TS)/(1+ACCL) GO TO 50

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FIRE5-T3 187
FIRE5-T3 188
FIRE5-T3 189
FIRE5-T3 190
FIRE5-T3 191
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FIRE5-T3 244
FIRE5-T3 245
FIRE5-T3 246
FIRE5-T3 247
FIRE5-T3 248

C      NON-LINEAR FIRE BOUNDARY CONDITION
C
C      O = A*(T-F-TS)**N + V*SB*(AB*EFF*ATE**4-ES*ATS**4)
C
C      WHERE   A = CONVECTION FACTOR
C              N = POWER OF CONVECTION TERM
C              V = VIEW FACTOR FOR RADIATION TERM
C              SB = STEFANN-BOLTZMANN CONSTANT
C              AB = ABSORPTION FACTOR OF FIRE
C              EFF = EMISSIVITY OF FIRE
C              TS = EMISSIVITY OF SURFACE
C              ATE = ABSOLUTE TEMPERATURE OF SURFACE OF FIRE
C              ATS = ABSOLUTE TEMPERATURE OF SURFACE OF SYSTEM
C
C      NBCTYP=10HNN-NLN BC
C      NI1=N10+NBCTAT
C
C      30 CONTINUE
C
C      CALL FIREMAT (C(N10),C(N11),N)
C      N12=N11+N
C
C      INPUT OF GEOMETRIC DESCRIPTION OF FIRE BOUNDARY
C
C      L(1)=C(N12) - FIRST NODE THAT BOUNDS FIRE SURFACE ELEMENT
C                  ( 1-D, 2-D, AND 3-D ELEMENTS )
C      L(2)=C(N13) - SECOND NODE THAT BOUNDS FIRE SURFACE ELEMENT
C                  ( 2-D AND 3-D ELEMENTS ONLY )
C      L(3)=C(N14) - THIRD NODE THAT BOUNDS FIRE SURFACE ELEMENT
C                  ( 3-D ELEMENTS ONLY )
C      L(4)=C(N15) - FOURTH NODE THAT BOUNDS FIRE SURFACE ELEMENT
C                  ( 1-D ELEMENTS ONLY )
C      LMAT(1)=C(N16) - FIRE BOUNDARY CONDITION TYPE ( MATERIAL )
C      LFIRE(1)=C(N17) - NUMBER OF FIRE ACTIVE ON SURFACE I, I-J,
C                      OR I-J-K-L
C      ATJKL(1)=C(N18) - AREA OF SURFACE I, I-J, OR I-J-K-L
C      LELEM(1)=C(N19) - NUMBER OF FINITE ELEMENT ADJACENT TO THE
C                      FIRE SURFACE ( 1-D AND 2-D ONLY )
C
C      NI3=N12+NUMFBC
C      NI4=N13+NUMFBC2+NUMFBC3D
C      NI5=N14+NUMFBC3D
C      NI6=N15+NUMFBC3D
C      NI7=N16+NUMFBC
C      NI8=N17+NUMFBC
C      NI9=N18+NUMFBC
C      NI0=N19+NUMFBC1D+NUMFBC2D
C
C      CALL FIREBC (C(N1),C(N2),C(N3),C(ND0),C(N6),C(N7),C(N8),C(N9))
C      GO TO 50
C
C      40 CONTINUE
C
C      WHEN NO FIRE B.C. IS SPECIFIED, INITIAIZE NI1 = N20
C
C      NI1=NI0
C      NI2=NI0
C      NI3=NI0
C      NI4=NI0
C      NI5=NI0

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FIRE5-T3 249      N16=N10
FIRE5-T3 250      N17=N10
FIRE5-T3 251      N18=N10
FIRE5-T3 252      N19=N10
FIRE5-T3 253      N20=N10
FIRE5-T3 254      C INPUT CONTROL CARD * EXOTHERMIC *
FIRE5-T3 255      C
FIRE5-T3 256      C
FIRE5-T3 257      C 50 READ (NIN,160) IREAD
FIRE5-T3 258      C  IF (IREAD(1).NE.58) GO TO 90
FIRE5-T3 259      C  N=1
FIRE5-T3 260      C
FIRE5-T3 261      C INPUT OF DATA DESCRIBING INTERNAL EXOTHERMIC HEAT GENERATION
FIRE5-T3 262      C
FIRE5-T3 263      C  NINTD - NUMBER OF 1-D ELEMENTS WITH INTERNAL HEAT GENERATION
FIRE5-T3 264      C  NINT2D - NUMBER OF 2-D ELEMENTS WITH INTERNAL HEAT GENERATION
FIRE5-T3 265      C  NINT3D - NUMBER OF 3-D ELEMENTS WITH INTERNAL HEAT GENERATION
FIRE5-T3 266      C  NINT - TOTAL NUMBER OF ELEMENTS WITH HEAT GENERATION
FIRE5-T3 267      C  NQINT - NUMBER OF DIFFERENT HEATING CURVES
FIRE5-T3 268      C  IEXC(1)=C(N20) - CONTROL DATA FOR HEATING CURVES
FIRE5-T3 269      C  EXYS(1)=C(N21) - CONTAINS FUNCTION VALUES FOR HEATING CURVE
FIRE5-T3 270      C          X-COORDINATE - TIME
FIRE5-T3 271      C          Y-COORDINATE - HEAT RATE
FIRE5-T3 272      C          S - SLOPE OF LINES CONNECTING X,Y PAIRS
FIRE5-T3 273      C  TEL(1)=C(N22) - ELEMENT NUMBER OF EACH ELEMENT UNDERGOING
FIRE5-T3 274      C          EXOTHERMIC HEATING
FIRE5-T3 275      C  THAT(1)=C(N23) - HEATING CURVE OF EACH ELEMENT UNDERGOING
FIRE5-T3 276      C          EXOTHERMIC HEATING
FIRE5-T3 277      C  VEL(1)=C(N24) - VOLUME OF EACH ELEMENT
FIRE5-T3 278      C
FIRE5-T3 279      C  NINT1D=NUMBER(N)
FIRE5-T3 280      C  NINT2D=NUMBER(N)
FIRE5-T3 281      C  NINT3D=NUMBER(N)
FIRE5-T3 282      C  NINT=NINT1D+NINT2D+NINT3D
FIRE5-T3 283      C  IF (NINT.NE.0) GO TO 60
FIRE5-T3 284      C  N21=N20
FIRE5-T3 285      C  N22=N20
FIRE5-T3 286      C  N23=N20
FIRE5-T3 287      C  N24=N20
FIRE5-T3 288      C  N25=N20
FIRE5-T3 289      C  GO TO 70
FIRE5-T3 290      C 60 NQINT=NUMBER(N)
FIRE5-T3 291      C  N21=N20+NQINT*3
FIRE5-T3 292      C  CALL EXOFUN (C(N20),C(N21),N)
FIRE5-T3 293      C  N22=N21+N
FIRE5-T3 294      C  N23=N22+NINT
FIRE5-T3 295      C  N24=N23+NINT
FIRE5-T3 296      C  N25=N24+NINT
FIRE5-T3 297      C  CALL EXOELS (C(N1),C(N2),C(N3),C(N4),C(N5),C(N7),C(NV),C(N22),C(N2
FIRE5-T3 298      C  13),C(N24))
FIRE5-T3 299      C
FIRE5-T3 300      C  ESTABLISH ADDITIONAL DYNAMICALLY DIMENSIONED VARIABLES REQUIRED
FIRE5-T3 301      C  IN THE HEAT FLOW ANALYSIS
FIRE5-T3 302      C
FIRE5-T3 303      C  Q(1)=C(N25) - FLOW VECTOR
FIRE5-T3 304      C  T(1)=C(N26) - TEMPERATURE VECTOR
FIRE5-T3 305      C  H(1)=C(N27) - HEAT LOAD (FLOW) VECTOR USED IN ANALYSIS
FIRE5-T3 306      C  AT(1,1)=C(N28) - ELEMENT TEMPERATURES
FIRE5-T3 307      C  A(1,1)=C(N29) - MODIFIED CONDUCTIVITY MATRIX
FIRE5-T3 308      C  NTOTAL - TOTAL AMOUNT OF STORAGE REQUIRED FOR BLANK COMMON
FIRE5-T3 309      C
FIRE5-T3 310      C 70 N26=N25+NUMNP

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FIRES-T3 311      N27=N26+NUMNP
FIRES-T3 312      N28=N27+NUMNP
FIRES-T3 313      N29=N28+NUMEL
FIRES-T3 314      NTOTAL=N29+NUMNF*MBAND
FIRES-T3 315      IF (NTOTAL.LE.NALLOW) GO TO 80
FIRES-T3 316      WRITE (INOUT,180) NTOTAL
FIRES-T3 317      STOP
C
C      INPUT CONTROL CARD * CONVERGENCE *
C
C      80 READ (ININ,160) IREAD
C      IF (IREAD(1).NE.38) GO TO 90
C
C      INPUT CONVERGENCE CONTROL DATA FOR ITERATIVE PROCESS
C
C      CALL CONVERG (NTOTAL)
C
C      TRANSFER CONTROL OF PROGRAM TO HEAT FLOW ANALYSIS ROUTINE
C      IN ORDER TO CARRY OUT STEP-BY-STEP TIME INTEGRATION
C
C      CALL FEATFLD (C(N1),C(N2),C(N3),C(ND0),C(ND1),C(ND2),C(ND3),C(ND4),
C      1,C(ND5),C(N4),C(N5),C(N6),C(N7),C(N8),C(N9),C(N10),C(N11),C(N12),C(
C      2(N13),C(N14),C(N15),C(N16),C(N17),C(N18),C(N19),C(N20),C(N21),C(
C      32),C(N23),C(N24),C(N25),C(N26),C(N27),C(N28),C(N29),C(NV),NUMNP,NU
C      4MFBC)
C      GO TO 10
C
C      90 CONTINUE
C
C      ERROR WAS DETECTED IN CONTROL WORD OR CONTROL VARIABLE
C      HAS IMPROPER VALUE AND PROGRAM IS TERMINATED
C
C      WRITE (INOUT,170) IREAD
C      STOP
C
C
C      100 FORMAT (6A10)
C      110 FORMAT (1H6,13(/))
C      120 FORMAT 16H ****
C      130 FORMAT (//,5X,50HFFFFF I RRRR EEEEE SSSSS TTTTT 333
C      133/5X,50HF   I R R E   S   T   3 /5X,50H
C      134   I R   R E   S   T   3 /5X,50HF   I
C      135   3 R   R E   S   T   3 /5X,50HFFF   I RRRR E
C      136   4FE   SSSSS ===   T   3333 /5X,50HF   I RR   E
C      137   5 S   T   3/5X,50HF   I RR   E   S
C      138   6   T   3/5X,50HF   I R R   E   S   T
C      139   7   3/5X,50HF   I R R   EEEE SSSSS   T   33333///
C      140 FORMAT (7X49HA THERMAL ANALYZER FOR THREE-DIMENSIONAL SYSTEMS,/7X,
C      1,45HWITH TEMPERATURE-DEPENDENT THERMAL PROPERTIES,/7X,31HSUBJCTE
C      20 TO A FIRE ENVIRONMENT,///)
C      150 FORMAT (//18X,24H-- TITLE OF RUN -- -,//1X,6A10//)
C      160 FORMAT (80R1)
C      170 FORMAT (6(1),45H-- PROGRAM TERMINATED - INPUT ERROR -- -,//1X
C      180R1)
C      180 FORMAT (//77H STOP - INCREASE BLANK COMMON SIZE TO,110)
C      END

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FIRES-T3 369
FIRES-T3 370
FIRES-T3 371
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FIRES-T3 375
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FIRES-T3 400
FIRES-T3 401
FIRES-T3 402
FIRES-T3 403
FIRES-T3 404
FIRES-T3 405

        FUNCTION NUMBER (1)

C
C
C      FUNCTION *NUMBER* OBTAINS AN INTEGER CONSTANT CONTAINED
C      ON AN ALPHA-NUMERIC CONTROL CARD. THE INTEGER CONSTANT MUST
C      BE CONTAINED BETWEEN COMMAS OR BETWEEN A COMMA AND A BLANK.
C
C
C      COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPUNCH,NUMNP,NELID,N
C      IEL2D,NEL3D,NUMEL,MHAND,NMAT,NFRC1D,NFRC2D,NFRC3D,NRCMAT,NRCTYP
C
C      K=0
10   J=IREAD(1)
     IF (J.EQ.56B) GO TO 20
     IF (J.GT.32B) GO TO 40
     I=I+1
     IF (I.EQ.81) GO TO 30
     GO TO 10
20   I=I+1
     J=IREAD(1)
     IF (J.EQ.55B.OR.J.EQ.56B) GO TO 30
     IF (J.LT.33B.OR.J.GT.44B) GO TO 40
     J=J-33B
     K=K*I0+J
     GO TO 20
30   NUMBER=K
     RETURN
C
C      40 CONTINUE
     PRINT 50, (IREAD(J),J=1,80)
     STOP
C
C
50   FORMAT (//,33H - - - PROGRAM TERMINATED - - -,//,13H INPUT ERR
     IOR,/IX,BOR)
     END

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FIRES-T3 406
FIRES-T3 407
FIRES-T3 408
FIRES-T3 409
FIRES-T3 410
FIRES-T3 411
FIRES-T3 412
FIRES-T3 413
FIRES-T3 414
FIRES-T3 415
FIRES-T3 416
FIRES-T3 417
FIRES-T3 418
FIRES-T3 419
FIRES-T3 420
FIRES-T3 421
FIRES-T3 422
FIRES-T3 423
FIRES-T3 424
FIRES-T3 425

        SUBROUTINE NODE (X,Y,Z,KODE,IN,INTOC)

C
C
C      SUBROUTINE *NODE*. INPUTS NODAL COORDINATES AND SETS
C      THE FLOW AND TEMPERATURE BOUNDARY CONDITIONS
C
C
C      COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPUNCH,NUMNP,NELID,N
C      IEL2D,NEL3D,NUMEL,MHAND,NMAT,NFRC1D,NFRC2D,NFRC3D,NRCMAT,NRCTYP
C      DIMENSION X(1), Y(1), Z(1), KODE(1), IOC(1)
C
C      OUTPUT PAGE HEADING
C
C      WRITE (NOUT,100)
C      WRITE (NOUT,110)
C      WRITE (NOUT,120) ITITLE
C      WRITE (NOUT,130)
C      WRITE (NOUT,140)
C      WRITE (NOUT,140) NUMNP
C      WRITE (NOUT,150)

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FIRE-S-T3 488      160 FORMAT (1S,3E10.0)
FIRE-S-T3 489      170 FORMAT (5(/),5H   -- - PROGRAM TERMINATED - NODE INPUT ERROR - -
FIRE-S-T3 490          1-,//1X,15,3E10.4)
FIRE-S-T3 491      180 FORMAT (16I5)
FIRE-S-T3 492      190 FORMAT (1S,3F15.4,7X,A4)
FIRE-S-T3 493      200 FORMAT (5(/),33H   -- - PROGRAM TERMINATED - - - //,47H  ERROR IN
FIRE-S-T3 494          TEMPERATURE BOUNDARY CONDITION INPUT)
FIRE-S-T3 495          END

FIRE-S-T3 496      SUBROUTINE ELEMENT (X,Y,Z,LN,MMTYPE,HAREA,THICK,VOLUME)
FIRE-S-T3 497      C
FIRE-S-T3 498      C      SUBROUTINE *ELEMENT* INPUTS ELEMENT DATA
FIRE-S-T3 499      C
FIRE-S-T3 500      C
FIRE-S-T3 501      C
FIRE-S-T3 502      COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPUNCH,NUMNP,NELID,N
FIRE-S-T3 503          NEL2D,NFL3D,NUMEL,MOND,NMAT,NFICID,NFRC2D,NFHCD,NIICMAT,NICTYP
FIRE-S-T3 504          DIMENSION X(1), Y(1), Z(1), LN(1), KX(1), MMTYPE(1), HAREA(1), TH
FIRE-S-T3 505          CK(1), VOLUME(1)
FIRE-S-T3 506          DIMENSION SI(8), TI(8), UI(8), XX(8), YY(8), ZZ(8), H(3,8), PSTL(8
FIRE-S-T3 507          1), DSTL(3,8), CJAC(3,3), POS(2)
FIRE-S-T3 508          DATA POS/-0.57735027,+0.57735027/
FIRE-S-T3 509          DATA SI/-1.,1.,-1.,1.,1.,-1.,1.,-1./
FIRE-S-T3 510          DATA TI/-1.,+1.,-1.,+1.,-1.,+1.,1.,-1./
FIRE-S-T3 511          DATA UI/-1.,-1.,+1.,-1.,+1.,-1.,1.,-1./
FIRE-S-T3 512      C
FIRE-S-T3 513      C
FIRE-S-T3 514      C      ONE - D I M E N S I O N A L   E L E M E N T S
FIRE-S-T3 515      C
FIRE-S-T3 516      C
FIRE-S-T3 517      C
FIRE-S-T3 518      C      MOND=0
FIRE-S-T3 519      C      IF (NELID>0.0) GO TO 80
FIRE-S-T3 520      C      NLM=0
FIRE-S-T3 521      C      NUM=0
FIRE-S-T3 522      C      WRITE (NIN,360) NELID
FIRE-S-T3 523      C      WRITE (NIN,370)
FIRE-S-T3 524      C      DO 70 N=1,NELID
FIRE-S-T3 525      C      NLM=ALN+2
FIRE-S-T3 526      C      IF (NUM-N) 10,20,20
FIRE-S-T3 527      C
FIRE-S-T3 528      C      READ ELEMENT CARD
FIRE-S-T3 529      C
FIRE-S-T3 530      C      10 READ (NIN,420) NUM,KX(1),KX(2),MMTYPE,HA
FIRE-S-T3 531          IF (NA>0.0) BA=1.0
FIRE-S-T3 532          IF (NUM>GT,NELID) GO TO 40
FIRE-S-T3 533          IF (N>E+1) GO TO 30
FIRE-S-T3 534      C
FIRE-S-T3 535      C      GENERATE LN ARRAY
FIRE-S-T3 536      C      20 LN(NLM-1)=LN(NLM-2)+1
FIRE-S-T3 537          LN(NLM)=LN(NLM-2)+1
FIRE-S-T3 538          MMTYPE(N)=MMTYPE(N-1) \
FIRE-S-T3 539          HAREA(N)=HAREA(N-1)
FIRE-S-T3 540      C      20 IF (NUM-N) 40,50,60
FIRE-S-T3 541      C
FIRE-S-T3 542      C      ERROR IN INPUT CARDS
FIRE-S-T3 543      C
FIRE-S-T3 544          40 WRITE (NIN,450) NUM,KX(1),KX(2),MMTYPE,HA

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FIRE5-T3 545      STOP
FIRE5-T3 546
FIRE5-T3 547
FIRE5-T3 548
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FIRE5-T3 600
FIRE5-T3 601
FIRE5-T3 602
FIRE5-T3 603
FIRE5-T3 604
FIRE5-T3 605
FIRE5-T3 606

C
      50 LM(NLM-1)=KX(1)
      LM(NLM)=KX(2)
      MMTYPE(N)=MTYPE
      BAREA(N)=BA

C
      OUTFLY

C
      60 WRITE (NOUT,480) N,LM(NLM-1),LM(NLM),MMTYPE(N),BAREA(N)

C
      DETERMINE BANDWIDTH AND STORE IF MAXIMUM

C
      J=LM(NLM-1)-LM(NLM)+1
      K=LM(NLM)-LM(NLM-1)+1
      IF (J.GT.MBAND) MBAND=J
      IF (K.GT.MBAND) MBAND=K
      70 CONTINUE

C
      TWO - D I M E N S I O N A L   E L E M E N T S

C
      EO IF (NEL2D.EQ.0) GO TO 180
      NLM=2*NELID
      NUM=0
      WRITE (NOUT,380) NEL2D
      WRITE (NOUT,400)
      DO 170 N=1,NEL2D
      NI=N*NELID
      NLW=NLW+4
      IF (NUM-NI) 90,100,100

C
      READ ELEMENT CARD

C
      90 READ (NIN,430) NUM,KX(1),KX(2),KX(3),KX(4),MTYPE,THK
      IF (THK.EQ.0.0) THK=1.0
      IF (NUM.GT.NEL2D) GO TO 120
      IF (N.EQ.1) GO TO 110

C
      GENERATE LM ARRAY

C
      100 LM(NLM-3)=LM(NLM-7)+1
      LM(NLM-2)=LM(NLM-6)+1
      LM(NLM-1)=LM(NLM-5)+1
      LM(NLM)=LM(NLM-4)+1
      MMTYPE(N1)=MMTYPE(N1-1)
      THICK(N)=THICK(N-1)
      110 IF (NUM-N) 120,130,140

C
      ERROR IN INPUT CARD

C
      120 WRITE (NOUT,460) NUM,KX(1),KX(2),KX(3),KX(4),MTYPE,THK
      STOP

C
      130 LM(NLM-3)=KX(1)
      LM(NLM-2)=KX(2)
      LM(NLM-1)=KX(3)
      LM(NLM)=KX(4)
      MMTYPE(N1)=MTYPE
      THICK(N)=THK

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FIRE5-T3 607      C      OUTPUT
FIRE5-T3 608      C
FIRE5-T3 609      C      140 WRITE (INOUT,490) N,LN(LNM-31),LN(LNM-21),LN(LNM-1),LN(LNM),MMTYPE(N)
FIRE5-T3 610      C      11,THICK(N)
FIRE5-T3 611      C
FIRE5-T3 612      C      DETERMINE BANDWIDTH AND STORE IF MAXIMUM
FIRE5-T3 613      C
FIRE5-T3 614      C      DO 160 L=1,4
FIRE5-T3 615      C      I=LN(LNM-4+L)
FIRE5-T3 616      C      DO 160 M=1,4
FIRE5-T3 617      C      J=LN(LNM-4+M)-I+1
FIRE5-T3 618      C      IF (MEAND-J) 150,160,160
FIRE5-T3 619      C      150 MIAAND=J
FIRE5-T3 620      C      160 CONTINUE
FIRE5-T3 621      C      170 CONTINUE
FIRE5-T3 622      C
FIRE5-T3 623      C
FIRE5-T3 624      C      THREE - DIMENSIONAL ELEMENTS
FIRE5-T3 625      C
FIRE5-T3 626      C
FIRE5-T3 627      C      180 IF (NEL3D.EQ.0) GO TO 350
FIRE5-T3 628      C      REWIND 6
FIRE5-T3 629      C      NLM=2*NEL1D+4*NEL2D
FIRE5-T3 630      C      NUM=0
FIRE5-T3 631      C      WRITE (INOUT,390) NEL3D
FIRE5-T3 632      C      WRITE (INOUT,410)
FIRE5-T3 633      C      DO 340 K=1,NEL3D
FIRE5-T3 634      C      N1=N+NEL1D+NEL2D
FIRE5-T3 635      C      NLM=NLM+8
FIRE5-T3 636      C      IF (NUM-N) 190,200,200
FIRE5-T3 637      C
FIRE5-T3 638      C      READ ELEMENT CARD
FIRE5-T3 639      C
FIRE5-T3 640      C      190 READ (ININ,440) NUM,(KX(I),I=1,8),MTYPE
FIRE5-T3 641      C      IF (NUM.GT.NEL3D) GO TO 230
FIRE5-T3 642      C      IF (N.EQ.1) GO TO 220
FIRE5-T3 643      C
FIRE5-T3 644      C      GENERATE LM ARRAY FOR MISSING ELEMENTS
FIRE5-T3 645      C
FIRE5-T3 646      C      200 CONTINUE
FIRE5-T3 647      C      DO 210 I=1,8
FIRE5-T3 648      C      210 LM(NLM+4*I)=LM(NLM-16+I)+1
FIRE5-T3 649      C      MMTYPE(N1)=MMTYPE(N1)-1
FIRE5-T3 650      C      220 IF (NUM-N) 230,240,260
FIRE5-T3 651      C
FIRE5-T3 652      C      ERROR IN INPUT CARDS
FIRE5-T3 653      C
FIRE5-T3 654      C      230 WRITE (INOUT,470) NUM,(KX(I),I=1,8),MTYPE
FIRE5-T3 655      C      STOP
FIRE5-T3 656      C
FIRE5-T3 657      C      240 CONTINUE
FIRE5-T3 658      C      DO 250 I=1,8
FIRE5-T3 659      C      250 LM(NLM+4*I)=KX(I)
FIRE5-T3 660      C      MMTYPE(N1)=MTYPE
FIRE5-T3 661      C
FIRE5-T3 662      C      OUTPUT
FIRE5-T3 663      C
FIRE5-T3 664      C      260 WRITE (INOUT,500) N,(LM(NLM-96+I),I=1,91),MMTYPE(N)
FIRE5-T3 665      C
FIRE5-T3 666      C      DETERMINE BANDWIDTH AND STORE IF MAXIMUM
FIRE5-T3 667      C
FIRE5-T3 668      C      DO 270 I=1,8

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FIRE-T3 669          I=LW(NLM-B+L)
FIRE-T3 670          DO 280 M=1,8
FIRE-T3 671          J=LW(NLM-B+M)-I+1
FIRE-T3 672          IF (M.EAND.J) 270,280,280
FIRE-T3 673          270 MRAND=J
FIRE-T3 674          280 CONTINUE
C
C      CALCULATE ELEMENT DATA NEEDED LATER IN ANALYSIS AND STORE ON
C      TAPE 6
C
      DO 290 I=1,8
      J=LW(NLM-B+I)
      XX(I)=X(J)
      YY(I)=Y(J)
290  ZZ(I)=Z(J)
      VOL=C.
      DO 330 III=1,2
      DO 330 JJJ=1,2
      DO 330 KKK=1,2
      SE=POS(III)
      TE=POS(JJJ)
      UE=POS(KKK)
      DO 300 I=1,8
      PSTU(I)=(I.+SE*SI(I))*(I.+TE*TI(I))*(I.+UE*UI(I))/8.
      DPSTU(1,I)=SI(I)*(I.+TE*TI(I))*(I.+UE*UI(I))/8.
      DPSTU(2,I)=TI(I)*(I.+SE*SI(I))*(I.+UE*UI(I))/8.
300  DPSTU(3,I)=UI(I)*(I.+SE*SI(I))*(I.+TE*TI(I))/8.
      DO 310 I=1,3
      CJAC(I,1)=0.
      CJAC(I,2)=0.
      CJAC(I,3)=0.
      DO 310 J=1,8
      CJAC(I,1)=CJAC(I,1)+DPSTU(I,J)*XX(J)
      CJAC(I,2)=CJAC(I,2)+DPSTU(I,J)*YY(J)
      CJAC(I,3)=CJAC(I,3)+DPSTU(I,J)*ZZ(J)
      CALL INVMAT (CJAC,DETJ,3)
      VDL=VOL+DETJ
      DO 320 I=1,3
      DO 320 J=1,8
      B(I,J)=0.
      DO 320 K=1,3
320  B(I,J)=B(I,J)+CJAC(I,K)*DPSTU(K,J)
      WRITE (6) DETJ,(PSTU(I),I=1,8),((B(I,J),J=1,8),I=1,3)
330  CONTINUE
      VOLUME(N)=VOL
340  CONTINUE
C
C      PRINT THE MAXIMUM BANDWIDTH
C
350  WRITE (NOUT,510) MRAND
      RETURN
C
C
360  FORMAT (//20H . . . . THERE ARE ,15,2H 1-D ELEMENTS . . . . )
370  FORMAT (//5H ELMT,15H   I   J   MAT,5X,5H AREA/)
380  FORMAT (//20H . . . . THERE ARE ,15,2H 2-D ELEMENTS . . . . )
390  FORMAT (//20H . . . . THERE ARE ,15,2H 3-D ELEMENTS . . . . )
400  FORMAT (//5H ELMT,25H   I   J   K   L   MAT,5X,10F THICKNESS/)


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FIRE5-T3 731      410 FORMAT (//5H ELMT,5OH   I   J   K   L   M   N   C   P   MAT
FIRE5-T3 732          I   /)
FIRE5-T3 733      420 FORMAT (4I5,F10.0)
FIRE5-T3 734      430 FORMAT (6I5,F10.0)
FIRE5-T3 735      440 FORMAT (10I5)
FIRE5-T3 736      450 FORMAT (//20H ERROR IN EL INPUT //4I5,F10.5)
FIRE5-T3 737      460 FORMAT (//20H ERROR IN CL INPUT //6I5,F10.5)
FIRE5-T3 738      470 FORMAT (//20H ERROR IN EL INPUT //10I5)
FIRE5-T3 739      480 FORMAT (4I5,F10.5)
FIRE5-T3 740      490 FORMAT (6I5,F15.5)
FIRE5-T3 741      500 FORMAT (10I5)
FIRE5-T3 742      510 FORMAT (///,.31H      * * * MAXIMUM BANDWIDTH IS ,14,6H * .)
FIRE5-T3 743      END

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FIRE5-T3 744      SUBROUTINE INVJAC (A,DETJ,NDIM)
FIRE5-T3 745          C
FIRE5-T3 746          C
FIRE5-T3 747          C      SUBROUTINE *INVJAC* INVERTS THE JACOBIAN MATRIX
FIRE5-T3 748          C
FIRE5-T3 749          C
FIRE5-T3 750          C      DIMENSION A(3,3), COFTR(3,3)
FIRE5-T3 751          C
FIRE5-T3 752          C
FIRE5-T3 753          C
FIRE5-T3 754          C      IF (NDIM.EQ.3) GO TO 20
FIRE5-T3 755          C
FIRE5-T3 756          C      2-D JACOBIAN
FIRE5-T3 757          C
FIRE5-T3 758          C      DETJ=A(1,1)*A(2,2)-A(1,2)*A(2,1)
FIRE5-T3 759          C      COFTR(1,1)=A(2,2)
FIRE5-T3 760          C      COFTR(1,2)=-A(2,1)
FIRE5-T3 761          C      COFTR(2,1)=-A(1,2)
FIRE5-T3 762          C      COFTR(2,2)=A(1,1)
FIRE5-T3 763          C      DO 10 I=1,2
FIRE5-T3 764          C      DO 10 J=1,2
FIRE5-T3 765          C      10 A(I,J)=COFTR(J,I)/DETJ
FIRE5-T3 766          C      RETURN
FIRE5-T3 767          C
FIRE5-T3 768          C      3-D JACOBIAN
FIRE5-T3 769          C
FIRE5-T3 770          C      20 DETJ=A(1,1)*(A(2,2)*A(3,3)-A(2,3)*A(3,2))-A(1,2)*(A(2,1)*A(3,3)-A(2,3)*A(3,1))+A(1,3)*(A(2,1)*A(3,2)-A(2,2)*A(3,1))
FIRE5-T3 771          C      COFTR(1,1)=A(2,2)*A(3,3)-A(2,3)*A(3,2)
FIRE5-T3 772          C      COFTR(1,2)=-A(2,1)*(A(3,3)*A(2,2)*A(3,1))
FIRE5-T3 773          C      COFTR(1,3)=A(2,1)*A(3,2)-A(2,2)*A(3,1)
FIRE5-T3 774          C      COFTR(2,1)=-A(1,2)*A(3,3)+A(1,3)*A(3,2)
FIRE5-T3 775          C      COFTR(2,2)=A(1,1)*A(3,3)-A(1,3)*A(3,1)
FIRE5-T3 776          C      COFTR(2,3)=-A(1,1)*A(3,2)+A(1,2)*A(3,1)
FIRE5-T3 777          C      COFTR(3,1)=A(1,2)*A(2,3)-A(1,3)*A(2,2)
FIRE5-T3 778          C      COFTR(3,2)=-A(1,1)*A(2,3)+A(1,3)*A(2,1)
FIRE5-T3 779          C      COFTR(3,3)=A(1,1)*A(2,2)-A(1,2)*A(2,1)
FIRE5-T3 780          C      DO 30 I=1,3
FIRE5-T3 781          C      DO 30 J=1,3
FIRE5-T3 782          C      30 A(I,J)=C(CFR(J,I))/DETJ
FIRE5-T3 783          C      RETURN
FIRE5-T3 784          C      END

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FIRES-T3 785
FIRES-T3 786
FIRES-T3 787
FIRES-T3 788
FIRES-T3 789
FIRES-T3 790
FIRES-T3 791
FIRES-T3 792
FIRES-T3 793
FIRES-T3 794
FIRES-T3 795
FIRES-T3 796
FIRES-T3 797
FIRES-T3 798
FIRES-T3 799
FIRES-T3 800
FIRES-T3 801
FIRES-T3 802
FIRES-T3 803
FIRES-T3 804
FIRES-T3 805
FIRES-T3 806
FIRES-T3 807
FIRES-T3 808
FIRES-T3 809
FIRES-T3 810
FIRES-T3 811
FIRES-T3 812
FIRES-T3 813
FIRES-T3 814
FIRES-T3 815
FIRES-T3 816
FIRES-T3 817
FIRES-T3 818
FIRES-T3 819
FIRES-T3 820
FIRES-T3 821
FIRES-T3 822
FIRES-T3 823
FIRES-T3 824
FIRES-T3 825
FIRES-T3 826
FIRES-T3 827
FIRES-T3 828
FIRES-T3 829
FIRES-T3 830
FIRES-T3 831
FIRES-T3 832
FIRES-T3 833
FIRES-T3 834
FIRES-T3 835
FIRES-T3 836
FIRES-T3 837
FIRES-T3 838
FIRES-T3 839
FIRES-T3 840
FIRES-T3 841
FIRES-T3 842
FIRES-T3 843
FIRES-T3 844
FIRES-T3 845
FIRES-T3 846

C          SUBROUTINE MATERIAL (MATL,XYS,NSTORE)
C
C          SUBROUTINE *MATERIAL* INPUTS THE NECESSARY MATERIAL PROPERTIES.
C          THE VALUES OF THESE PROPERTIES ARE EITHER IN THE FORM OF A
C          CONSTANT OR A LINEARIZED FUNCTION OF TEMPERATURE.
C          NSTORE CONTAINS THE STARTING LOCATION FOR STORING THE LINEARIZED
C          MATERIAL PROPERTY FUNCTIONS AND ALSO RETURNS THE REQUIRED STORAGE
C          FOR XYS TO FIRES-T3.
C
C          COMMON /CONTROL/ ITITLE(6),IREAD(80),KTN,NCUT,NPUNCH,NUMNP,NEL1D,N
C          NEL2D,NEL3D,NUMEL,MBOARD,NMAT,NFBC1D,NFBC2D,NFBC3D,NBCRAT,NRCTYP
C          DIMENSION MATL(1), XYS(1)
C          NSTORE=1
C
C          OUTPUT PAGE HEADING
C
C          WRITE (NOUT,201)
C          WRITE (NOUT,30)
C          WRITE (NOUT,40) ITITLE
C          WRITE (NOUT,50) NMAT
C          WRITE (NOUT,30)
C
C          DO 10 M=1,NMAT
C          WRITE (NOUT,60) M
C
C          READ IN CONTROL VARIABLES. IF INTEGER VARIABLE IS ZERO THAN
C          MATERIAL VALUE IS CONSTANT. IF INTEGER CONSTANT IS GREATER THAN
C          2 THEN IT IS THE NUMBER OF POINTS REQUIRED TO DESCRIBE THE
C          LINEARIZED TEMPERATURE DEPENDENT FUNCTION
C
C          READ (NIN,70) MK,MCP,MD
C          NS=(M-1)*6
C
C          INPUT CONDUCTIVITY DATA
C
C          WRITE (NOUT,80)
C          MATL(NS+1)=NSTORE
C          MATL(NS+2)=MK
C          CALL MATIN (MK,XYS(NSTORE),XYS(NSTORE+MK),XYS(NSTORE+MK+MK))
C          NSTORE=NSTORE+3*MK
C          IF (MK.EQ.0) NSTORE=NSTORE+1
C
C          INPUT SPECIFIC HEAT DATA
C
C          WRITE (NOUT,90)
C          MATL(NS+3)=NSTORE
C          MATL(NS+4)=MCP
C          CALL MATIN (MCP,XYS(NSTORE),XYS(NSTORE+MCP),XYS(NSTORE+MCP+MCP))
C          NSTORE=NSTORE+3*MCP
C          IF (MCP.EQ.0) NSTORE=NSTORE+1
C
C          INPUT DENSITY DATA
C
C          WRITE (NOUT,100)
C          MATL(NS+5)=NSTORE
C          MATL(NS+6)=MD
C          CALL MATIN (MD,XYS(NSTORE),XYS(NSTORE+MD),XYS(NSTORE+MD+MD))
C          NSTORE=NSTORE+3*MD
C          IF (MD.EQ.0) NSTORE=NSTORE+1

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FIRES-T3 904      WRITE (INOUT,80) I,X(I),Y(I)
FIRES-T3 905      WRITE (NOUT,90) S(I)
FIRES-T3 906      30 CONTINUE
FIRES-T3 907      WRITE (INOUT,80) K,X(K),Y(K)
FIRES-T3 908      RETURN
FIRES-T3 909      C
FIRES-T3 910      C
FIRES-T3 911      C
FIRES-T3 912      40 WRITE (NOUT,100) K
FIRES-T3 913      STOP
FIRES-T3 914      C
FIRES-T3 915      50 FORMAT (BE10.0)
FIRES-T3 916      60 FORMAT (/,39H MATERIAL PARAMETER OF CONSTANT VALUE ,G11.3)
FIRES-T3 917      70 FORMAT (15X,19HNODE TEMPERATURE,6X,5HVALUE,7X,5HSLOPE,/)
FIRES-T3 918      80 FORMAT (19,F13.1,6X,G11.3)
FIRES-T3 919      90 FORMAT (39X,G11.3)
FIRES-T3 920      100 FORMAT (//,44H -----PROGRAM TERMINATED-----,/,14H
FIRES-T3 921      1 INPUT ERROR,/,23H CONTROL CONSTANT IS ,15,22H AND IT MUST BE F
FIRES-T3 922      21THER,/,34H 0 OR GREATER THAN OR EQUAL TO .2)
FIRES-T3 923      END

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FIRES-T3 924      FUNCTION VMAT (K,X,Y,S,T,NAME)
FIRES-T3 925      C
FIRES-T3 926      C
FIRES-T3 927      C
FIRES-T3 928      C
FIRES-T3 929      C
FIRES-T3 930      C
FIRES-T3 931      COMMON /CONTROL/ TITLE(6),IREAD(80),NIN,NOUT,KPUNCH,NUNPF,NEL1D,N
FIRES-T3 932      NEL2D,NEL3D,NMEL,NBAND,NMAT,NFBC1D,NFBC2D,NFBC3D,NBCMAT,NRCTYP
FIRES-T3 933      UTMEASURE X(1), Y(1), S(1)
FIRES-T3 934      C
FIRES-T3 935      IF (K.NE.0) GO TO 10
FIRES-T3 936      VMAT=X(1)
FIRES-T3 937      RETURN
FIRES-T3 938      C
FIRES-T3 939      10 I=0
FIRES-T3 940      20 I=I+1
FIRES-T3 941      IF (I.GT.K) GO TO 50
FIRES-T3 942      IF (T-X(I)) 40,30,20
FIRES-T3 943      30 VMAT=Y(I)
FIRES-T3 944      RETURN
FIRES-T3 945      C
FIRES-T3 946      40 IF (T.EQ.1) GO TO 50
FIRES-T3 947      VMAT=Y(I-1)+S(I-1)*(T-X(I-1))
FIRES-T3 948      RETURN
FIRES-T3 949      C
FIRES-T3 950      C
FIRES-T3 951      50 WRITE (INOUT,60) NAME,T,X(I),X(K)
FIRES-T3 952      STOP
FIRES-T3 953      C
FIRES-T3 954      C
FIRES-T3 955      C
FIRES-T3 956      60 FORMAT (//,44H -----PROGRAM TERMINATED-----,/,14H
FIRES-T3 957      19H BOUNDS OF CURVE DESCRIBING MATERIAL PARAMETER ,A10,1SH HAVE
FIRES-T3 958      2 BEEN EXCEEDED,/,23H THE TEMPERATURE WAS ,F10.3,20H THE LOWER BOUND IS ,F10.3,24H AND THE UPPER BOUND IS ,F10.3
FIRES-T3 959      END
FIRES-T3 960

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FIRES-T3 961          SUBROUTINE CONVERG (INTOTAL)
FIRES-T3 962          C
FIRES-T3 963          C
FIRES-T3 964          C
FIRES-T3 965          C
FIRES-T3 966          C
FIRES-T3 967          C
FIRES-T3 968          C
FIRES-T3 969          C
FIRES-T3 970          C
FIRES-T3 971          C
FIRES-T3 972          C
FIRES-T3 973          C
FIRES-T3 974          C
FIRES-T3 975          C
FIRES-T3 976          C
FIRES-T3 977          C
FIRES-T3 978          C
FIRES-T3 979          C
FIRES-T3 980          C
FIRES-T3 981          C
FIRES-T3 982          C
FIRES-T3 983          C
FIRES-T3 984          C
FIRES-T3 985          C
FIRES-T3 986          C
FIRES-T3 987          C
FIRES-T3 988          C
FIRES-T3 989          C
FIRES-T3 990          C
FIRES-T3 991          C
FIRES-T3 992          C
FIRES-T3 993          C
FIRES-T3 994          C
FIRES-T3 995          C
FIRES-T3 996          C
FIRES-T3 997          C
FIRES-T3 998          C
FIRES-T3 999          C
FIRES-T3 1000         C
FIRES-T3 1001         C
FIRES-T3 1002         C
FIRES-T3 1003         C
FIRES-T3 1004         C
FIRES-T3 1005         C
FIRES-T3 1006         C
FIRES-T3 1007         C
FIRES-T3 1008         C
FIRES-T3 1009         C
FIRES-T3 1010         C
FIRES-T3 1011         C
FIRES-T3 1012         C
FIRES-T3 1013         C
FIRES-T3 1014         C
FIRES-T3 1015         C
FIRES-T3 1016         C
FIRES-T3 1017         C
FIRES-T3 1018         C
FIRES-T3 1019         C
FIRES-T3 1020         C
FIRES-T3 1021         C
FIRES-T3 1022         C

          SUBROUTINE *CONVERG* INPUTS THE CONVERGENCE CRITERIA CONTROLLING
          THE NATURE OF THE THERMAL ANALYSIS.  THE PROGRAM CONTAINS THE
          THE CAPABILITIES FOR ITERATIVE SOLUTIONS DEALING WITH THE ENTIRE
          SYSTEM AND/OR THE FIRE BOUNDARY CONDITIONS

          COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPUNCH,NUMNP,NEL1D,N
          IEL2D,NEL3D,NMEL,NRND,NMAT,NFBC1D,NFBC2D,NFBC3D,NBCMAT,NBCTYP
          COMMON /CONRG/ NCONV,CONV,BETA,NCONU,CONU,ALPHA

          OUTPUT PAGE HEADING

          WRITE (NOUT,20)
          WRITE (NOUT,30)
          WRITE (NOUT,40) ITITLE
          WRITE (NOUT,50)
          WRITE (NOUT,30)
          WRITE (NOUT,60)

          CONVERGENCE DETERMINED WHEN

          2*ABS(T(I)-T(I-1))/T(I)+T(I-1) .LESS THAN. CONV OR CONU

          AND IF CONVERGENCE IS NOT ACHIEVED THE NEXT ESTIMATE OF THE
          SYSTEMS TEMPERATURES IS OBTAINED THROUGH

          T(I+1) = T(I) + BETA (OR ALPHA) *(T(I)-T(I-1))

          WHERE      SYSTEM      FIRE BC
                  CRITERIA    CRITERIA

          NCONU      NCONV   - NUMBER OF PERMISSIBLE
                      - ITERATIONS (IF PARTICULAR
                      - ITERATION IS NOT DEFINED)
          CONU       CONV    - PERMISSABLE RELATIVE ERROR
          ALPHA      BETA    - OVER RELAXATION FACTOR

          INPUT CONVERGENCE CRITERIA

          READ (NIN,70) NCONV,CONV,BETA,NCONU,CONU,ALPHA
          IF (NCONU.EQ.0) GO TO 10

          OUTPUT SYSTEM CONVERGENCE CRITERIA

          WRITE (NOUT,80) CONU,NCONU,ALPHA
10 CONTINUE

          OUTPUT FIRE BOUNDARY CONDITION CONVERGENCE CRITERIA

          WRITE (NOUT,90) CONV,NCONV,BETA

          DIVIDE PERMISSABLE ERROR BY TWO

          CONV=CONV*.50
          CONU=CONU*.50

          STORAGE REQUIREMENT FOR FLANK COMMON IS NOW PRINTED

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FIRES-T3 1080          READ (NIN,370) IA,NDT,TIME,TEMP,JP
FIRES-T3 1081          C
FIRES-T3 1082          C   OUTPUT HEADING FOR INITIAL TIME STEP
FIRES-T3 1083          C
FIRES-T3 1084          C   WRITE (NINOUT,380)
FIRES-T3 1085          C   WRITE (NINOUT,390)
FIRES-T3 1086          C   WRITE (NINOUT,400) ITITLE
FIRES-T3 1087          C
FIRES-T3 1088          C   IF (IA,NE,4HSTEP) GO TO 40
FIRES-T3 1089          C
FIRES-T3 1090          C   WRITE (NINOUT,410) NDT,TIME
FIRES-T3 1091          C   WRITE (NINOUT,390)
FIRES-T3 1092          C
FIRES-T3 1093          C   INITIALIZE TEMPERATURES
FIRES-T3 1094          C
FIRES-T3 1095          C   IF (TEMP,NE,0.0) GO TO 10
FIRES-T3 1096          C   READ (NIN,420) TT(I),I=1,NUMNP
FIRES-T3 1097          C   GO TO 30
FIRES-T3 1098          C   10 DO 20 I=1,NUMNP
FIRES-T3 1099          C   20 TT(I)=TEMP
FIRES-T3 1100          C
FIRES-T3 1101          C   OUTPUT INITIAL TEMPERATURES
FIRES-T3 1102          C
FIRES-T3 1103          C   30 CALL PROUT (4,1,AT,LM,T1,D,MAIN,NCON,1)
FIRES-T3 1104          C   GO TO 50
FIRES-T3 1105          C
FIRES-T3 1106          C   40 CONTINUE
FIRES-T3 1107          C
FIRES-T3 1108          C   TERMINATE PROGRAM - INITIAL TIME STEP CARD ERROR
FIRES-T3 1109          C
FIRES-T3 1110          C   WRITE (NINOUT,430) IA,NDT,TIME,TEMP,JP
FIRES-T3 1111          C   STOP
FIRES-T3 1112          C
FIRES-T3 1113          C
FIRES-T3 1114          C   50 CONTINUE
FIRES-T3 1115          C
FIRES-T3 1116          C   READ TIME STEP CARD
FIRES-T3 1117          C   WHERE IA = CONTROL WORD (STEP)
FIRES-T3 1118          C   NDT = SEQUENCE NUMBER
FIRES-T3 1119          C   DT = TIME STEP INTERVAL
FIRES-T3 1120          C   ITOF = NUMBER OF NON-ZERO TEMPERATURE OR FLOW B.C.
FIRES-T3 1121          C   TFIRE(I) = TEMPERATURE OF FIRE FOR CURRENT TIME STEP
FIRES-T3 1122          C   I CAN VARY FROM 1 TO 4
FIRES-T3 1123          C   II = PRINTED OUTPUT OPTION
FIRES-T3 1124          C   0 = NO PRINTED OUTPUT
FIRES-T3 1125          C   1 = OUTPUT NODAL POINT TEMPERATURES
FIRES-T3 1126          C   2 = OUTPUT ELEMENT TEMPERATURES
FIRES-T3 1127          C   3 = OUTPUT BOTH NODAL AND ELEMENT TEMPERATURES
FIRES-T3 1128          C   12 = PUNCHED OUTPUT OPTION
FIRES-T3 1129          C   CODE SAME AS FOR PRINTED DATA
FIRES-T3 1130          C   16 = DEBUGGING OUTPUT OPTION
FIRES-T3 1131          C   17 = CHANGE OF FIRE SURFACE ELEMENTS OPTION
FIRES-T3 1132          C
FIRES-T3 1133          C   60 READ (NIN,440) IA,NDT,DT,ITOF,TFIRE(1),TFIRE(2),TFIRE(3),TFIRE(4),
FIRES-T3 1134          C   IT1,IT2,IT3,IT4
FIRES-T3 1135          C   MAIN=C
FIRES-T3 1136          C   NDT=NDT+1
FIRES-T3 1137          C
FIRES-T3 1138          C   CHECK SEQUENCING OF TIME STEP CARD
FIRES-T3 1139          C
FIRES-T3 1140          C   IF (NDT,EQ,NDT,AND,IA,EQ,4HSTEP) GO TO 90
FIRES-T3 1141          C
FIRES-T3 1142          C   PROGRAM TERMINATED IF SEQUENCING ERROR

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FIREST3 1142
FIREST3 1143
FIREST3 1144
FIREST3 1145
FIREST3 1146
FIPES-T3 1147
FIREST3 1148
FIREST3 1149
FIREST3 1150
FIREST3 1151
FIREST3 1152
FIREST3 1153
FIREST3 1154
FIREST3 1155
FIREST3 1156
FIREST3 1157
FIREST3 1158
FIREST3 1159
FIREST3 1160
FIREST3 1161
FIREST3 1162
FIREST3 1163
FIREST3 1164
FIREST3 1165
FIREST3 1166
FIREST3 1167
FIREST3 1168
FIREST3 1169
FIREST3 1170
FIREST3 1171
FIREST3 1172
FIREST3 1173
FIREST3 1174
FIREST3 1175
FIREST3 1176
FIREST3 1177
FIREST3 1178
FIREST3 1179
FIPES-T3 1180
FIREST3 1181
FIREST3 1182
FIREST3 1183
FIREST3 1184
FIPES-T3 1185
FIREST3 1186
FIREST3 1187
FIREST3 1188
FIPES-T3 1189
FIREST3 1190
FIREST3 1191
FIPES-T3 1192
FIREST3 1193
FIREST3 1194
FIPES-T3 1195
FIREST3 1196
FIREST3 1197
FIREST3 1198
FIREST3 1199
FIREST3 1200
FIREST3 1201
FIREST3 1202
FIPES-T3 1203

C
      WRITE (INOUT,450)
70  WRITE (INOUT,460) TA,NDT,DT,ITDF,TFIRE(1),TFIRE(2),TFIRE(3),TFIRE(4)
     1,11,12,16,17
      STCP
80  WRITE (INOUT,470)
      GO TO 70

C
      ESTABLISH TIME INTERVAL DT
C
90  IF (DT) 100,110,120
100  WRITE (INOUT,480)
      RETURN

C
110  IF (DE.EQ.0.0) GO TO 80
     DT=DS
     GO TO 130
120  DS=DT
130  CONTINUE

C
      TIME=TIME+DT

C
      OUTPUT HEADING FOR TIME STEP
C
      WRITE (INOUT,380)
      WRITE (INOUT,390)
      WRITE (INOUT,400) ITITLE
      WRITE (INOUT,490) NDT,TIME,DT
      WRITE (INOUT,390)
      WRITE (INOUT,500) ITDF
      IF (NUMFBC.EQ.0) GO TO 140
      WRITE (INOUT,510) (I,TFIRE(I),I=1,4)

140  CONTINUE
     DT2=1./DT

C
C
      BEGINNING OF SYSTEM ITERATION
C
150  MAIN=MAIN+1
C
      SAVE THE INITIALLY ASSUMED TEMPERATURES IN VECTOR T1
C
      DO 160 N=1,NUMNP
160  T1(N)=T(N)
      IF (T1.EQ.0) CALL PROUT (1,T,AT,LM,T1,B,MAIN,NCON,11)

C
      FORM CONDUCTIVITY MATRIX AND STORE IN MATRIX A, AND ALSO FORM
      THE HEAT CAPACITY MATRIX AND STORE IN VECTOR Q
C
      CALL HCONDc (AT,A,NUMNP,MATL,XYS,T,Q,LM,MNTYPE,BAREA,THICK,X,Y,Z,V
     IOLUME)

C
      INITIALIZE HEAT LOAD VECTOR TO 0.0 - VECTOR R
C
      DO 170 I=1,NUMNP
170  R(I)=0.0

C
      ADD INTERNAL HEAT GENERATION TO VECTOR R
C
      IF (KINT.EQ.0) GO TO 180
      CALL GEXG (LM,IEL+IMAT,TEXO,EXYS,AT,MATL,VFL,MMTYPE,R,XYS,TIME)

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FIRES-T3 1204
FIRES-T3 1205
FIRES-T3 1206
FIRES-T3 1207
FIRES-T3 1208
FIRES-T3 1209
FIRES-T3 1210
FIRES-T3 1211
FIRES-T3 1212
FIRES-T3 1213
FIRES-T3 1214
FIRES-T3 1215
FIRES-T3 1216
FIRES-T3 1217
FIRES-T3 1218
FIRES-T3 1219
FIRES-T3 1220
FIRES-T3 1221
FIRES-T3 1222
FIRES-T3 1223
FIRES-T3 1224
FIRES-T3 1225
FIRES-T3 1226
FIRES-T3 1227
FIRES-T3 1228
FIRES-T3 1229
FIRES-T3 1230
FIRES-T3 1231
FIRES-T3 1232
FIRES-T3 1233
FIRES-T3 1234
FIRES-T3 1235
FIRES-T3 1236
FIRES-T3 1237
FIRES-T3 1238
FIRES-T3 1239
FIRES-T3 1240
FIRES-T3 1241
FIRES-T3 1242
FIRES-T3 1243
FIRES-T3 1244
FIRES-T3 1245
FIRES-T3 1246
FIRES-T3 1247
FIRES-T3 1248
FIRES-T3 1249
FIRES-T3 1250
FIRES-T3 1251
FIRES-T3 1252
FIRES-T3 1253
FIRES-T3 1254
FIRES-T3 1255
FIRES-T3 1256
FIRES-T3 1257
FIRES-T3 1258
FIRES-T3 1259
FIRES-T3 1260
FIRES-T3 1261
FIRES-T3 1262
FIRES-T3 1263
FIRES-T3 1264
FIRES-T3 1265

C      INPUT ANY NON-ZERO TEMPERATURE AND FLOW BOUNDARY CONDITIONS
C      AND ADD THE RELATED TERMS TO MATRIX A AND VECTOR B
C
180 CALL HATEMP (1TOF,D,KODE,B,A,NUMNP,MAIN,T3,C)
C      ADD CAPACITY TERMS TO CONDUCTIVITY MATRIX A
C
DO 200 N=1,NUMNP
IF (KODE(N).EQ.4HTEMP) GO TO 200
IF (C(N)) 190,200,190
190 A(N,1)=A(N,1)+Q(N)*DT2
200 CONTINUE
C      TRIANGULARIZE THE MODIFIED CONDUCTIVITY MATRIX
C
CALL MSYM (1,B,MA,A,NUMNP)
C
IF (MAIN.NE.1) GO TO 220
C      CALCULATE EFFECTIVE LOAD VECTOR E
C
C      CALCULATE CAPACITY MATRIX CONTRIBUTION TO THE EFFECTIVE LOAD
C      AND SAVE IN VECTOR T2
C
DO 210 II=1,NUMNP
IF (KODE(II).EQ.4HTEMP) GO TO 210
T2(II)=Q(II)*T(II)*DT2
210 CONTINUE
220 DO 230 II=1,NUMNP
IF (KODE(II).EQ.4HTEMP) GO TO 230
B(II)=B(II)+T2(II)
230 Q(II)=B(II)
NCQA=0
IF (NUMFBC.EQ.0) GO TO 260
C      IF FIRE BOUNDARY CONDITION SURFACE CONFIGURATION IS TO BE CHANGED
C      ON THIS TIME STEP, INPUT THE NEW DATA
C
IF (17.EQ.0) GO TO 250
CALL FIREBC (X,Y,Z,KODE,BAREA,THICK,L1,LJ,JK,LL,LMAT,LFIRE,ATJKL,L
IELEM)
C
C      BEGINNING OF FIRE B.C. ITERATION
C
240 CONTINUE
C
C      CALCULATE THE HEAT FLOW RELATED TO THE FIRE B.C.
C
250 CALL FIRE (LT,LJ,JK,LL,LMAT,LFIRE,ATJKL,MAF,EXY,G,T,TFIRE,P)
NCN=NCN+1
C
C      CALCULATE THE TEMPERATURES THROUGH BACK SUBSTITUTION
C
260 CALL MSYM (2,B,MA,A,NUMNP)
C
IF (NUMFBC.EQ.0) GO TO 280
IF (NCN>0) GO TO 240
IF (16.NE.0) CALL PRDOUT (2,T,ATJKL,LL,MAIN,NCQA,II)
C

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FIRES-T3 1266      C      CHECK FOR CONVERGENCE OF FIRE B.C. CYCLE AGAINST PERMISSIBLE ERROR
FIRES-T3 1267      C
FIRES-T3 1268      DO 270 N=1,NUMNP
FIRES-T3 1269      DX=AES(B(N)-T(N))
FIRES-T3 1270      DY=CCNV*ABS(B(N)+T(N))
FIRES-T3 1271      IF (DX.GT.DY) GO TO 300
FIRES-T3 1272      270 CONTINUE
FIRES-T3 1273      260 DO 290 N=1,NUMNP
FIRES-T3 1274      290 T(N)=F(N)
FIRES-T3 1275      GC TO 320
FIRES-T3 1276
FIRES-T3 1277      C      IF CONVERGENCE NOT OBTAINED CHECK FOR POSSIBILITY OF EXCEEDING
FIRES-T3 1278      C      PERMISSIBLE NUMBER OF CYCLES FOR FIRE B.C.
FIRES-T3 1279      C
FIRES-T3 1280      360 IF (NCON.GT.NCCNV) CALL PROUT (3,T,AT,LH,T1,B,MAIN,NCON,11)
FIRES-T3 1281      C      ESTIMATE NEW APPROXIMATION OF TEMPERATURES FOR FIRE B.C.
FIRES-T3 1282      C
FIRES-T3 1283      FIRES-T3 1284      DO 310 JJ=1,NUMNP
FIRES-T3 1285      DX=B(JJ)-T(JJ)
FIRES-T3 1286      T(JJ)=B(JJ)+BETA*DX
FIRES-T3 1287      310 P(JJ)=C(JJ)
FIRES-T3 1288      GO TO 240
FIRES-T3 1289
FIRES-T3 1290      320 IF (NCCNU.EQ.0) GC TO 360
FIRES-T3 1291
FIRES-T3 1292      C      CHECK CONVERGENCE OF SYSTEM CYCLE AGAINST PERMISSIBLE ERROR
FIRES-T3 1293      C
FIRES-T3 1294      DO 330 N=1,NUMNP
FIRES-T3 1295      DX=AES(T(N)-T1(N))
FIRES-T3 1296      DY=CONU*ABS(T(N)+T1(N))
FIRES-T3 1297      IF (DX.GT.DY) GO TO 340
FIRES-T3 1298      330 CONTINUE
FIRES-T3 1299      GO TO 360
FIRES-T3 1300
FIRES-T3 1301      C      CHECK TO SEE IF NUMBER OF SYSTEM ITERATIONS HAS EXCEEDED
FIRES-T3 1302      C      PERMISSIBLE NUMBER OF ITERATIONS
FIRES-T3 1303      C
FIRES-T3 1304      340 IF (MAIN.GT.NCCNU) CALL PROUT (3,T,AT,LH,T1,B,MAIN,NCCN,11)
FIRES-T3 1305
FIRES-T3 1306      C      ESTIMATE NEW APPROXIMATION OF SYSTEMS TEMPERATURES
FIRES-T3 1307      C
FIRES-T3 1308      DO 350 N=1,NUMNP
FIRES-T3 1309      DX=T(N)-T1(N)
FIRES-T3 1310      350 T(N)=T(N)+ALPHA*DX
FIRES-T3 1311      GO TO 150
FIRES-T3 1312      360 CONTINUE
FIRES-T3 1313
FIRES-T3 1314      C      CHECK FOR DESIRED OUTPUT
FIRES-T3 1315      C
FIRES-T3 1316      IF (11.NE.0) CALL PROUT (4,T,AT,LH,T1,B,MAIN,NCON,11)
FIRES-T3 1317      IF (12.NE.0) CALL PROUT (11,12,T,AT,X,Y,Z,TIME,IP1,IP2,LH,JP)
FIRES-T3 1318      NC=NCCN-1
FIRES-T3 1319      NS=MAIN-1
FIRES-T3 1320      WRITE (NDOUT,530) NS
FIRES-T3 1321      WRITE (NDOUT,520) NC
FIRES-T3 1322      GO TO 60
FIRES-T3 1323
FIRES-T3 1324
FIRES-T3 1325
FIRES-T3 1326      370 FORMAT (A4,16.2F10.0,2X,43)
FIRES-T3 1327      380 FORMAT (1H6.5(/))

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FIRES-T3 1328
FIRES-T3 1329
FIRES-T3 1330
FIRES-T3 1331
FIRES-T3 1332
FIRES-T3 1333
FIRES-T3 1334
FIRES-T3 1335
FIRES-T3 1336
FIRES-T3 1337
FIRES-T3 1338
FIRES-T3 1339
FIRES-T3 1340
FIRES-T3 1341
FIRES-T3 1342
FIRES-T3 1343
FIRES-T3 1344
FIRES-T3 1345
FIRES-T3 1346
FIRES-T3 1347
FIRES-T3 1348
FIRES-T3 1349
FIRES-T3 1350
FIRES-T3 1351
FIRES-T3 1352

390 FORMAT (6I8 *****)
400 FORMAT (/5X,50HFIRES-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL,/
1/IX,6A10)
410 FORMAT (/1X,27HINITIAL SEQUENCE NUMBER IS ,I4,2I8 AND INITIAL TIME
1 IS ,F8.2)
420 FORMAT (7(4X,F6.1))
430 FORMAT (5(/),69H - - - PROGRAM TERMINATED - ERROR IN INITIAL TIM
1E STEP CARD - - -,//1X,A4,I6,2F10.2,2X,A3)
440 FORMAT (A4,I6,F10.0,I5,4F10.0,C4I3)
450 FORMAT (///,43H TIME STEP CARD OUT OF SEQUENCE - CARD NO.,I5/,12H
1 INPLT CARD)
460 FORMAT (//,57H - - - PROGRAM TERMINATED - TIME STEP CARD WAS -
1- - -,//1X,A4,I6,F10.2,I5,4F10.2,4I5)
470 FORMAT (///,30H NO TIME INTERVAL ESTABLISHED)
480 FORMAT (///,19H PROBLEM COMPLETED)
490 FORMAT (//2X,17HTIME STEP NUMBER ,I4,8H - TIME ,F7.3,I3H - TIME STE
IP ,F7.3/)
500 FORMAT (//5X,50H NUMBER OF NON-ZERO FLOW OR TEMPERATURE CONDITIONS,
1I5)
510 FORMAT (//5X,24H FIRE BOUNDARY CONDITION, 4(/7X,S#FIRE(,11,4H) = ,F1
10.3,2X))
520 FORMAT (15,32H N. C. ITERATIONS WERE PERFORMED)
530 FORMAT (//15,33H SYSTEM ITERATIONS WERE PERFORMED)
END

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FIRES-T3 1353
FIRES-T3 1354
FIRES-T3 1355
FIRES-T3 1356
FIRES-T3 1357
FIRES-T3 1358
FIRES-T3 1359
FIRES-T3 1360
FIRES-T3 1361
FIRES-T3 1362
FIRES-T3 1363
FIRES-T3 1364
FIRES-T3 1365
FIRES-T3 1366
FIRES-T3 1367
FIRES-T3 1368
FIRES-T3 1369
FIRES-T3 1370
FIRES-T3 1371
FIRES-T3 1372
FIRES-T3 1373
FIRES-T3 1374
FIRES-T3 1375
FIRES-T3 1376
FIRES-T3 1377
FIRES-T3 1378
FIRES-T3 1379
FIRES-T3 1380
FIRES-T3 1381
FIRES-T3 1382
FIRES-T3 1383
FIRES-T3 1384

SUBROUTINE HCONDG (AT,A,NP,MATL,XYS,T,Q,LH,MTYPE,BAREA,THICK,X,Y,
IZ,VOLUME)
C
C
C   SUBROUTINE *HCONDG* FORMS CONDUCTIVITY AND HEAT CAPACITY MATRICES
C
C
COMMON /CONTROL/ ITITLE(6),ITREAD(H0),NIN,NOUT,NPLUNCH,NUMNP,NELEID,N
IEL2D,NELE3D,NUMEL,MRAND,NMAT,NFRC10,NFHC2D,NFHC3D,NDCMAT,NDCtyp
DIMENSION AT(1), A(NP,1), MATL(1), XYS(1), T(1), Q(1), S(8,8), LMC
(1), PSTU(8), B(3,8), MTYPE(1), BAREA(1), THICK(1), X(1), Y(1), Z(
21), FCS(2), SI(4), TI(4), XX(4), YY(4), DRSTU(2,4), CJAC(3,3), VOL
3UME(1)
DATA FCS/-0.57735027,+0.57735027/
DATA SI/-1.+1.,1.+1./
DATA TI/-1.,-1.,1.+1./
C
C
C   INITIALIZE THE SYSTEMS CONDUCTIVITY MATRIX TO 0.0
C   AND THE SYSTEMS HEAT CAPACITY MATRIX TO 0.0
C
DO 10 I=1,NUMNP
10  Q(I)=0.
      M0=MRAND*NUMNP
      DO 20 I=1,M0
20  A(I)=0.0
C
C   ONE-DIMENSIONAL ELEMENTS
C
      T0 = NFRC10,IQ,0.01,0.01

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FIREST3 1385
FIREST3 1386
FIREST3 1387
FIREST3 1388
FIREST3 1389
FIREST3 1390
FIREST3 1391
FIREST3 1392
FIREST3 1393
FIREST3 1394
FIREST3 1395
FIREST3 1396
FIREST3 1397
FIREST3 1398
FIREST3 1399
FIREST3 1400
FIREST3 1401
FIREST3 1402
FIREST3 1403
FIREST3 1404
FIREST3 1405
FIREST3 1406
FIREST3 1407
FIREST3 1408
FIREST3 1409
FIREST3 1410
FIREST3 1411
FIREST3 1412
FIREST3 1413
FIREST3 1414
FIREST3 1415
FIREST3 1416
FIREST3 1417
FIREST3 1418
FIREST3 1419
FIREST3 1420
FIREST3 1421
FIREST3 1422
FIREST3 1423
FIREST3 1424
FIREST3 1425
FIREST3 1426
FIREST3 1427
FIREST3 1428
FIREST3 1429
FIREST3 1430
FIREST3 1431
FIREST3 1432
FIREST3 1433
FIREST3 1434
FIREST3 1435
FIREST3 1436
FIREST3 1437
FIREST3 1438
FIREST3 1439
FIREST3 1440
FIREST3 1441
FIREST3 1442
FIREST3 1443
FIREST3 1444
FIREST3 1445
FIREST3 1446

      NLM=C
      DO 40 N=1,NEL1D
      NLM=NLM+2
      DO 30 I=1,2
      DO 30 J=1,2
      30 S(I,J)=C+0
      II=LH(NLM-1)
      KK=LH(NLM)
      AA=X(II)-X(KK)
      BB=Y(II)-Y(KK)
      CC=Z(II)-Z(KK)
      XL=SQRT(AA*AA+BB*BB+CC*CC)
      AT(N)=0.5*(T(II)+T(KK))
      TEMP=AT(N)
      MS=NNTYPE(N)
      MS=(MS-1)*6
      J=MATL(MS+1)
      K=MATL(MS+2)
      COND=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   K(T)  )
      J=MATL(MS+3)
      K=MATL(MS+4)
      SPHT=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   CP(T)  )
      J=MATL(MS+5)
      K=MATL(MS+6)
      DENS=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   D(T)  )
      S(1,1)=BAREA(N)*COND/XL
      S(1,2)=-S(1,1)
      S(2,1)=S(1,2)
      S(2,2)=S(1,1)
      QSTCFE=SPHT*DENS*XK*BAREA(N)/2.

      C      ADD ELEMENT CONDUCTIVITY AND CAPACITY TO SYSTEM MATRICES
      C
      C      O(II)=O(II)+QSTORE
      C      Q(KK)=Q(KK)+QSTORE
      C      A(II,1)=A(II,1)+S(1,1)
      C      JJ=KK-II+1
      C      IF (JJ.GT.0) A(II,JJ)=A(II,JJ)+S(1,2)
      C      JJ=II-KK+1
      C      IF (JJ.GT.0) A(KK,JJ)=A(KK,JJ)+S(1,2)
      C      A(KK,1)=A(KK,1)+S(2,2)
      40 CONTINUE

      C      T W O - D I M E N S I O N A L   E L E M E N T S
      C
      C
      50 IF (NEL2D.EQ.0) GO TO 270
      NLM=2*NEL1D
      DO 260 N=1,NEL2D
      NI=N+NEL1D
      NLM=NLM+4
      DO 60 I=1,4
      DO 60 J=1,4
      60 S(I,J)=0.0
      LL1=LH(NLM-3)
      LL2=LH(NLM-2)
      LL3=LH(NLM-1)
      LL4=LH(NLM)
      AT(NI)=0.25*(T(LL1)+T(LL2)+T(LL3)+T(LL4))

      C      TEST FOR ORIENTATION OF 2-D ELEMENT - X-Y, X-Z, OR Y-Z PLANE

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FIRE5-T3 1447
FIRE5-T3 1448
FIRE5-T3 1449
FIRE5-T3 1450
FIRE5-T3 1451
FIRE5-T3 1452
FIRE5-T3 1453
FIRE5-T3 1454
FIRE5-T3 1455
FIRE5-T3 1456
FIRE5-T3 1457
FIRE5-T3 1458
FIRE5-T3 1459
FIRE5-T3 1460
FIRE5-T3 1461
FIRE5-T3 1462
FIRE5-T3 1463
FIRE5-T3 1464
FIRE5-T3 1465
FIRE5-T3 1466
FIRE5-T3 1467
FIRE5-T3 1468
FIRE5-T3 1469
FIRE5-T3 1470
FIRE5-T3 1471
FIRE5-T3 1472
FIRE5-T3 1473
FIRE5-T3 1474
FIRE5-T3 1475
FIRE5-T3 1476
FIRE5-T3 1477
FIRE5-T3 1478
FIRE5-T3 1479
FIRE5-T3 1480
FIRE5-T3 1481
FIRE5-T3 1482
FIRE5-T3 1483
FIRE5-T3 1484
FIRE5-T3 1485
FIRE5-T3 1486
FIRE5-T3 1487
FIRE5-T3 1488
FIRE5-T3 1489
FIRE5-T3 1490
FIRE5-T3 1491
FIRE5-T3 1492
FIRE5-T3 1493
FIRE5-T3 1494
FIRE5-T3 1495
FIRE5-T3 1496
FIRE5-T3 1497
FIRE5-T3 1498
FIRE5-T3 1499
FIRE5-T3 1500
FIRE5-T3 1501
FIRE5-T3 1502
FIRE5-T3 1503
FIRE5-T3 1504
FIRE5-T3 1505
FIRE5-T3 1506
FIRE5-T3 1507
FIRE5-T3 1508
C
      J0=LW(NLM-3)
      ZZZ=Z(J0)
      DO 70 I=1,3
      J=LW(NLM-3+I)
      DZ=AES(ZZZ-Z(J))
      IF (DZ.GT.+0.00001) GO TO 90
 70 CONTINUE
      DO 80 I=1,4
      J=LW(NLM-4+I)
      XX(I)=X(J)
      80 YY(I)=Y(J)
      GO TO 160
 90 YYYY=Y(J0)
      DO 100 I=1,3
      J=LW(NLM-3+I)
      DY=APS(YYYY-Y(J))
      IF (DY.GT.+0.00001) GO TO 120
100 CONTINUE
      DO 110 I=1,4
      J=LW(NLM-4+I)
      XX(I)=X(J)
      110 YY(I)=Z(J)
      GO TO 160
120 XXX=X(J0)
      DO 130 I=1,3
      J=LW(NLM-3+I)
      DX=AES(XXX-X(I))
      IF (DX.GT.+0.00001) GO TO 150
130 CONTINUE
      DO 140 I=1,4
      J=LW(NLM-4+I)
      XX(I)=Y(J)
      140 YY(I)=Z(J)
      GO TO 160
150 WRITE (NOUT,370) N
      STOP
160 VOL=C.C
      DO 220 III=1,2
      DO 220 JJJ=1,2
      SF=POS(III)
      TE=PCS(JJJ)
      DO 170 I=1,4
      PSTU(I)=(1.+SF*SI(I))*(1.+TE*T1(I))/4.
      DPSTU(1,I)=SI(I)*(1.+TE*T1(I))/4.
      170 DPSTU(2,I)=T1(I)*(1.+SF*SI(I))/4.
      DO 180 I=1,2
      CJAC(1,I)=0.0
      CJAC(1,2)=0.0
      DO 180 J=1,4
      CJAC(1,I)=(JAC(1,1)+DPSTU(1,J))*XX(J)
      180 CJAC(1,2)=CJAC(1,2)+DPSTU(1,J)*YY(J)
      CALL INVMAT (CJAC,DETJ,2)
      VCL=VCL+DETJ*THICK(N)
      DO 190 I=1,2
      DO 190 J=1,4
      B(I,J)=0.0
      DO 190 K=1,2
      190 B(I,J)=B(I,J)+CJAC(I,K)*DPSTU(K,J)
      ATT=C.
      DO 200 I=1,4
      L=LW(NLM-4+I)

```

```

FIRE5-T3 1509      200 ATT=ATT+PSTU(I)*T(L)
FIRE5-T3 1510      TEMP=ATT
FIRE5-T3 1511      MS=MNTYPE(N)
FIRE5-T3 1512      MS=(MS-1)*6
FIRE5-T3 1513      J=MATL(MS+1)
FIRE5-T3 1514      K=MATL(MS+2)
FIRE5-T3 1515      COND=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   K(T)    )
FIRE5-T3 1516      J=MATL(MS+3)
FIRE5-T3 1517      K=MATL(MS+4)
FIRE5-T3 1518      SPHT=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   CP(T)    )
FIRE5-T3 1519      J=MATL(MS+5)
FIRE5-T3 1520      K=MATL(MS+6)
FIRE5-T3 1521      DEENS=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   DET(T)    )
FIRE5-T3 1522      DETCON=DETJ*COND*THICK(N)
FIRE5-T3 1523      DC 210 I=1,4
FIRE5-T3 1524      DO 210 J=1,4
FIRE5-T3 1525      DO 210 K=1,2
FIRE5-T3 1526      210 S(I,J)=S(I,J)+DETCON*B(K,I)*E(K,J)
FIRE5-T3 1527      220 CONTINUE
FIRE5-T3 1528      OSTORE=DEENS*SPHT*VOL/4.

C
C      ADD ELEMENT CAPACITY MATRIX TO SYSTEM CAPACITY MATRIX
C
DO 230 L=1,4
I=L*M(NLM-4+L)
230 Q(I)=Q(I)+OSTORE

C
C      ADD ELEMENT CONDUCTIVITY TO SYSTEM CONDUCTIVITY MATRIX
C
DO 250 L=1,4
I=L*M(NLM-4+L)
DO 250 M=1,4
J=L*M(NLM-4+M)-I+1
IF (J) 250,25C,240
240 A(I,J)=A(I,J)+S(L,M)
250 CONTINUE
260 CONTINUE

C
C      THREE - D I M E N S I O N A L   E L E M E N T S
C
270 IF (NEL3D.EQ.0) GO TO 360
REWIND 6
NLM=2*NEL1D+4*NEL2D
DO 35C N=1,NEL3D
N1=N+NEL1D+NEL2D
NLM=NLM+B
DO 280 I=1,8
DO 280 J=1,8
280 S(I,J)=0.0
LL1=L*M(NLM-7)
LL2=L*M(NLM-6)
LL3=L*M(NLM-5)
LL4=L*M(NLM-4)
LL5=L*M(NLM-3)
LL6=L*M(NLM-2)
LL7=L*M(NLM-1)
LL8=L*M(NLM)
AT(N1)=+.125*(T(LL1)+T(LL2)+T(LL3)+T(LL4)+T(LL5)+T(LL6)+T(LL7)+T(LL
181)
DO 310 I1=1,2

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```

FIRE5-T3 1571      DO 310 JJJ=1,2
FIRE5-T3 1572      DO 310 KKK=1,2
FIRE5-T3 1573      READ (6) DETJ,(PSTU(I),I=1,8),((E(I,J),J=1,8),I=1,3)
FIRE5-T3 1574      ATT=0.
FIRE5-T3 1575      DO 290 I=1,8
FIRE5-T3 1576      L=LW(NLM-B+I)
FIRE5-T3 1577      290 ATT=ATT+FSTU(I)+T(L)
FIRE5-T3 1578      TEMP=ATT
FIRE5-T3 1579      MS=MATTYPE(N1)
FIRE5-T3 1580      ME=(ME-1)*6
FIRE5-T3 1581      J=MATL(MS+1)
FIRE5-T3 1582      K=MATL(MS+2)
FIRE5-T3 1583      COND=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   K(T)   )
FIRE5-T3 1584      J=MATL(MS+3)
FIRE5-T3 1585      K=MATL(MS+4)
FIRE5-T3 1586      SPHT=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   CP(T)   )
FIRE5-T3 1587      J=MATL(MS+5)
FIRE5-T3 1588      K=MATL(MS+6)
FIRE5-T3 1589      DENS=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H   D(T)   )
FIRE5-T3 1590      DETCON=DETJ+COND
FIRE5-T3 1591      DO 300 I=1,8
FIRE5-T3 1592      DO 300 J=1,8
FIRE5-T3 1593      DO 300 K=1,3
FIRE5-T3 1594      300 S(I,J)=S(I,J)+DETCON*B(I,K,I)*B(K,J)
FIRE5-T3 1595      310 CONTINUE
FIRE5-T3 1596      VOL=VOLUME(N)
FIRE5-T3 1597      QSTORE=DENS*SFHT*VOL/B.
C
C      ADD ELEMENT CAPACITY MATRIX TO SYSTEM CAPACITY MATRIX
C
DO 320 L=1,8
I=LW(NLM-B+L)
320 O(I)=O(I)+QSTORE
C
C      ADD ELEMENT CONDUCTIVITY TO THE SYSTEMS CONDUCTIVITY MATRIX - A
C
DO 340 L=1,8
I=LW(NLM-B+L)
DO 340 M=1,8
J=LW(NLM-B+M)-I+1
IF (J) 340,340,330
330 A(I,J)=A(I,J)+S(L,M)
340 CONTINUE
350 CONTINUE
C
360 RETURN
C
C
370 FORMAT (/>3IH STOP-ERROR IN 2-D ELEMENT NO. ,16,/>3OH NOT IN X-Y, X
1-Z, OR Y-Z PLANE)
END

```

```

FIRE5-T3 1622      SUBROUTINE HATEMP (ITOF,D,KODE,P,A,ND,MAIN,FT,J)
FIRE5-T3 1623      C
FIRE5-T3 1624      C
FIRE5-T3 1625      C
FIRE5-T3 1626      C
FIRE5-T3 1627      C
C
C      SUBROUTINE KHATEMP APPLIES THE FIXED TEMPERATURE OR FLOW
C      BOUNDARY CONDITIONS
C

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FIRES-T3 1628
FIRES-T3 1629
FIRES-T3 1630
FIRES-T3 1631
FIRES-T3 1632
FIRES-T3 1633
FIRES-T3 1634
FIRES-T3 1635
FIRES-T3 1636
FIRES-T3 1637
FIRES-T3 1638
FIRES-T3 1639
FIRES-T3 1640
FIRES-T3 1641
FIRES-T3 1642
FIRES-T3 1643
FIRES-T3 1644
FIRES-T3 1645
FIRES-T3 1646
FIRES-T3 1647
FIRES-T3 1648
FIRES-T3 1649
FIRES-T3 1650
FIRES-T3 1651
FIRES-T3 1652
FIRES-T3 1653
FIRES-T3 1654
FIRES-T3 1655
FIRES-T3 1656
FIRES-T3 1657
FIRES-T3 1658
FIRES-T3 1659
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FIRES-T3 1661
FIRES-T3 1662
FIRES-T3 1663
FIRES-T3 1664
FIRES-T3 1665
FIRES-T3 1666
FIRES-T3 1667
FIRES-T3 1668
FIRES-T3 1669
FIRES-T3 1670
FIRES-T3 1671
FIRES-T3 1672
FIRES-T3 1673
FIRES-T3 1674
FIRES-T3 1675
FIRES-T3 1676
FIRES-T3 1677
FIRES-T3 1678
FIRES-T3 1679
FIRES-T3 1680
FIRES-T3 1681
FIRES-T3 1682
FIRES-T3 1683
FIRES-T3 1684
FIRES-T3 1685
FIRES-T3 1686
FIRES-T3 1687
FIRES-T3 1688

C
COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,KPUNCH,KUNPF,KELID,N
IEL2D,KEL3D,KUMEL,MBAND,NMAT,NFRCID,NFRC2D,NFRC3D,NHCMAT,NRCTYP
DIMENSION D(1), KODE(1), B(1), A(NP,1), FT(1), J(1)
IF (MAIN.NE.1) GO TO 30
C
C      INITIALIZE TEMPERATURE AND FLOW B.C. TO 0.0
C
DO 10 IC I=1,NUMNP
10 D(I)=0.0
C
IF (ITOF.EQ.0) GO TO 30
C
WRITE (NOUT,100)
C
C      INPUT NON-ZERO TEMPERATURE AND FLOW B.C. .
C
READ (NIN,110) (J(I),FT(I),I=1,ITOF)
WRITE (NOUT,120)
C
C      OUTPUT THE NON-ZERO BOUNDARY CONDITIONS AND STORE IN MATRIX D
C
DO 20 I=1,ITOF
  IT=J(I)
  D(I)=FT(I)
  JJ=KODE(IT)
  WRITE (NOUT,130) IT,JJ,D(I)
20 CONTINUE
C
30 DO 90 N=1,NUMNP
C
C      MODIFY MATRIX B FOR FLOW B.C.
C
  B(N)=E(N)+D(N)
  IF (KODE(N).EQ.4*HFLOW) GO TO 90
C
C      MODIFY A AND B MATRIX FOR TEMPERATURE B.C.
C
  DO 80 M=2,MBAND
    K=N-M+1
    IF (K) 50,50,40
    40 B(K)=B(K)-A(K,M)*D(N)
    A(K,M)=0.0
    50 L=N+M-1
    IF (NUMNP-L) 70,60,60
    60 B(L)=E(L)-A(N,M)*D(N)
    70 A(N,M)=0.0
    80 CONTINUE
    A(N,1)=1.0
    B(N)=C(N)
  90 CONTINUE
C
RETURN
C
C
100 FORMAT (//5X,31F-VALUES OF TEMPERATURES OR FLOWS/10X,32HFOR NON-ZER
10 BOUNDARY CONDITIONS)
110 FORMAT (5(1S,F10.2))
120 FORMAT (/,BH      NODE,7X,AHTYPE,10X,5HVALUE)
130 FORMAT (1B,7X,A4,5X,F10.2)
END

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FIRES-T3 1689
FIRES-T3 1690
FIRES-T3 1691
FIRES-T3 1692
FIRES-T3 1693
FIRES-T3 1694
FIRES-T3 1695
FIRES-T3 1696
FIRES-T3 1697
FIRES-T3 1698
FIRES-T3 1699
FIRES-T3 1700
FIRES-T3 1701
FIRES-T3 1702
FIRES-T3 1703
FIRES-T3 1704
FIRES-T3 1705
FIRES-T3 1706
FIRES-T3 1707
FIRES-T3 1708
FIRES-T3 1709
FIRES-T3 1710
FIRES-T3 1711
FIRES-T3 1712
FIRES-T3 1713
FIRES-T3 1714
FIRES-T3 1715
FIRES-T3 1716
FIRES-T3 1717
FIRES-T3 1718
FIRES-T3 1719
FIRES-T3 1720
FIRES-T3 1721
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FIRES-T3 1724
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FIRES-T3 1726
FIRES-T3 1727
FIRES-T3 1728
FIRES-T3 1729
FIRES-T3 1730
FIRES-T3 1731
FIRES-T3 1732
FIRES-T3 1733
FIRES-T3 1734
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FIRES-T3 1736
FIRES-T3 1737
FIRES-T3 1738
FIRES-T3 1739
FIRES-T3 1740
FIRES-T3 1741
FIRES-T3 1742
FIRES-T3 1743
FIRES-T3 1744
FIRES-T3 1745
FIRES-T3 1746
FIRES-T3 1747
FIRES-T3 1748
FIRES-T3 1749
FIRES-T3 1750

C      SUBROUTINE MSYM (KKK,B,MA,A,KP)
C
C      SUBROUTINE *MSYM* IS AN EQUATION SOLVER
C      BASED ON A MODIFIED SYMSOL - VARIABLE BANDWIDTH WITH ZEROS IN BAND
C
C      COMMON /CCNTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPUNCH,NUMNP,NEL1D,N
C      IEL2D,NEL3D,NUMEL,MBAND,NMAT,NFRC1D,NFRC2D,NFBC3D,NBCMAT,NBCTYF
C      DIMENSION B(1), MA(1), A(NP,1)
C
C      NEO=NUMNP
C
C      GO TO (10,70), KKK
C
C      *****
C      REDUCE MATRIX.... A
C
C      *****
C
C      10 NEO0=NEQ-1
C          DO 60 N=1,NEQ
C
C          M=MBAND
C          DO 20 I=2,MBAND
C              IF (A(N,I).NE.0.) GO TO 30
C          20 M=M-1
C          20 MA(N)=M
C
C          I=N
C          DO 50 L=2,M
C              I=I+1
C              CC=A(N,L)/A(N,1)
C              IF (CC.EQ.0.) GO TO 50
C              J=0
C              DO 40 K=L,M
C                  J=J+1
C                  40 A(I,J)=A(I,J)-CC*A(N,K)
C                  A(N,L)=CC
C              50 CONTINUE
C          60 CONTINUE
C          GO TO 120
C
C      *****
C      REDUCE VECTOR.... B AND BACKSUBSTITUTE
C
C      *****
C
C      70 DO 90 N=1,NFEC
C          CC=B(N)
C          IF (CC.EQ.0.) GO TO 90
C          M=MA(N)
C          I=N
C          DO 80 L=2,N
C              I=I+1
C              B(I)=B(I)-CC*A(N,I)
C              B(N)=CC*A(N,I)
C          80 CONTINUE
C          B(NFEC)=B(NFEC)/A(NFEC,1)
C

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FIRE5-T3 1751          NN=NFO
FIRE5-T3 1752          DO 110 K=1,NEGO
FIRE5-T3 1753          NN=NN-1
FIRE5-T3 1754          K=KA(NN)
FIRE5-T3 1755          I=NN
FIRE5-T3 1756          DO 100 K=2,M
FIRE5-T3 1757          I=I+1
FIRE5-T3 1758          100 B(NN)=B(NN)-A(NN,K)*B(I)
FIRE5-T3 1759          110 CONTINUE
FIRE5-T3 1760          120 RETURN
FIRE5-T3 1761          END

FIRE5-T3 1762          SUBROUTINE PROUT (K,T,AT,LM,T1,B,MAIN,NCON,I1)
C
C
FIRE5-T3 1763          SUBROUTINE *PROUT* PRINTS TEMPERATURE DISTRIBUTIONS
C
C
FIRE5-T3 1764          ( BOTH NODAL AND ELEMENT )
C
C
FIRE5-T3 1765          COMMON /CONTROL/ ITITLE(6),IRREAD(80),NIN,NIOUT,NPLACH,NUMNP,KEL1D,
FIRE5-T3 1766          KEL2D,KEL3D,NUMEL,MRAND,MMAT,NFRC1D,NFBC2D,NFBC3D,NBCMAT,NRCTYP
FIRE5-T3 1767          DIMENSION T(1), AT(1), TI(1), B(1), LM(1)
C
C
FIRE5-T3 1768          GO TO 10,20,30,40, K
C
C
FIRE5-T3 1769          10 CONTINUE
C
C
FIRE5-T3 1770          DEBUGGING OUTPUT FOR TEMPERATURES AT BEGINNING OF SYSTEM CYCLE
C
C
FIRE5-T3 1771          WRITE (NCUT,140) MAIN
FIRE5-T3 1772          WRITE (NOUT,230) (N,T(N),N=1,NUMNP)
FIRE5-T3 1773          RETURN
C
C
FIRE5-T3 1774          20 CONTINUE
C
C
FIRE5-T3 1775          DEBUGGING OUTPUT TEMPERATURES FOR FIRE B.C. CYCLE
C
C
FIRE5-T3 1776          WRITE (NCUT,150) NCON
FIRE5-T3 1777          WRITE (NOUT,230) (N,B(N),N=1,NUMNP)
FIRE5-T3 1778          RETURN
C
C
FIRE5-T3 1779          30 CONTINUE
C
C
FIRE5-T3 1780          OUTPUT DATA FOR DUMP WHEN PROBLEM HAS NOT CONVERGED AFTER
FIRE5-T3 1781          PERMISSIELE NUMBER OF CYCLES
C
C
FIRE5-T3 1782          WRITE (NCUT,160)
FIRE5-T3 1783          WRITE (NCUT,170) MAIN,NCON
FIRE5-T3 1784          WRITE (NOUT,180)
FIRE5-T3 1785          WRITE (NOUT,230) (N,T1(N),N=1,NUMNP)
FIRE5-T3 1786          WRITE (NOUT,190)
FIRE5-T3 1787          WRITE (NCUT,230) (N,T(N),N=1,NUMNP)
FIRE5-T3 1788          WRITE (NOUT,200)
FIRE5-T3 1789          WRITE (NOUT,230) (N,B(N),N=1,NUMNP)
FIRE5-T3 1790          STOP
C
C
FIRE5-T3 1791          40 CONTINUE

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FIRES-T3 1808
FIRES-T3 1809
FIRES-T3 1810
FIRES-T3 1811
FIRES-T3 1812
FIRES-T3 1813
FIRES-T3 1814
FIRES-T3 1815
FIRES-T3 1816
FIRES-T3 1817
FIRES-T3 1818
FIRES-T3 1819
FIRES-T3 1820
FIRES-T3 1821
FIRES-T3 1822
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FIRES-T3 1824
FIRES-T3 1825
FIRES-T3 1826
FIRES-T3 1827
FIRES-T3 1828
FIRES-T3 1829
FIRES-T3 1830
FIRES-T3 1831
FIRES-T3 1832
FIRES-T3 1833
FIRES-T3 1834
FIRES-T3 1835
FIRES-T3 1836
FIRES-T3 1837
FIRES-T3 1838
FIRES-T3 1839
FIRES-T3 1840
FIRES-T3 1841
FIRES-T3 1842
FIRES-T3 1843
FIRES-T3 1844
FIRES-T3 1845
FIRES-T3 1846
FIRES-T3 1847
FIRES-T3 1848
FIRES-T3 1849
FIRES-T3 1850
FIRES-T3 1851
FIRES-T3 1852
FIRES-T3 1853
FIRES-T3 1854
FIRES-T3 1855
FIRES-T3 1856
FIRES-T3 1857
FIRES-T3 1858
FIRES-T3 1859
FIRES-T3 1860
FIRES-T3 1861
FIRES-T3 1862
FIRES-T3 1863
FIRES-T3 1864
FIRES-T3 1865
FIRES-T3 1866
FIRES-T3 1867
FIRES-T3 1868
FIRES-T3 1869

C          OUTPUT OF RESULTS FOR A TIME STEP
C
C          IF (II.EC.1.OR.II.EC.3) 50,60
      EO CONTINUE
C          OUTPUT NODAL POINT TEMPERATURES
C
C          WRITE (INCUT,210)
C          WRITE (INCUT,220)
C          WRITE (INCUT,230) (N,T(N),N=1,NUMNP)
      EO CONTINUE
C          IF (II.EC.2.OR.II.EC.3) 70,130
C          OUTPUT ELEMENT TEMPERATURES
C          ONE-DIMENSIONAL ELEMENTS
C
      70 IF (NEL1D.EC.0) GO TO 90
      NLM=C
      WRITE (INCUT,240)
      WRITE (NOUT,220)
      DO 80 N=1,NEL1D
      NLM=NLM+2
      LL1=LNM(NLM-1)
      LL2=LNM(NLM)
      EO AT(N)=C*5*(T(LL1)+T(LL2))
      WRITE (NOUT,230) (N,AT(N),N=1,NEL1D)
C          TWO-DIMENSIONAL ELEMENTS
C
      90 IF (NEL2D.EC.0) GO TO 110
      NLM=2*NEL1D
      WRITE (INCUT,250)
      WRITE (NOUT,220)
      DO 100 N=1,NEL2D
      NLM=NLM+4
      LL1=LNM(NLM-3)
      LL2=LNM(NLM-2)
      LL3=LNM(NLM-1)
      LL4=LNM(NLM)
      100 AT(N+NEL1D)=0.25*(T(LL1)+T(LL2)+T(LL3)+T(LL4))
      WRITE (NOUT,230) (N,AT(N+NEL1D),N=1,NEL2D)
C          THREE-DIMENSIONAL ELEMENTS
C
      110 IF (NEL3D.EC.0) GO TO 130
      NLM=2*NEL1D+4*NEL2D
      WRITE (INCUT,260)
      WRITE (NOUT,220)
      DO 120 N=1,NEL3D
      NLM=NLM+8
      LL1=LNM(NLM-7)
      LL2=LNM(NLM-6)
      LL3=LNM(NLM-5)
      LL4=LNM(NLM-4)
      LL5=LNM(NLM-3)
      LL6=LNM(NLM-2)
      LL7=LNM(NLM-1)
      LL8=LNM(NLM)
      NI=N+NEL1D+NEL2D
      120 AT(N)=0.125*(T(LL1)+T(LL2)+T(LL3)+T(LL4)+T(LL5)+T(LL6)+T(LL7)+T(LL8))

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FIRES-T3 1870      1L91)
FIRES-T3 1871      N2=NEL1D+NEL2D
FIRES-T3 1872      WRITE (NOUT,230) (A,AT(N+N2),K=1,NEL3D)
C
C   130 RETURN
C
C   140 FORMAT (//,62H NODAL POINT TEMPERATURES AT BEGINNING OF SYSTEM CYC
ILE NUMBER,16/)
150 FORMAT (//,40H NODAL POINT TEMPERATURE FOR B.C. CYCLE,15/)
160 FORMAT (//,20H PROGRAM TERMINATED,/,59H CONVERGENCE NOT OBTAINED
I IN REQUIRED NUMBER OF ITERATIONS)
170 FORMAT (//,15H SYSTEM CYCLE ,15,16H AND B.C. CYCLE ,15)
180 FORMAT (//,33H SYSTEM NODAL POINT TEMPERATURES)
190 FORMAT (//,53H NODAL POINT TEMPERATURES AT BEGINNING OF B.C. CYCL
IE)
200 FORMAT (//,47H NODAL POINT TEMPERATURES AT END OF B.C. CYCLE)
210 FORMAT (//54H ----- NODAL POINT TEMPERATURES -----
1-)
220 FORMAT (IX,4(1SH N TEMP. ))
230 FORMAT (4(F6,F9.2))
240 FORMAT (//,57H ----- TEMPERATURE OF 1-D ELEMENTS -----
1---/ )
250 FORMAT (//,57H ----- TEMPERATURE OF 2-D ELEMENTS -----
1---/ )
260 FORMAT (//,57H ----- TEMPERATURE OF 3-C ELEMENTS -----
1---/ )
END

```

```

FIRES-T3 1898      SUBROUTINE POUT (I1,I2,T,AT,X,Y,Z,TIME,IP1,IP2,LN,JP)
FIRES-T3 1899
FIRES-T3 1900
FIRES-T3 1901
FIRES-T3 1902
FIRES-T3 1903
FIRES-T3 1904
FIRES-T3 1905
FIRES-T3 1906
FIRES-T3 1907
FIRES-T3 1908
FIRES-T3 1909
FIRES-T3 1910
FIRES-T3 1911
FIRES-T3 1912
FIRES-T3 1913
FIRES-T3 1914
FIRES-T3 1915
FIRES-T3 1916
FIRES-T3 1917
FIRES-T3 1918
FIRES-T3 1919
FIRES-T3 1920
FIRES-T3 1921
FIRES-T3 1922
FIRES-T3 1923
FIRES-T3 1924
FIRES-T3 1925
FIRES-T3 1926

C
C   SUBROUTINE *POUT* PUNCHES THE TEMPERATURE DISTRIBUTIONS THAT
C   RESULT FROM THE ANALYSIS DONE IN THE PROGRAM.
C   JP - IDENTIFIER TO BE USED IN LAST 8 COLUMNS
C   IP1 - COUNTER FOR NODAL DATA CARDS PUNCHED
C   IP2 - COUNTER FOR ELEMENT DATA CARDS PUNCHED
C
C   COMMON /CONTROL/ ITITLE(6),IRFAD(80),NIN,NOUT,RPUNCH,RUPNF,NEL1D,N
IEL2D,NEL3D,NUMEL,MBOARD,NMAT,NFHCD,NFBCCD,NFBCC3D,NBCMAT,NBCTYP
DIMENSION T(1), AT(1), X(1), Y(1), Z(1), LN(1)
INTEGER ELEM
NP=NFLACH
C
C   IF (I2.EQ.1.OR.I2.EQ.3) 10,100
C
C   PUNCHING NODAL DATA
C
10 WRITE (NOUT,240)
NODE=4NNODE
IF (JF,FC,3H ) GO TO 20
ENCODE (4,250,NNOD) JP
20 CONTINUE
IF (IP1.NE.0) GO TO 60
C
C   NODAL COORDINATES PUNCHED THE FIRST TIME TEMPERATURE DATA IS
C   REQUESTED

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```

FIRE5-T3 1927          C
FIRE5-T3 1928          IF(I1=IP1+1)
FIRE5-T3 1929          WRITE (NP,260) ITITLE, NODE, IP1
FIRE5-T3 1930          IP1=IP1+1
FIRE5-T3 1931          WRITE (NP,270) NODE, IP1
FIRE5-T3 1932          IP1=IP1+1
FIRE5-T3 1933          WRITE (NP,280) NODE, IP1
FIRE5-T3 1934          N1=1
FIRE5-T3 1935          30 N2=N1+2
FIRE5-T3 1936          IP1=IP1+1
FIRE5-T3 1937          IF (NUMNP-N2) 50,40,40
FIRE5-T3 1938          40 WRITE (NP,290) (I,X(I),Y(I),Z(I),I=N1,N2),NCDE,IP1
FIRE5-T3 1939          IF (N2.EQ.NUMNP) GO TO 60
FIRE5-T3 1940          N1=N2+1
FIRE5-T3 1941          GO TO 30
FIRE5-T3 1942          50 N2=NUMNP
FIRE5-T3 1943          N=NUMNP+1-N1
FIRE5-T3 1944          M=72-N*22
FIRE5-T3 1945          ENCODE (30,300,L) N,M
FIRE5-T3 1946          WRITE (NP,L) (I,X(I),Y(I),Z(I),I=N1,N2),NCDE,IP1
FIRE5-T3 1947          C PUNCHING NODAL POINT TEMPERATURES
FIRE5-T3 1948          C
FIRE5-T3 1949          C
FIRE5-T3 1950          60 IF(I1=IP1+1)
FIRE5-T3 1951          WRITE (NP,310) TIME,NUMNP,NCDE,IP1
FIRE5-T3 1952          N1=1
FIRE5-T3 1953          70 N2=N1+6
FIRE5-T3 1954          IP1=IP1+1
FIRE5-T3 1955          IF (NUMNP-N2) 90,80,80
FIRE5-T3 1956          80 WRITE (NP,320) (I,T(I),I=N1,N2),NCDE,IP1
FIRE5-T3 1957          IF (N2.EQ.NUMNP) GO TO 100
FIRE5-T3 1958          N1=N2+1
FIRE5-T3 1959          GO TO 70
FIRE5-T3 1960          90 N2=NUMNP
FIRE5-T3 1961          N=NUMNP+1-N1
FIRE5-T3 1962          M=72-N*10
FIRE5-T3 1963          ENCODE (30,330,L) N,M
FIRE5-T3 1964          WRITE (NP,L) (I,T(I),I=N1,N2),NODE,IP1
FIRE5-T3 1965          100 IF ((P2.EQ.2.0R.P1.EQ.3)) 110,230
FIRE5-T3 1966          C PUNCHING ELEMENT DATA
FIRE5-T3 1967          C
FIRE5-T3 1968          C
FIRE5-T3 1969          110 WRITE (NOUT,340)
FIRE5-T3 1970          IF ((P2.NE.0)) GO TO 130
FIRE5-T3 1971          ELEM=4*IELEM
FIRE5-T3 1972          IF (JP.EQ.3H) 1 GO TO 120
FIRE5-T3 1973          ENCODE (4,350,ELEM) JP
FIRE5-T3 1974          120 CONTINUE
FIRE5-T3 1975          C PUNCHING ELEMENT TEMPERATURES IN ORDER 1-D, 2-D, J-C.
FIRE5-T3 1976          C
FIRE5-T3 1977          130 IP2=IP2+1
FIRE5-T3 1978          WRITE (NP,360) TIME,NUMEL,ELEM,IP2
FIRE5-T3 1979          IF ((1.E0.2.0R.P1.EQ.3)) GO TO 190
FIRE5-T3 1980          C CALCULATE THE AVERAGE ELEMENT TEMPERATURE
FIRE5-T3 1981          C
FIRE5-T3 1982          IF ((NELEL.D.EQ.0)) GO TO 150
FIRE5-T3 1983          NLIM=0
FIRE5-T3 1984          DO 140 N=1,NELEL
FIRE5-T3 1985          NLIM=NLIM+2
FIRE5-T3 1986          NELEL=NLIM-1
FIRE5-T3 1987          140
FIRE5-T3 1988

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FIRE5-T3 1980          LL2=LM(NLM)
FIRE5-T3 1990          140 AT(N)=0.5*(T(LL1)+T(LL2))
FIRE5-T3 1991          150 IF (NEL20.EQ.0) GO TO 170
FIRE5-T3 1992          NLM=NLM+1
FIRE5-T3 1993          DO 160 N=1,NEL20
FIRE5-T3 1994          NLM=NLM+4
FIRE5-T3 1995          LL1=LM(NLM-3)
FIRE5-T3 1996          LL2=LM(NLM-2)
FIRE5-T3 1997          LL3=LM(NLM-1)
FIRE5-T3 1998          LL4=LM(NLM)
FIRE5-T3 1999          160 AT(N+NEL1D)=0.25*(T(LL1)+T(LL2)+T(LL3)+T(LL4))
FIRE5-T3 2000          170 IF (NEL3D.EQ.0) GO TO 190
FIRE5-T3 2001          NLM=2*NEL1D+4*NEL2D
FIRE5-T3 2002          DO 180 N=1,NEL3D
FIRE5-T3 2003          NLM=NLM+8
FIRE5-T3 2004          LL1=LM(NLM-7)
FIRE5-T3 2005          LL2=LM(NLM-6)
FIRE5-T3 2006          LL3=LM(NLM-5)
FIRE5-T3 2007          LL4=LM(NLM-4)
FIRE5-T3 2008          LL5=LM(NLM-3)
FIRE5-T3 2009          LL6=LM(NLM-2)
FIRE5-T3 2010          LL7=LM(NLM-1)
FIRE5-T3 2011          LL8=LM(NLM)
FIRE5-T3 2012          N1=N+NEL1D+NEL2D
FIRE5-T3 2013          180 AT(N)=.125*(T(LL1)+T(LL2)+T(LL3)+T(LL4)+T(LL5)+T(LL6)+T(LL7)+T(LL8))
C
C 150 CONTINUE
C 151 N1=1
C 200 N2=N1+6
C     IP2=IP2+1
C     IF (NUMFL-N2) 220,210,210
C 210 WRITE (NP,320) (I,AT(I),I=N1,N2),ELEM,IP2
C     IF (N2.EQ.NUMEL) GO TO 230
C     N1=N2+1
C     GO TO 200
C 220 N2=NUMEL
C     N=NUMEL+1-N1
C     M=72-N*10
C     ENCODE (30,330,L) N,M
C     WRITE (NP,L) (I,AT(I),I=N1,N2),ELEM,IP2
C 230 CONTINUE
C
C     RETURN
C
C 240 FORMAT (//,3H   * * * PUNCHING NODAL DATA * * *)
C 250 FORMAT (1H,N,A3)
C 260 FORMAT (7A10,A2,A4,I4)
C 270 FORMAT (52H NODAL POINT TEMPERATURES FOR SELECT TIME INTERVALS,2D
C           IX,A6,I4)
C 280 FORMAT (40H NODAL POINT COORDINATES - NUMBER,X,Y,Z,32X,A4,I4)
C 290 FORMAT (3(I4,3F6.3),6X,A4,I4)
C 300 FORMAT (1H(,I1,1H(14,3F6.3),,12,8HX,A4,I4))
C 310 FORMAT (41H ---NODAL POINT TEMPERATURES AT TIME = ,F7.3,3H = ,E1
C           1,6H NODES,12X,A4,I4)
C 320 FORMAT (7(I4,F6.1),2X,A4,I4)
C 330 FORMAT (1H(,I1,10H(14,F6.1),,12,8HX,A4,I4))
C 340 FORMAT (//,3H   * * * PUNCHING ELEMENT DATA * * *)
C 350 FORMAT (1H,E,A3)
C 360 FORMAT (37H ---ELEMENT TEMPERATURES AT TIME = ,F7.3,3H = ,E2,8H
C           ELEMENTS,13X,A4,I4)

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FIREST-13 2051

END

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FIRE5-T3 2052          SUBROUTINE FIREMAT (MAT,FXYS,NSTORE)
FIRE5-T3 2053
FIRE5-T3 2054
C
C   SUBROUTINE *FIREMAT* INPUTS THE VARIABLES REQUIRED IN THE
C   ASSESSMENT OF THE HEAT FLOW ASSOCIATED WITH BOTH LINEAR AND
C   NON-LINEAR FIRE BOUNDARY CONDITIONS
C
C
C   COMMON /CONTROL/ ITITLE(6),TREAD(80),NIN,NOUT,NPUNCH,NUMNP,NEL1D,N
C   IEL2D,NEL3D,NUMEL,MRAND,NMAT,NFBCID,NFBC2D,NFBC3D,NBCMAT,NRCTYP
C   DIMENSION MAT(1), FXYS(1)
C
C   OUTPUT PAGE HEADING
C
C
C   WRITE (NOUT,40)
C   WRITE (NOUT,50)
C   WRITE (NOUT,60) ITITLE
C
C   IF (NRCTYP.EQ.1)HLINEAR BC ) GO TO 20
C
C   NON-LINEAR FIRE BOUNDARY CONDITION
C
C
C   WRITE (NOUT,70)
C   WRITE (NOUT,50)
C   WRITE (NOUT,80)
C   READ (NIN,90) SP,TSHIFT
C   WRITE (NOUT,100) SP,TSHIFT
C   WRITE (NOUT,110)
C
C   FXYS(1)=50
C   FXYS(2)=TSHIFT
C   NSTORE=3
C
C   INPUT DIFFERENT MATERIAL PROPERTIES FOR FIRE BC
C
C
C   DO 10 IF1,NBCMAT
C   MAT(IF1)=NSTORE
C   READ (NIN,90) A,P,V,AB,EF,ES
C   WRITE (NOUT,120) 1,A,P,V,AB,EF,ES
C   FXYS(NSTORE)=A
C   FXYS(NSTORE+1)=P
C   FXYS(NSTORE+2)=V
C   FXYS(NSTORE+3)=AB
C   FXYS(NSTORE+4)=EF
C   FXYS(NSTORE+5)=ES
C   NSTORE=NSTORE+6
C   10 CONTINUE
C   RETURN
C
C   20 CONTINUE
C
C   LINEAR FIRE BOUNDARY CONDITION
C
C
C   WRITE (NOUT,130)
C   WRITE (NOUT,50)
C   WRITE (NOUT,140)

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FIRE5-T3 2108
FIRE5-T3 2109
FIRE5-T3 2110
FIRE5-T3 2111
FIRE5-T3 2112
FIRE5-T3 2113
FIRE5-T3 2114
FIRE5-T3 2115
FIRE5-T3 2116
FIRE5-T3 2117
FIRE5-T3 2118
FIRE5-T3 2119
FIRE5-T3 2120
FIRE5-T3 2121
FIRE5-T3 2122
FIRE5-T3 2123
FIRE5-T3 2124
FIRE5-T3 2125
FIRE5-T3 2126
FIRE5-T3 2127
FIRE5-T3 2128
FIRE5-T3 2129
FIRE5-T3 2130
FIRE5-T3 2131
FIRE5-T3 2132
FIRE5-T3 2133
FIRE5-T3 2134
FIRE5-T3 2135
FIRE5-T3 2136
FIRE5-T3 2137
FIRE5-T3 2138
FIRE5-T3 2139
FIRE5-T3 2140
FIRE5-T3 2141
FIRE5-T3 2142
FIRE5-T3 2143
FIRE5-T3 2144
FIRE5-T3 2145
FIRE5-T3 2146
FIRE5-T3 2147
FIRE5-T3 2148
FIRE5-T3 2149
FIRE5-T3 2150

C      NSTORE=1
      DC 30 I=1,NBRCMAT
      WRITE (INPUT,150) I
      READ (NIN,160) K
      MS=(I-1)*2
      MAT(MS+1)=NSTORE
      MAT(MS+2)=K

C      CALL MATIN (K,FXY5(NSTORE),FXY5(NSTORE+K),FXY5(NSTORE+K+K))

C      NSTORE=NSTORE+3*K
      IF (K.EQ.0) NSTORE=NSTORE+1
      20 CONTINUE

C      RETURN

C      *****
C      40 FORMAT (1H6,5(1/))
      50 FORMAT (6H *****)
      60 FORMAT (/5X50HFIRE5-T3 - FIRE RESPONSE OF STRUCTURES - THERMAL,/
     1/IX,6A10)
      70 FORMAT (/5X,34HNCN-LINEAR FIRE BOUNDARY CONDITION,/)
      80 FORMAT (//1X,60H0=A*(TF-TS)**N+SH*V*(AB*EF*(TF+TSHIFT)**4-ES*(TS+T
     1SHIFT)**4))
      90 FORMAT (E10.0)
      100 FORMAT (//1X,5WWHERE/6X,28HTF - PSUEDO FIRE TEMPERATURE,/6X,24HTS
     1-SURFACE TEMPERATURE,/6X,33HST - STEFAN BOLZTMANN CONSTANT = ,E12
     2.4/6X,47HTSHIFT - SHIFT TO ABSOLUTE TEMPERATURE SCALE = ,F8.1/1X,3
     3HND)
      110 FORMAT (//58H MAT CONVECT CONVECT VIEW AEMCFRT FIRE SU
     1RFACE,/58H NUM FACTOR POWER FACTOR EMISSIV EMISS
     21V,/10X,3H(A),6X,3H(N),6X,3H(V),5X,4H(EA),5X,4H(EE),5X,4H(ES))
      120 FORMAT (1/14,6F9.3)
      130 FORMAT (/5X,30HLINEAR FIRE BOUNDARY CONDITION,/)
      140 FORMAT (//9X,16H0 = H(T)*(TF-TS)//10X,5WWHERE,/15X,31HH(T) - HEAT
     1TRANSFER COEFFICIENT,/15X,28HTF - PSUEDO FIRE TEMPERATURE,/15X,24HT
     2S - SURFACE TEMPERATURE,/15X,33HT - AVERAGE TEMPERATURE (TF+TS)/2,
     3//)
      150 FORMAT (//,25H    . . . MATERIAL NUMBER ,14,6H . . . )
      160 FORMAT (15)
      END

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FIRE-S-T3 2151          SUBROUTINE FIREPC (X,Y,Z,KODE,RAREA,THICK,L1,L2,LK,LMAT,LFIRE,A
FIRE-S-T3 2152          ITJKL,LELEM)
FIRE-S-T3 2153          C
FIRE-S-T3 2154          C
FIRE-S-T3 2155          C      SUBROUTINE *FIREPC* INPUTS THE GEOMETRIC DESCRIPTION OF THE
FIRE-S-T3 2156          C      SURFACE OF THE SYSTEM THAT WILL BE DIRECTLY EXPOSED TO THE
FIRE-S-T3 2157          C      FIRE ENVIRONMENT
FIRE-S-T3 2158          C
FIRE-S-T3 2159          C
FIRE-S-T3 2160          COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NIOUT,NPLNCH,NUMPY,NFLID,N
FIRE-S-T3 2161          IEL2D,NFL3D,NUMEL,NPAND,NMAT,NPCED,NPC2D,NPC3D,NHCMAT,NHC1YP
FIRE-S-T3 2162          COMMON /SURFACE/ NS1,NS2,NS3
FIRE-S-T3 2163          DIMENSION X(1), Y(1), KODE(1), U(1), LU(1), LMAT(1), LFIRE(1), ZU
FIRE-S-T3 2164          LU(1), LK(1), LMAT(1), LFIRE(1), THICK(1), LELEM(1)

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FIRE5-T3 2227
FIRE5-T3 2228
FIRE5-T3 2229
FIRE5-T3 2230
FIRE5-T3 2231
FIRE5-T3 2232
FIRE5-T3 2233
FIRE5-T3 2234
FIRE5-T3 2235
FIRE5-T3 2236
FIRE5-T3 2237
FIRE5-T3 2238
FIRE5-T3 2239
FIRE5-T3 2240
FIRE5-T3 2241
FIRE5-T3 2242
FIRE5-T3 2243
FIRE5-T3 2244
FIRE5-T3 2245
FIRE5-T3 2246
FIRE5-T3 2247
FIRE5-T3 2248
FIRE5-T3 2249
FIRE5-T3 2250
FIRE5-T3 2251
FIRE5-T3 2252
FIRE5-T3 2253
FIRE5-T3 2254
FIRE5-T3 2255
FIRE5-T3 2256
FIRE5-T3 2257
FIRE5-T3 2258
FIRE5-T3 2259
FIRE5-T3 2260
FIRE5-T3 2261
FIRE5-T3 2262
FIRE5-T3 2263
FIRE5-T3 2264
FIRE5-T3 2265
FIRE5-T3 2266
FIRE5-T3 2267
FIRE5-T3 2268
FIRE5-T3 2269
FIRE5-T3 2270
FIRE5-T3 2271
FIRE5-T3 2272
FIRE5-T3 2273
FIRE5-T3 2274
FIRE5-T3 2275
FIRE5-T3 2276
FIRE5-T3 2277
FIRE5-T3 2278
FIRE5-T3 2279
FIRE5-T3 2280
FIRE5-T3 2281
FIRE5-T3 2282
FIRE5-T3 2283
FIRE5-T3 2284
FIRE5-T3 2285
FIRE5-T3 2286
FIRE5-T3 2287
FIRE5-T3 2288

C          DO 70 I=11,12
C          IF (LMAT(I).GT.NBCMAT) GO TO 80
C          II=LJ(I)
C          JJ=LJ(I-NFBC1D)
C          IF (KODE(II).EQ.4HTEMP.OR.KODE(JJ).EQ.4HTEMP) GO TO 80
70 CONTINUE
GO TO 90

C          80 IO=I-NFBC1D
C          WRITE (NOUT,280) IO,LJ(I),LJ(IO),LMAT(I),LFIRE(I),LELEM(I)
C          STOP

C          CALCULATE AREA OF FIRE B.C. SURFACE ELEMENT
C
C          GO CONTINUE
C          DO 100 I=11,12
C          II=LJ(I)
C          JJ=LJ(I-NFBC1D)
C          DX=X(II)-X(JJ)
C          DY=Y(II)-Y(JJ)
C          DZ=Z(II)-Z(JJ)
C          D=DX*DX+DY*DY+DZ*DZ
C          IK=LELEM(I)
C          100 AIJKL(I)=SORT(D)*THICK(IK)

C          OUTPUT SURFACE ELEMENT DATA
C
C          WRITE (NOUT,300)
C          WRITE (NOUT,330) (I,LJ(I+NFBC1D),LJ(I),LMAT(I+NFBC1D),LFIRE(I+NFBC1D),
C          11D),AIJKL(I+NFBC1D),I=1,NS2

C          REDUCE AREA TO 1/2 FOR FUTURE CALCULATIONS
C
C          DO 110 I=11,12
C          110 AIJKL(I)=AIJKL(I)/2.

C          THREE - DIMENSIONAL ELEMENTS
C
C          120 IF (NS3.EQ.0) GO TO 190
C          WRITE (NOUT,220) NS3
C
C          INPUT PIPE B.C. DATA - TWO ELEMENTS PER CARD
C
C          II=NFBC1D+NFBC2D+1
C          I2=NFEC1D+NFBC2D+NS3
C          NI=NFEC1D+NFHC2D
C          READ (NIN,230) (LT(I),LJ(I-NFBC2D),LK(I-NI),LL(I-NI),LNAT(I),LFIRE
C          I(I),I=11,12)

C          CHECK VALIDITY OF MATERIAL REQUIREMENTS AND PREVIOUSLY DECLARED
C          BOUNDARY CONDITIONS. FOR A SURFACE TO BE CONSIDERED A FIRE B.C.
C          IT MUST BE BOUNDED BY NODES THAT HAVE A DECLARED FLOW B.C.
C
C          DO 130 I=1,NS3
C          IF (LMAT(I+NI).GT.NBCMAT) GO TO 140
C          II=LJ(I+NI)
C          JJ=LJ(I+NFBC2D)
C          KK=LK(I)
C          LLL=LL(I)
C          II=(KODE(II).EQ.4HTEMP.OR.KODE(JJ).EQ.4HTEMP) GO TO 140
C          II=(KODE(KK).EQ.4HTEMP.OR.KODE(LL).EQ.4HTEMP) GO TO 140
130 CONTINUE

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FIRES-T3 2289      130 CONTINUE
FIRES-T3 2290      GO TO 150
FIRES-T3 2291
C
FIRES-T3 2292      140 CONTINUE
FIRES-T3 2293      WRITE (INOUT,260) I,LJ(I+N1),LJ(I+NFBC2D),LK(I),LL(I),LMAT(I+N1),LF
FIRES-T3 2294      FIRE(I+N1)
FIRES-T3 2295      STOP
C
C      CALCULATE THE AREA OF THE FIRE HC SURFACE ELEMENTS
C
FIRES-T3 2296      150 CONTINUE
FIRES-T3 2300      DO 170 I=1,NS3
FIRES-T3 2301      II=LJ(I+N1)
FIRES-T3 2302      JJ=LJ(NFBC2D+I)
FIRES-T3 2303      KK=LK(I)
FIRES-T3 2304      DX=X(II)-X(JJ)
FIRES-T3 2305      DY=Y(II)-Y(JJ)
FIRES-T3 2306      DZ=Z(II)-Z(JJ)
FIRES-T3 2307      D=DX*DX+DY*DY+DZ*DZ
FIRES-T3 2308      XL1=SQRT(D)
FIRES-T3 2309      DX=X(JJ)-X(KK)
FIRES-T3 2310      DY=Y(JJ)-Y(KK)
FIRES-T3 2311      DZ=Z(JJ)-Z(KK)
FIRES-T3 2312      D=DX*DX+DY*DY+DZ*DZ
FIRES-T3 2313      XL2=SQRT(D)
FIRES-T3 2314      ATJKL(I+N1)=XL1*XL2
FIRES-T3 2315      IF (LK(I)-LL(I)) .LT. 170, 160, 170
FIRES-T3 2316      ATJKL(I+N1)=0.5*ATJKL(I+N1)
FIRES-T3 2317      170 CONTINUE
C
C      OUTPUT SURFACE ELEMENT DATA
C
FIRES-T3 2318      WRITE (INOUT,310)
FIRES-T3 2319      WRITE (INOUT,340) I,LJ(I+N1),LJ(I+NFBC2D),LK(I),LL(I),LMAT(I+N1),LF
FIRES-T3 2320      FIRE(I+N1),ATJKL(I+N1),I=1,NS3
C
C      REDUCE THE SURFACE AREA BY 1/4 TO AID IN FUTURE COMPUTATION
C
C
FIRES-T3 2321      DO 190 I=II,12
FIRES-T3 2322      180 ATJKL(I)=.25*ATJKL(I)
C
FIRES-T3 2323      190 RETURN
C
C
C
FIRES-T3 2324      200 FORMAT (//$/16H + . . . THERE ARE,14,40H 1-D SURFACE NODES EXPOSED TO
FIRES-T3 2325      1C FIRE . . .)
FIRES-T3 2326      210 FORMAT (//$/16H + . . . THERE ARE,14,41H 2-D SURFACE ELEMENTS EXPOSED
FIRES-T3 2327      TO 1C FIRE . . .)
FIRES-T3 2328      220 FORMAT (//$/16H + . . . THERE ARE,14,41H 3-D SURFACE ELEMENTS EXPOSED
FIRES-T3 2329      TO 1C FIRE . . .)
FIRES-T3 2330      230 FORMAT (1215)
FIRES-T3 2331      240 FORMAT (1615)
FIRES-T3 2332      250 FORMAT (1515)
FIRES-T3 2333      260 FORMAT (51/,54F) -- PROGRAM TERMINATED - FIRE HC INPUT ERROR -
FIRES-T3 2334      I = -1,X,715)
FIRES-T3 2335      270 FORMAT (51/,54F) -- PROGRAM TERMINATED - FIRE HC INPUT ERROR -
FIRES-T3 2336      I = -1,X,515)
FIRES-T3 2337      280 FORMAT (51/,54F) -- PROGRAM TERMINATED - FIRE PC INPUT ERROR -
FIRES-T3 2338      I = -1,X,515)
FIRES-T3 2339      290 FORMAT (//9X,47H DESCRIPTION OF SURFACE DIRECTLY EXPOSED TO FIRE, //,
FIRES-T3 2340      15BH FIREFC) NONE     MAT     FIRE     AREA/ZBH SURFACE    I   TYPE
FIRES-T3 2341
FIRES-T3 2342
FIRES-T3 2343
FIRES-T3 2344
FIRES-T3 2345
FIRES-T3 2346
FIRES-T3 2347
FIRES-T3 2348
FIRES-T3 2349
FIRES-T3 2350

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FIRE5-T3 2351      2   TYPE//)
FIRE5-T3 2352 300 FORMAT (//9X,47HDESCRIPTION OF SURFACE DIRECTLY EXPOSED TO FIRE,//
FIRE5-T3 2353     145H FIREEL  NODE  NODE  MAT  FIRE    AREA/35H SURFACE   1
FIRE5-T3 2354     2   J   TYPE  TYPE//)
FIRE5-T3 2355 310 FORMAT (//9X,47HDESCRIPTION OF SURFACE DIRECTLY EXPOSED TO FIRE,//
FIRE5-T3 2356     159H FIRERC  NODE  NODE  NODE  MAT  FIRE    AREA/49H
FIRE5-T3 2357     2 SURFACE  I   J   K   L   TYPE  TYPE//)
FIRE5-T3 2358 320 FORMAT (4I7,F10.3)
FIRE5-T3 2359 330 FORMAT (5I7,F10.3)
FIRE5-T3 2360 340 FORMAT (7I7,F10.3)
FIRE5-T3 2361 350 FORMAT (180RI)
FIRE5-T3 2362 360 FORMAT (6(/),45H - - - PROGRAM TERMINATED - INPUT ERROR - - -,//IX
FIRE5-T3 2363     1,80RI)
FIRE5-T3 2364     END

```

```

FIRES-T3 2408      C      ADD HEAT FLOW TO NODES BOUNDING SURFACE ELEMENT
FIRES-T3 2409      C
FIRES-T3 2410      C      B(I)=E(I)+0
FIRES-T3 2411      C      20 CONTINUE
FIRES-T3 2412      C
FIRES-T3 2413      C      TWO - D I M E N S I O N A L   E L E M E N T S
FIRES-T3 2414      C
FIRES-T3 2415      C      40 IF (NS2.EC.0) GO TO 60
FIRES-T3 2416      C      DO 50 N=1,NS2
FIRES-T3 2417      C
FIRES-T3 2418      C      ESTABLISH BASIC FIRE B.C. VARIABLES FOR THIS SURFACE ELEMENT
FIRES-T3 2419      C
FIRES-T3 2420      C      N3=N+NFBC1D
FIRES-T3 2421      C      I=L1(N3)
FIRES-T3 2422      C      J=LJ(N)
FIRES-T3 2423      C      M=LMAT(N3)
FIRES-T3 2424      C      LF=LFIRE(N3)
FIRES-T3 2425      C      TF=TFIRE(LF)
FIRES-T3 2426      C      TS=.5*(T(I)+T(J))
FIRES-T3 2427      C
FIRES-T3 2428      C      FIND HEAT FLOW DUE TO FIRE B.C.
FIRES-T3 2429      C
FIRES-T3 2430      C      Q=QFIRE(TF,TS,LF,NBCTYP,M,N3,TSHIFT,SB,TF4,MAT,FXYS,AIJKL)
FIRES-T3 2431      C
FIRES-T3 2432      C      ADD HEAT FLOW TO NODES BOUNDING SURFACE ELEMENT
FIRES-T3 2433      C
FIRES-T3 2434      C      B(I)=E(I)+Q
FIRES-T3 2435      C      B(J)=E(J)+Q
FIRES-T3 2436      C      50 CONTINUE
FIRES-T3 2437      C
FIRES-T3 2438      C      THREE - D I M E N S I O N A L   E L E M E N T S
FIRES-T3 2439      C
FIRES-T3 2440      C      60 IF (NS3.EC.0) GO TO 80
FIRES-T3 2441      C      DO 70 N=1,NS3
FIRES-T3 2442      C
FIRES-T3 2443      C      ESTABLISH BASIC FIRE B.C. VARIABLES FOR THIS SURFACE ELEMENT
FIRES-T3 2444      C
FIRES-T3 2445      C      N3=N+NFBC1D+NFBC2D
FIRES-T3 2446      C      I=L1(N3)
FIRES-T3 2447      C      J=LJ(N+NFBC2D)
FIRES-T3 2448      C      NK=LK(N)
FIRES-T3 2449      C      NL=LL(N)
FIRES-T3 2450      C      M=LMAT(N3)
FIRES-T3 2451      C      LF=LFIRE(N3)
FIRES-T3 2452      C      TF=TFIRE(LF)
FIRES-T3 2453      C      TS=.25*(T(I)+T(J)+T(NK)+T(NL))
FIRES-T3 2454      C
FIRES-T3 2455      C      FIND HEAT FLOW DUE TO FIRE B.C.
FIRES-T3 2456      C
FIRES-T3 2457      C      Q=QFIRE(TF,TS,LF,NBCTYP,M,N3,TSHIFT,SB,TF4,MAT,FXYS,AIJKL)
FIRES-T3 2458      C
FIRES-T3 2459      C      ADD HEAT FLOW TO NODES BOUNDING SURFACE ELEMENT
FIRES-T3 2460      C
FIRES-T3 2461      C      B(I)=E(I)+Q
FIRES-T3 2462      C      B(J)=E(J)+Q
FIRES-T3 2463      C      B(NK)=E(NK)+Q
FIRES-T3 2464      C      B(NL)=E(NL)+Q
FIRES-T3 2465      C      70 CONTINUE
FIRES-T3 2466      C
FIRES-T3 2467      C      80 RETURN
FIRES-T3 2468      C      END

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```

FIRES-T3 2469      FUNCTION OFIRE (TF,TS,LF,NRCTYP,N,N3,TSHIFT,SG,TF4,MAT,FXYS,ATJKL)
FIRES-T3 2470      C
FIRES-T3 2471      C
FIRES-T3 2472      C
FIRES-T3 2473      C
FIRES-T3 2474      C
FIRES-T3 2475      C
FIRES-T3 2476      C
FIRES-T3 2477      C
FIRES-T3 2478      C
FIRES-T3 2479      C
FIRES-T3 2480      C
FIRES-T3 2481      C
FIRES-T3 2482      C
FIRES-T3 2483      C
FIRES-T3 2484      C
FIRES-T3 2485      C
FIRES-T3 2486      C
FIRES-T3 2487      C
FIRES-T3 2488      C
FIRES-T3 2489      C
FIRES-T3 2490      C
FIRES-T3 2491      C
FIRES-T3 2492      C
FIRES-T3 2493      C
FIRES-T3 2494      C
FIRES-T3 2495      C
FIRES-T3 2496      C
FIRES-T3 2497      C
FIRES-T3 2498      C
FIRES-T3 2499      C
FIRES-T3 2500      C
FIRES-T3 2501      C
FIRES-T3 2502      C
FIRES-T3 2503      C
FIRES-T3 2504      C
FIRES-T3 2505      C
FIRES-T3 2506      C
FIRES-T3 2507      C
FIRES-T3 2508      C
FIRES-T3 2509      C
FIRES-T3 2510      C
FIRES-T3 2511      C
FIRES-T3 2512      C
FIRES-T3 2513      C
FIRES-T3 2514      C
FIRES-T3 2515      C
FIRES-T3 2516      C
FIRES-T3 2517      C
FIRES-T3 2518      C
FIRES-T3 2519      C
FIRES-T3 2520      C
FIRES-T3 2521      C
FIRES-T3 2522      C
FIRES-T3 2523      C
FIRES-T3 2524      C
FIRES-T3 2525      C
FIRES-T3 2526      C
FIRES-T3 2527      C

C      FUNCTION *OFIRE* FINDS HEAT FLOW DUE TO FIRE ON SURFACE ELEMENT

C      DIMENSION TF4(1), MAT(1), FXYS(1), ATJKL(1)

C      DT=TF-TS
C      IF (NRCTYP.EQ.1) GO TO 10
C
C      LINEAR FIRE BOUNDARY CONDITION
C
C      M=(N-1)*2
C      JJ=MAT(M+1)
C      K=MAT(M+2)
C      TA=0.5*(TF+TS)
C      H=VMAT(K,FXYS(JJ),FXYS(JJ+K),FXYS(JJ+K+K),TA,10H-H(T))
C      Q=AIJKL(N3)*H*DT
C      GO TO 50
C
C      10 CONTINUE
C
C      NON-LINEAR FIRE BOUNDARY CONDITION
C
C      K=MAT(N)
C      A=FXYS(K)
C      QC=0.0
C      IF (A.EQ.0) GO TO 30
C
C      CALCULATE CONVECTION TERM - QC
C
C      P=FXYS(K+1)
C      IF (P.EQ.1.0) GO TO 20
C      SIGN=1.0
C      IF (CT.LT.0) SIGN=-1.0
C      DT=SIGN*DT
C      QC=SIGN*A*DT*#P
C      GO TO 30
C      20 QC=A*DT
C      30 V=FXYS(K+2)
C      OR=0.0
C      IF (V.EQ.0) GO TO 40
C
C      CALCULATE RADIATION TERM - QR
C
C      TS=TS+TSHIFT
C      TS=T+TS
C      TS=T+TS
C      AB=FXYS(K+3)
C      EF=FXYS(K+4)
C      ES=FXYS(K+5)
C      QR=SER*V*(AB*EF*TF4(LF)-ES*TS)
C      40 QR=AIJKL(N3)*(CC+OR)
C      50 QR=0.0
C      RETURN
C
C      END

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FIRES-T3 2528
FIRES-T3 2529
FIRES-T3 2530
FIRES-T3 2531
FIRES-T3 2532
FIRES-T3 2533
FIRES-T3 2534
FIRES-T3 2535
FIRES-T3 2536
FIRES-T3 2537
FIRES-T3 2538
FIRES-T3 2539
FIRES-T3 2540
FIRES-T3 2541
FIRES-T3 2542
FIRES-T3 2543
FIRES-T3 2544
FIRES-T3 2545
FIRES-T3 2546
FIRES-T3 2547
FIRES-T3 2548
FIRES-T3 2549
FIRES-T3 2550
FIRES-T3 2551
FIRES-T3 2552
FIRES-T3 2553
FIRES-T3 2554
FIRES-T3 2555
FIRES-T3 2556
FIRES-T3 2557
FIRES-T3 2558
FIRES-T3 2559
FIRES-T3 2560
FIRES-T3 2561
FIRES-T3 2562
FIRES-T3 2563
FIRES-T3 2564
FIRES-T3 2565
FIRES-T3 2566
FIRES-T3 2567
FIRES-T3 2568
FIRES-T3 2569
FIRES-T3 2570
FIRES-T3 2571
FIRES-T3 2572
FIRES-T3 2573
FIRES-T3 2574
FIRES-T3 2575
FIRES-T3 2576
FIRES-T3 2577
FIRES-T3 2578
FIRES-T3 2579
FIRES-T3 2580
FIRES-T3 2581
FIRES-T3 2582
FIRES-T3 2583
FIRES-T3 2584
FIRES-T3 2585
FIRES-T3 2586
FIRES-T3 2587
FIRES-T3 2588
FIRES-T3 2589

C          SUBROUTINE EXEELS (X,Y,Z,LM,HAREA,THICK,VOLUME,IEL,IMAT,VEL)

C          SUBROUTINE *EXEELS* INPUTS DATA DESCRIBING ALL ELEMENTS WHICH HAVE
C          INTERNAL HEAT GENERATION

C          COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NOUT,NPLNCH,NUMNP,NEL1D,
C          IEL2D,NEL3D,NUMEL,MBAND,NMAT,NFBC1D,NFBC2D,NFHC3D,NRCMAT,NHCTYP
C          COMMON /EXOTH/ NINT1D,NINT2D,NINT3D,NINT,NCINT
C          DIMENSION X(1), Y(1), Z(1), LM(1), HAREA(1), THICK(1), VOLUME(1),
C          ITEL(1), IMAT(1), VEL(1)

C          ONE - D I M E N S I O N A L   E L E M E N T S

C          IF (NINT1D.EQ.0) GO TO 20

C          INPUT 1-D ELEMENTS WITH INTERNAL HEAT GENERATION - 8 ELEMENTS/CARD

C          READ (NIN,70) (IEL(I),IMAT(I),I=1,NINT1D)
C          FIND VOLUME OF ELEMENTS AND OUTPUT DATA

C          DO 10 N=1,NINT1D
C              IE=IEL(N)
C              II=LM(2*IE-1)
C              JJ=LM(2*IE)
C              DX=X(II)-X(JJ)
C              DY=Y(II)-Y(JJ)
C              DZ=Z(II)-Z(JJ)
C              DL=SCRT((DX*DX+DY*DY+DZ*DZ))
C 10      VEL(N)=DL*HAREA(IE)
C          WRITE (NOUT,80) NINT1D
C          WRITE (NOUT,90) (I,IEL(I),IMAT(I),VEL(I),I=1,NINT1D)

C          T W O - C I M E N S I O N A L   E L E M E N T S

C          20 IF (NINT2D.EQ.0) GO TO 40

C          INPUT 2-D ELEMENTS WITH INTERNAL HEAT GENERATION - 8 ELEMENTS/CARD

C          I1=NINT1D+1
C          I2=NINT1D+NINT2D
C          READ (NIN,70) (IEL(I),IMAT(I),I=I1,I2)
C          FIND VOLUME OF ELEMENTS AND OUTPUT DATA

C          DO 30 N=I1,I2
C              II=IEL(N)
C              IARG=2*NINT1D+4*IE-3
C              IEL=LM(IARG)
C              JJ=LM(IARG+1)
C              KK=LM(IARG+2)
C              LL=LM(IARG+3)
C              AJ=X(JJ)-X(II)
C              AK=X(KK)-X(II)
C              PJ=Y(JJ)-Y(II)
C              MK=Y(KK)-Y(II)
C              AREA=(AJ*PK-AK*PJ)/2.
C              AJ=X(LL)-X(II)
C              AK=X(LL)-X(II)
C              PJ=Y(LL)-Y(II)

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FIRE5-T3 2590
FIRE5-T3 2591
FIRE5-T3 2592
FIRE5-T3 2593
FIRE5-T3 2594
FIRE5-T3 2595
FIRE5-T3 2596
FIRE5-T3 2597
FIRE5-T3 2598
FIRE5-T3 2599
FIRE5-T3 2600
FIRE5-T3 2601
FIRE5-T3 2602
FIRE5-T3 2603
FIRE5-T3 2604
FIRE5-T3 2605
FIRE5-T3 2606
FIRE5-T3 2607
FIRE5-T3 2608
FIRE5-T3 2609
FIRE5-T3 2610
FIRE5-T3 2611
FIRE5-T3 2612
FIRE5-T3 2613
FIRE5-T3 2614
FIRE5-T3 2615
FIRE5-T3 2616
FIRE5-T3 2617
FIRE5-T3 2618
FIRE5-T3 2619
FIRE5-T3 2620
FIRE5-T3 2621
FIRE5-T3 2622
FIRE5-T3 2623
FIRE5-T3 2624
FIRE5-T3 2625
FIRE5-T3 2626
FIRE5-T3 2627
FIRE5-T3 2628
FIRE5-T3 2629
FIRE5-T3 2630

      BK=Y(LL)-Y(1)
      AREA=AREA+(AJ*BK-AK*BJ)/2.
      30 VEL(N)=AREA*THICK(IF)
      WRITE (NOUT,100) NINT2D
      WRITE (NOUT,90) (I,IEL(I+NINTID),IMAT(I+NINTID),VEL(I+NINTID)),I=1,NINT2D

C   THREE - D I M E N S I O N A L   E L E M E N T S

      40 IF (NINT3D.EQ.0) GO TO 60
C   INPUT 3-D ELEMENTS WITH INTERNAL HEAT GENERATION - 8 ELEMENTS/CARD
C
      I1=NINTID+NINT2D+1
      I2=INT
      READ (NIN,70) (IEL(I),IMAT(I),I=I1,I2)

C   FIND VOLUME OF ELEMENTS AND OUTPUT DATA

      DO 50 N=I1,I2
      IE=IEL(N)
      50 VEL(N)=VOLUME(IE)
      NI=NINTID+NINT2D
      WRITE (NOUT,110) NINT3D
      WRITE (NOUT,90) (I,IEL(I+NI),IMAT(I+NI),I=1,NINT2D)

C
      60 RETURN
C
C
C
      70 FORMAT (A(215))
      80 FORMAT (//9X,16H. . . THERE ARE ,14,2RH 1-D EXOTHERMIC ELEMENTS .
      1 .//2(30H      N    EL    FUN    VOLUME))
      90 FORMAT (2(I10,215,F10.4))
      100 FORMAT (//9X,16H. . . THERE ARE ,14,2RH 2-D EXOTHERMIC ELEMENTS .
      1 .//2(30H      N    EL    FUN    VOLUME))
      110 FORMAT (//9X,16H. . . THERE ARE ,14,2RH 3-D EXOTHERMIC ELEMENTS .
      1 .//2(30H      N    EL    FUN    VOLUME))
      END

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FIRE5-T3 2631
FIRE5-T3 2632
FIRE5-T3 2633
FIRE5-T3 2634
FIRE5-T3 2635
FIRE5-T3 2636
FIRE5-T3 2637
FIRE5-T3 2638
FIRE5-T3 2639
FIRE5-T3 2640
FIRE5-T3 2641
FIRE5-T3 2642
FIRE5-T3 2643
FIRE5-T3 2644
FIRE5-T3 2645
FIRE5-T3 2646

      SUBROUTINE EXOFUN (T,X0,EXYS,NSTCRE)
C
C
      SUBROUTINE *EXOFUN* INPUTS THE EXOTHERMIC HEAT GENERATION CURVE
      AS A LINEARIZED FUNCTION OF TIME
C
      COMMON /CONTROL/ ITITLE(6),IREAD(80),KIN,KOUT,KPUNCH,NUMNP,NELID,N
      IEL2D,NEL3D,NUMEL,MAND,MRAT,NEHC1D,NEHC2D,NEHC3D,NHCMAT,NHCTYP
      COMMON /XOUTH/ NINTID,NINT2D,NINT3D,NINT,NINT
      DIMENSION TXO(1), EXYS(1)
      NSTCRE=1
C
      OUTPUT PAGE HEADING
C
      WRITE (NOUT,40)

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FIRES-T3 2647      WRITE (INOUT,50)
FIRES-T3 2648      WRITE (INOUT,60) ITITLE
FIRES-T3 2649      WRITE (INOUT,70) NOINT
FIRES-T3 2650      WRITE (INOUT,50)
C
FIRES-T3 2651      DO 30 N=1,NOINT
FIRES-T3 2652      C
FIRES-T3 2653      READ CONTROL CARD
C
FIRES-T3 2654      C
FIRES-T3 2655      C
FIRES-T3 2656      C      NSTORE = IEXO(1) - STARTING POINT FOR FUNCTION N IN EXYS
FIRES-T3 2657      C      MK = IEXO(2) - NUMBER OF POINTS IN INPUT HEATING CURVE
FIRES-T3 2658      C      NT = IEXO(3) - TYPE OF INPUT HEATING DATA
FIRES-T3 2659      C          EQ. 0 - HEAT RATE PER UNIT VOLUME
FIRES-T3 2660      C          EQ. 1 - HEAT RATE PER UNIT MASS
FIRES-T3 2661      C
FIRES-T3 2662      C      WRITE (INOUT,110) N
FIRES-T3 2663      C      READ (INR,120) MK,NT
FIRES-T3 2664      C      MS=(N-1)*3
FIRES-T3 2665      C      IEXO(MS+1)=NSTORE
FIRES-T3 2666      C      IEXO(MS+2)=MK
FIRES-T3 2667      C      IEXO(MS+3)=NT
C
FIRES-T3 2668      C      INPUT HEATING FUNCTION
C
FIRES-T3 2669      C      N1=NSTORE-1
C
FIRES-T3 2670      C      READ (INR,130) (EXYS(N1+I),EXYS(N1+MK+I),I=1,MK)
C
FIRES-T3 2671      C      M=MK-1
C
FIRES-T3 2672      C      DETERMINE SLOPES AND PRINT HEATING FUNCTION
C
FIRES-T3 2673      C
C
FIRES-T3 2674      C      DO 10 I=1,M
C
FIRES-T3 2675      C      N1=NSTORE+I-1
C
FIRES-T3 2676      C      EXYS(N1+MK+MK)=(EXYS(N1+MK+1)-EXYS(N1+MK))/((EXYS(N1+1)-EXYS(N1))
C
FIRES-T3 2677      C      WRITE (INOUT,80)
C
FIRES-T3 2678      C      DC 20 I=1,M
C
FIRES-T3 2679      C      N1=NSTORE+I-1
C
FIRES-T3 2680      C      WRITE (INOUT,90) I,EXYS(N1),EXYS(N1+MK)
C
FIRES-T3 2681      C      20 WRITE (INOUT,100) EXYS(N1+MK+MK)
C
FIRES-T3 2682      C      WRITE (INOUT,90) MK,EXYS(NSTORE+MK-1),EXYS(NSTORE+MK+MK-1)
C
FIRES-T3 2683      C      NSTORE=NSTORE+3*MK
C
FIRES-T3 2684      C      30 CONTINUE
C
FIRES-T3 2685      C      RETURN
C
C
FIRES-T3 2686      C
C
FIRES-T3 2687      C
C
FIRES-T3 2688      C
C
FIRES-T3 2689      C
C
FIRES-T3 2690      C
C
FIRES-T3 2691      C
C
FIRES-T3 2692      C
C
FIRES-T3 2693      C
C
FIRES-T3 2694      C
C
FIRES-T3 2695      C
C
FIRES-T3 2696      C
C
FIRES-T3 2697      C
C
FIRES-T3 2698      C
C
FIRES-T3 2699      C
C
FIRES-T3 2700      C
C
FIRES-T3 2701      C
C
FIRES-T3 2702      C
C
FIRES-T3 2703      C
C
FIRES-T3 2704      C
C
FIRES-T3 2705      C
C
FIRES-T3 2706      C
C
        END

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```

FIRE5-T3 2707          SUBROUTINE QEXO (LM,TEL,IMAT,TEXC,EXYS,AT,MATL,VEL,MNTYPE,B,XYS,T)
FIRE5-T3 2708          IME)
FIRE5-T3 2709          C
FIRE5-T3 2710          C
FIRE5-T3 2711          C
FIRE5-T3 2712          C
FIRE5-T3 2713          C
FIRE5-T3 2714          C
FIRE5-T3 2715          C
FIRE5-T3 2716          C
FIRE5-T3 2717          C
FIRE5-T3 2718          C
FIRE5-T3 2719          C
FIRE5-T3 2720          C
FIRE5-T3 2721          C
FIRE5-T3 2722          C
FIRE5-T3 2723          C
FIRE5-T3 2724          C
FIRE5-T3 2725          C
FIRE5-T3 2726          C
FIRE5-T3 2727          C
FIRE5-T3 2728          C
FIRE5-T3 2729          C
FIRE5-T3 2730          C
FIRE5-T3 2731          C
FIRE5-T3 2732          C
FIRE5-T3 2733          C
FIRE5-T3 2734          C
FIRE5-T3 2735          C
FIRE5-T3 2736          C
FIRE5-T3 2737          C
FIRE5-T3 2738          C
FIRE5-T3 2739          C
FIRE5-T3 2740          C
FIRE5-T3 2741          C
FIRE5-T3 2742          C
FIRE5-T3 2743          C
FIRE5-T3 2744          C
FIRE5-T3 2745          C
FIRE5-T3 2746          C
FIRE5-T3 2747          C
FIRE5-T3 2748          C
FIRE5-T3 2749          C
FIRE5-T3 2750          C
FIRE5-T3 2751          C
FIRE5-T3 2752          C
FIRE5-T3 2753          C
FIRE5-T3 2754          C
FIRE5-T3 2755          C
FIRE5-T3 2756          C
FIRE5-T3 2757          C
FIRE5-T3 2758          C
FIRE5-T3 2759          C
FIRE5-T3 2760          C
FIRE5-T3 2761          C
FIRE5-T3 2762          C
FIRE5-T3 2763          C
FIRE5-T3 2764          C
FIRE5-T3 2765          C
FIRE5-T3 2766          C
FIRE5-T3 2767          C
FIRE5-T3 2768          C

          SUBROUTINE *CEXC* COMPUTES NOCAL HEAT FLOW DUE TO EXOTHERMIC
          REACTION WITHIN ELEMENTS

          COMMON /CONTROL/ ITITLE(6),IREAD(80),NIN,NMUT,NPUNCH,NUMNF,NEL1D,N
          IEL2D,NEL3D,NUREL,NBAND,NMAT,NFOC1D,NFOC2D,NFHG3D,NBCMAT,NHCTYP
          COMMON /EXOTH/ NINT1D,NINT2D,NINT3D,NINT,NINT
          DIMENSION LM(), TEL(), IMAT(), IEXO(), EXYS(), AT(), MATL()
          1, VEL(), MNTYPE(), B(), XYS()

          C   ONE - D I M E N S I O N A L   E L E M E N T S

          IF (NINT1D.EQ.0) GO TO 30
          DC 20 N=1,NINT1D
          IE=IEL(N)
          II=LM(2*IE-1)
          JJ=LM(2*IE)
          MS=IMAT(N)
          MS=3*(MS-1)
          J=IEXO(MS+1)
          K=IEXO(MS+2)
          LTYPE=IEXO(MS+3)
          D=VMAT(K,EXYS(J),EXYS(J+K),EXYS(J+K+K),TIME,1OHHEAT FUNCT)
          IF (LTYPE.EQ.0) GO TO 10
          TEMP=AT(IE)
          MS=MNTYPE(IE)
          MS=(MS-1)*6
          J=MATL(MS+5)
          K=MATL(MS+6)
          DENS=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TIME,1OH          DENS)
          D=0*DENS
          10 DADD=0*VEL(N)/2*
          B(II)=E(II)+DADD
          20 B(JJ)=B(JJ)+DADD
          C   T W E - D I M E N S I O N A L   E L E M E N T S

          30 IF (NINT2D.EQ.0) GO TO 60
          II=NINT1D+1
          IZ=NINT1D*NINT2D
          DC 50 N=II,I2
          IE=TEL(N)
          NI=2*NEL1D+4*IE
          II=LM(NI-3)
          JJ=LM(NI-2)
          KK=LM(NI-1)
          LL=LM(NI)
          MS=IMAT(N)
          MS=3*(MS-1)
          J=IEXO(MS+1)
          K=IEXO(MS+2)
          LTYPE=IEXO(MS+3)
          D=VMAT(K,EXYS(J),EXYS(J+K),EXYS(J+K+K),TIME,1OHHEAT FUNCT)
          IF (LTYPE.EQ.0) GO TO 40
          TEMP=AT(NEL1D+IE)
          MS=MNTYPE(NEL1D+IE)
          MS=(MS-1)*6
          J=MATL(MS+5)

```

```

FIRES-T3 2769
FIRES-T3 2770
FIRES-T3 2771
FIRES-T3 2772
FIRES-T3 2773
FIRES-T3 2774
FIRES-T3 2775
FIRES-T3 2776
FIRES-T3 2777
FIRES-T3 2778
FIRES-T3 2779
FIRES-T3 2780
FIRES-T3 2781
FIRES-T3 2782
FIRES-T3 2783
FIRES-T3 2784
FIRES-T3 2785
FIRES-T3 2786
FIRES-T3 2787
FIRES-T3 2788
FIRES-T3 2789
FIRES-T3 2790
FIRES-T3 2791
FIRES-T3 2792
FIRES-T3 2793
FIRES-T3 2794
FIRES-T3 2795
FIRES-T3 2796
FIRES-T3 2797
FIRES-T3 2798
FIRES-T3 2799
FIRES-T3 2800
FIRES-T3 2801
FIRES-T3 2802
FIRES-T3 2803
FIRES-T3 2804
FIRES-T3 2805
FIRES-T3 2806
FIRES-T3 2807
FIRES-T3 2808
FIRES-T3 2809
FIRES-T3 2810
FIRES-T3 2811
FIRES-T3 2812
FIRES-T3 2813
FIRES-T3 2814
FIRES-T3 2815
FIRES-T3 2816
FIRES-T3 2817
FIRES-T3 2818
FIRES-T3 2819

      K=MATL(MS+6)
      DFNS=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H      DENS)
      O=G*DENS
      40 QADD=C*VEL(N)/8.
      B(I1)=B(I1)+QADD
      B(JJ)=B(JJ)+QADD
      B(KK)=B(KK)+QADD
      50 B(LL)=B(LL)+QADD

C      THREE-DIMENSIONAL ELEMENTS
C
      60 IF (INTINTD,E0,0) GO TO 90
      I1=NINT1D+NINT2D+1
      I2=NINT
      DO 80 N=I1,I2
      IE=IEL(N)
      NI=2*NEL1D+4*NEL2D+8*IE
      IJ=LM(NI-7)
      JJ=LM(NI-6)
      KK=LM(NI-5)
      LL=LM(NI-4)
      ITT=LM(NI-3)
      JJJ=LM(NI-2)
      KKR=LM(NI-1)
      LLL=LM(NI)
      MS=IEL(N)
      MS=(MS-1)*3
      J=IEXD(MS+1)
      K=IEXC(MS+2)
      LTYPE=IE XD(MS+3)
      G=VMAT(K,EXYS(J),EXYS(J+K),EXYS(J+K+K),TIME,10HHEAT FUNCT)
      IF (LTYPE,E0,0) GO TO 70
      TEMP=AT(NEL1D+NEL2D+IE)
      MS=NN1TYPE(NEL1D+NEL2D+IE)
      MS=(MS-1)*6
      K=MATL(MS+6)
      J=MATL(MS+5)
      DENS=VMAT(K,XYS(J),XYS(J+K),XYS(J+K+K),TEMP,10H      DENS)
      O=G*DENS
      70 QADD=O*VEL(N)/8.
      B(I1)=B(I1)+QADD
      B(JJ)=B(JJ)+QADD
      B(KK)=B(KK)+QADD
      B(LL)=B(LL)+QADD
      B(I11)=B(I11)+QADD
      B(JJJ)=B(JJJ)+QADD
      B(KKK)=B(KKK)+QADD
      80 B(LLL)=B(LLL)+QADD

C      GO RETURN
      END

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<b>TITLE AND SUBTITLE (CITE IN FULL)</b> FIRES - T3 a Computer Program for the Fire Response of Structures -- Thermal (Three-Dimensional Version)					
<b>CONTRACT OR GRANT NUMBER</b> NBS-G7-9006-10/77		<b>TYPE OF REPORT AND/OR PERIOD COVERED</b> October 1977			
<b>AUTHOR(S) (LAST NAME, FIRST INITIAL, SECOND INITIAL)</b> Iding, R., Bresler, B., and Nizamuddin Department of Civil Engineering University of California, Berkeley, CA		<b>PERFORMING ORGANIZATION (CHECK (X) ONE BOX)</b> <input type="checkbox"/> NIST/GAITHERSBURG <input type="checkbox"/> NIST/BOULDER <input type="checkbox"/> JILA/BOULDER			
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<p>This report documents the computer program FIRES-T3 (Fire Response of Structures - Thermal - Three-Dimensional Version). The program evaluates the temperature distribution history of general three-dimensional solids or composites such as reinforced concrete subjected to fire environments. There are also options of two-dimensional and one-dimensional heat flow analyses. FIRES-T3 is based on a finite element formulation which considered the temperature dependence of thermal properties and the nonlinearities inherent in modeling the fire boundary condition. The finite element formulation, the idealization of the fire boundary condition, and the general numerical approach used in the program are discussed. Included in the report are appendices that provide input instructions for FIRES-T3, sample problems with listings of their input and output, and a complete Fortran listing of the computer program.</p>					
<b>KEY WORDS (MAXIMUM OF 9; 28 CHARACTERS AND SPACES EACH; SEPARATE WITH SEMICOLONS; ALPHABETIC ORDER; CAPITALIZE ONLY PROPER NAMES)</b> computer modes; concrete slabs; fire models; heat transfer; steel structures; structural design					
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